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# HYDREL flat cage guidance systems Linear recirculating roller guidance systems

Nowadays the performance of a linear guidance system is no longer determined simply by its load carrying capacity, rigidity and accuracy. In addition to these "basic virtues", there is a demand for customised guidance system solutions. These are solutions that are precisely matched to the requirements of the application and thus have a positive influence on the overall design concept of the machine or installation. As a result, technically accomplished and particularly costeffective complete designs can be achieved that fulfil the requirements of the market. In practice, therefore, various classes of guidance systems have gained acceptance whose suitability is often confined to particular applications.

Where linear locating or non-locating bearings with extremely high load carrying capacity and very smooth running are required for *limited* stroke lengths, *flat cage guidance systems* can be used. These guidance systems comprise a guideway arrangement separated by needle roller, cylindrical roller or ball flat cages. Flat cage guidance systems have extremely high rigidity, high accuracy and low friction and require significantly less space than other linear guidance systems.

Where locating/locating or locating/non-locating bearing arrangements with very high load carrying capacity are required for *unlimited* displacement distances, *linear recirculating roller guidance systems* can be used. These guidance systems comprise INA linear recirculating roller bearings with cylindrical rollers as rolling elements and HYDREL guideways with up to four raceways for the recirculating bearings. Linear recirculating roller guidance systems are preloaded and have extremely high rigidity, high accuracy and low friction; they allow compact designs that fulfil the requirements of the specific application.

Publication FRF gives information on the standard range of these HYDREL flat cage guidance systems and INA/HYDREL linear recirculating roller guidance systems. It supersedes the publications FKF and RUF. Any information in previous editions that does not concur with the data in this edition is therefore invalid.

INA-Schaeffler KG Linear Technology Division Homburg (Saar)

HYDREL AG Romanshorn (Switzerland)

# Linear guidance systems

Overview

Linear guidance systems		Applications	Load directions
Linear ball bearings KH	600	Machine building     Machine housings     Packaging machines     Handling equipment	
Linear ball bearings KS, KB		<ul> <li>Jigs and fixtures</li> <li>For compensation of angular misalignment.</li> </ul>	
Linear plain bearings PAB	00	Hydrodynamically lubricated linear guidance systems with low noise levels.	
Track roller guidance systems LF		<ul> <li>Machine building, packaging machines</li> <li>Handling equipment.</li> </ul>	
Two-row linear ball bearing and guideway assemblies KUE		<ul> <li>Machine building</li> <li>Sheet metalworking machines</li> <li>Plastics injection moulding machines</li> <li>Declaring machines</li> </ul>	
Four-row linear ball bearing and guideway assemblies KUVE		<ul> <li>Fackaging machines</li> <li>Handling equipment</li> <li>Machine tools</li> <li>For high load carrying capacity, rigidity and accuracy.</li> </ul>	
Six-row linear ball bearing and guideway assemblies KUSE			
Linear roller bearing and guideway assemblies RUE		<ul> <li>Machine tools</li> <li>For very high load carrying capacity, rigidity and accuracy.</li> </ul>	
Linear recirculating ball bearing units KUVS		<ul> <li>Machine building, handling equipment</li> <li>Linear guidance systems adaptable to individual requirements.</li> </ul>	
Linear recirculating roller bearings RUS, PR	J.	<ul> <li>Machine tools</li> <li>Adaptable locating/non-locating systems with very high load carrying capacity, rigidity and accuracy.</li> </ul>	
Flat cage guidance systems M, V, ML, J, S		<ul> <li>Machine tools</li> <li>For very high load carrying capacity, rigidity and accuracy</li> <li>Very low friction with limited stroke length.</li> </ul>	
Miniature linear recirculating ball bearing and guideway assemblies		<ul> <li>Precision engineering, manufacture of electronic components</li> <li>For low-friction applications.</li> </ul>	
Miniature carriage units with flat cages RMWE		<ul> <li>Precision engineering, manufacture of electronic components, microscope focussing systems</li> <li>For high demands for ease of movement and low wear.</li> </ul>	
Miniature linear guidance sets with flat cages RWS			
Plain guidance systems GFS		<ul> <li>Operating and handling equipment</li> <li>For maintenance-free applications.</li> </ul>	
Actuators	500	<ul> <li>Operating and handling equipment</li> <li>Complete driven linear systems comprising mechanical components, electric motor and controller.</li> </ul>	
Tables			

Load carrying capacity	Bearing type/ Friction	Acceleration	Travel speed	Operating temperature	Information
C/C <sub>0</sub> N		a <sub>max</sub> m/s²	v <sub>max</sub> m/min	°C	
up to 6 800/ up to 7 000	Ball recirculation	50	120	to +120	Catalogue 801
up to 8000/ up to 10600	Ball recirculation	100	120 to 300	to +120	Catalogue 801
up to 1000000	Plain bearing	50	180	to +80	Catalogue 801
up to 4 800/ up to 8 000	Track roller	50	600	-20 to +120	Catalogue 801
up to 28 000/ up to 37 000	Ball recirculation	150	180	-10 to +100	Catalogue 605
up to 82 000/ up to 181 000	Ball recirculation	150	300	-10 to +100	Catalogue 605
up to 125 400/ up to 312 000	Ball recirculation	150	300	-10 to +100	Catalogue 605
up to 270 000/ up to 640 000	Roller recirculation	100	120 to 180	-10 to +100	Catalogue 605
up to 26 000/ up to 46 500	Ball recirculation	150	180	-10 to +100	Available by agreement
up to 620 000/ up to 790 000	Needle cage	100	50 to 120	-10 to +100	Publication FRF
dependent on cage length	Needle cage	250	50 to 100	to +150	Publication FRF
up to 4 800/ up to 9 600	Ball recirculation	150	-	-10 to +100	MAI 81
up to 17 500/ up to 45 700	Roller cage	250	50 to 100	to +120	MAI 77
dependent on cage length	Roller cage	250	50 to 100	to +120	MAI 79
to 480	Plain bearing	-	60	-40 to +80	MAI 78
_	-	-	-	-	Publication ALE
-	-	-	-	-	Publication ALE

# Zwischentitel



HYDREL flat cage guidance systems

- comprise HYDREL/EGIS guideways of various types with appropriate flat or angled flat cages. The cage type depends on the guideway design
- have high precision raceways and cages with a large number of precision rolling elements – needle rollers, cylindrical rollers or balls
- are particularly suitable for limited stroke applications due to the relative movement between the cage and the two raceways
- are used as locating or non-locating bearings depending on their design
- give uniformly high accuracy over a wide range of loads
- have high load carrying capacity and rigidity with low friction coefficients
- are suitable for high accelerations
- do not involve recirculation of rolling elements and are characterised by smooth, quiet running
- differ from plain guidance systems in that accuracy is practically unaffected by the lubricant film

require very little space compared to other linear guidance systems. In addition to the individual components described, HYDREL can also supply complete, ready-to-fit flat cage guidance systems. Due to its engineering expertise, HYDREL is also an ideal choice as a manufacturer of special parts to meet specific customer requirements with integral functional elements of a flat cage guidance system.

INA-Schaeffler KG Linear Technology Division Homburg (Saar)

HYDREL AG Romanshorn (Switzerland)

# Product range

Overview/selection scheme



Positive cage	Preload by adjusting	Accessories	<sup>2</sup> )		Flat cage <sup>2</sup> )			Anti- corrosion	max. Acceleration	Features
control	gib	End pieces	Wipers	sealing strips	Metal cage	Plastic cage	With friction damping	protection <sup>2</sup> )		see Page
									to 250 m/s <sup>2</sup>	50
						•	•		to 250 m/s <sup>2</sup>	56
									to 100 m/s <sup>2</sup>	62
									to 250 m/s <sup>2</sup>	68
									to 250 m/s <sup>2</sup>	74
									to 250 m/s <sup>2</sup>	88
•						•	•		to 250 m/s <sup>2</sup>	100

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- 48 Open arrangements

# Product range

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- 50 HYDREL/EGIS M and V guideways
- with angled needle roller flat cage and angled cylindrical roller flat cage
- 56 HYDREL/EGIS ML and V guideways with adjusting gib and angled needle roller flat cage
- 62 HYDREL/EGIS M and V guideways with integral toothed rack for positive control of angled needle roller flat cage
- 68 HYDREL/EGIS M guideways with sliding layer and V guideways
- 74 HYDREL/EGIS J and S guideways with needle roller flat cage
- 82 HYDREL/EGIS L counterstay system with needle roller and cylindrical roller flat cages, locating/non-locating bearing units

# 88 Accessories

- 88 End pieces for HYDREL/EGIS guideways
- 90 Wipers for HYDREL/EGIS guideways
- 92 End pieces for HYDREL/EGIS L counterstay system
- 94 Flat cages single row
  - double row
- 104 Angled flat cages
- 108 Needle roller flat cages with friction damping
- 113 Order forms

# **Product index**

sorted alphanumerically

Page	Туре	Description
95	BF	Needle roller flat cage made from profiled steel strip, single row
90	EAJ	Polyurethane wiper for J guideways, with end piece, for protecting raceway against contamination
90	EAM	Polyurethane wiper for M guideways, with end piece, for protecting raceway against contamination
90	EAML	Polyurethane wiper for ML guideways, with end piece, for protecting raceway against contamination
90	EAV	Polyurethane wiper for V guideways, with end piece, for protecting raceway against contamination
88	EJ	Steel end piece for J guideways, for preventing cage from creeping out of load zone
88	EM	Steel end piece for M guideways, for preventing cage from creeping out of load zone
88	EML	Steel end piece for ML guideways, for preventing cage from creeping out of load zone
93	ESM	Insert nuts for guideways, allowing conversion of type 15 holes to threaded holes
88	EV	Steel end piece for V guideways, for preventing cage from creeping out of load zone
94	FF	Needle roller flat cage made from plastic, single row, dovetail grooves on ends
95	FFW	Needle roller angled flat cage of series FFZW, shanks at right angles to each other
95	FFZW	Needle roller flat cage made from plastic, double row, rows connected by elastic crosspiece, rows can be set at any angle to each other
94	Н	Needle roller flat cage made from light metal, single row
97	HB	Ball flat cage, multi-row, made from light metal, with high precision balls
108	HG	Needle roller flat cage with friction damping
108	HGW	Needle roller angled flat cage with friction damping
96	HR	Cylindrical roller flat cage, single row, light metal carrier with plastic retainer inserts
97	HRW	Cylindrical roller flat cage, double row, light metal carrier with plastic retainer inserts, shanks at right angles to each other

Page	Туре	Description
95	HW	Angled needle roller flat cage made from light metal, double row, shanks at right angles to each other
95	HZW	Needle roller flat cage made from light metal, double row
97	HRZW	Cylindrical roller flat cage, double row, light metal carrier with plastic retainer inserts
74	J	Guideway for needle roller flat cages, matched to S guideway
82	LU	U counterstay for L counterstay system LUE
82	LUE	L counterstay system, comprising M, V, J and S guideways, one L counterstay, angled needle roller flat cages and needle roller flat cages
50	Μ	Guideway for angled needle roller and angled cylindrical roller flat cages, optionally with sliding layer, matched to V guideway
56	ML	M guideway with adjusting gib for preloading system, for angled needle roller flat cages, matched to V guideway
62	MVZ	Set of M and V guideways with integral toothed rack and pinion in cage for positive cage control, dimensionally identical to M and V guideways, with angled needle roller flat cage
74	S	Guideway for needle roller flat cages, matched to J guideway
50	V	Guideway for angled needle roller and angled cylindrical roller flat cages, matched to M or ML guideway

# Index of suffixes

Sorting criteria: A-Z

Suffix	Definition
BK	Metal flat cage made from aluminium or steel, protected against corrosion by plating on all surfaces
BR	Metal flat cage made from aluminium or steel, sliding surfaces with friction-reducing coating
DSV	Anti-corrosion protection by thin layer chromium plating
E1	No threaded holes in end faces of guideways
E2	Guideways with raceway lead zones on end faces
F	Cage made from unhardened steel
G	Flat cage: other grades of rolling elements
L	Hole pattern on left, mirror image of R
LA/	Indication of end distances if these are outside the limit values in the publication and/or are asymmetrical. Indication of $C_5/C_6$ by: $C_5$ distance between start of guideway and first hole $C_6$ distance between last hole and end of guideway
LB1	Raceway lined with TURCITE® B (standard)
LP21	Raceway lined with Permaglide <sup>®</sup> P21
MS	Cage made from brass
PP	M guideways sealed on all sides. Longitudinal surfaces with contact lip seals. End faces fitted with wipers EAM
P	Restricted positional tolerance for hole pattern in guideways. Allows fitting to predrilled hole pattern. The suffix P is immediately after the grade
Q2	Accuracy class, high precision
Q6	Accuracy class, precision
Q10	Accuracy class, normal
R	Hole pattern on right, mirror image of L
RS	Damping force differs from standard in needle roller flat cages with friction damping
US	M/V guideways with restricted tolerance for dimensions $A_1$ and $A_2$ . US parts can be used in any combination
ZZ	M guideways sealed on all sides. Longitudinal sealing by gap seals, end faces fitted with wipers EAM
2SX	X in the sort suffix indicates that linear guidance systems of non-identical design are sorted together. Example: M2SX with ML2SX or V/152SX with V/032SX
4SX	Matching in sets for combinations of M/V, J/S guideways in open designs

# Symbols and units

Unless stated otherwise in the text, the values used in this catalogue have the following symbols, units and definitions.

b	mm	Distance between guidance systems
С	Ν	Basic dynamic load rating for a cage length of 100 mm
C <sub>w</sub>	Ν	Effective dynamic load rating
CL	N/mm	Rigidity of flat cage guidance system
C <sub>DIN</sub>	Ν	Basic dynamic load rating C for a rating life of 100000 m
C <sub>0</sub>	Ν	Basic static load rating for a cage length of 100 mm
C <sub>Ow</sub>	Ν	Effective static load rating
C <sub>4</sub>	mm	Hole pitch
C <sub>5,</sub> C <sub>6</sub>	mm	Distance between start or end of guideway and nearest hole
C <sub>5min</sub> , C <sub>6min</sub>	mm	Minimum value for $C_5$ and $C_6$
C <sub>50000</sub>	Ν	Basic dynamic load rating C for a rating life of 50000 m
D <sub>w</sub>	mm	Ball diameter
е	mm	Distance between centre of first or last cage pocket and end of cage
f <sub>H</sub>	-	Dynamic hardness factor
f <sub>HO</sub>	-	Static hardness factor
f <sub>α</sub>	-	Dynamic load direction factor
$f_{\alpha 0}$	-	Static load direction factor
F	Ν	Operating load, load on guidance system
F <sub>i</sub>	Ν	Variable load
F <sub>R</sub>	Ν	Displacement resistance
F <sub>RV</sub>	Ν	Table displacement friction
F <sub>w</sub>	-	Limiting load for effective cage length
F <sub>perm</sub>	max.	Permissible limiting load for reference length of 100 mm
Н	mm	Distance between end points of stroke
k <sub>F</sub>	-	Dynamic load factor
k <sub>OF</sub>	-	Static load factor
К	-	Factor for calculating elastic deformation, dependent on the product type
К <sub>Р</sub>	-	Load correction factor
Κ <sub>U</sub>	-	Environmental correction factor
K <sub>W</sub>	-	Stroke correction factor
K <sub>LF</sub>	-	Bearing factor

L	m	Basic rating life in 100000 m
L	mm	Length of guideway
L <sub>A</sub>	mm	Length of guideway with allowance for wiper
L <sub>h</sub>	h	Basic rating life in hours
L <sub>w</sub>	mm	Rolling element length
L <sub>K</sub>	mm	Length of cage
M <sub>A</sub>	-	Tightening torque for pressure screws
n	-	Maximum possible number of hole pitches
n <sub>osc</sub>	min <sup>-1</sup>	Number of return strokes per minute
р	-	Life exponent
р	N/mm <sup>2</sup>	Contact pressure
Р	Ν	Equivalent dynamic load
P <sub>0</sub>	Ν	Maximum equivalent static load
q <sub>i</sub>	%	Proportion of total duration
RS	Ν	Damping force of flat cages with friction damping in direction of movement
S <sub>0</sub>	-	Static load safety factor
t	mm	Pocket pitch in basic cage element
L		1 5
t t <sub>f</sub>	h	Basic lubrication interval in hours
t t <sub>f</sub> t <sub>fG</sub>	h h	Basic lubrication interval in hours Guide value for lubricant operating life in hours
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub>	h h h	Basic lubrication interval in hours Guide value for lubricant operating life in hours Guide value for relubrication interval in hours
t t <sub>f</sub> t <sub>fR</sub> Vi	h h h m/min	Basic lubrication interval in hours Guide value for lubricant operating life in hours Guide value for relubrication interval in hours Variable speed
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> V	h h h m/min m/min	Basic lubrication interval in hours Guide value for lubricant operating life in hours Guide value for relubrication interval in hours Variable speed Equivalent dynamic speed
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> V V V	h h h m/min m/min	Basic lubrication interval in hours Guide value for lubricant operating life in hours Guide value for relubrication interval in hours Variable speed Equivalent dynamic speed Number of holes
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> <del>V</del> x Z	h h h m/min m/min -	Basic lubrication interval in hours Guide value for lubricant operating life in hours Guide value for relubrication interval in hours Variable speed Equivalent dynamic speed Number of holes Number of rolling elements per row
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> V V V X Ζ	h h m/min m/min - -	<ul> <li>Basic lubrication interval in hours</li> <li>Guide value for lubricant operating life in hours</li> <li>Guide value for relubrication interval in hours</li> <li>Variable speed</li> <li>Equivalent dynamic speed</li> <li>Number of holes</li> <li>Number of rolling elements per row</li> <li>Load direction angle deviating from main load direction</li> </ul>
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> V V V X Z α δ	h h m/min m/min - -	<ul> <li>Basic lubrication interval in hours</li> <li>Guide value for lubricant operating life in hours</li> <li>Guide value for relubrication interval in hours</li> <li>Variable speed</li> <li>Equivalent dynamic speed</li> <li>Number of holes</li> <li>Number of rolling elements per row</li> <li>Load direction angle deviating from main load direction</li> <li>Elastic deformation at contact points</li> </ul>
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> V V V X Z α δ	h h m/min m/min - - v µm	<ul> <li>Basic lubrication interval in hours</li> <li>Guide value for lubricant operating life in hours</li> <li>Guide value for relubrication interval in hours</li> <li>Variable speed</li> <li>Equivalent dynamic speed</li> <li>Number of holes</li> <li>Number of rolling elements per row</li> <li>Load direction angle deviating from main load direction</li> <li>Elastic deformation at contact points</li> <li>Coefficient of friction</li> </ul>
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> V V X Z α δ μ μ	h h m/min m/min - - » µm -	<ul> <li>Basic lubrication interval in hours</li> <li>Guide value for lubricant operating life in hours</li> <li>Guide value for relubrication interval in hours</li> <li>Variable speed</li> <li>Equivalent dynamic speed</li> <li>Number of holes</li> <li>Number of rolling elements per row</li> <li>Load direction angle deviating from main load direction</li> <li>Elastic deformation at contact points</li> <li>Coefficient of friction</li> <li>Coefficient of rolling friction</li> </ul>
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> <del>V</del> x Z α δ μ μ μ <sub>R</sub> Δh	h h m/min m/min - - v µm - -	<ul> <li>Basic lubrication interval in hours</li> <li>Guide value for lubricant operating life in hours</li> <li>Guide value for relubrication interval in hours</li> <li>Variable speed</li> <li>Equivalent dynamic speed</li> <li>Number of holes</li> <li>Number of rolling elements per row</li> <li>Load direction angle deviating from main load direction</li> <li>Elastic deformation at contact points</li> <li>Coefficient of rolling friction</li> <li>Permissible height offset</li> </ul>
t <sub>f</sub> t <sub>fG</sub> t <sub>fR</sub> V <sub>i</sub> V V X Z α δ μ μ μ R Δh Σ <sub>R</sub>	h h m/min m/min - - v µm - um N	<ul> <li>Basic lubrication interval in hours</li> <li>Guide value for lubricant operating life in hours</li> <li>Guide value for relubrication interval in hours</li> <li>Variable speed</li> <li>Equivalent dynamic speed</li> <li>Number of holes</li> <li>Number of rolling elements per row</li> <li>Load direction angle deviating from main load direction</li> <li>Elastic deformation at contact points</li> <li>Coefficient of friction</li> <li>Coefficient of rolling friction</li> <li>Permissible height offset</li> <li>Total frictional force</li> </ul>



# Load carrying capacity and life

The dimensioning of a flat cage guidance system is based on the basic dynamic and static load ratings of the flat cage fitted.

The basic load ratings of linear guidance systems without rolling element recirculation are defined in accordance with DIN 636-3. Basic dynamic load ratings in accordance with DIN 636 are calculated on a basic rating life of 100000 m.

# Basic load ratings to DIN, basic load ratings as used in Far East

Suppliers from the Far East often give basic load ratings based on a rating life of 50000 m. The resulting basic load ratings are up to 20% higher than those calculated to DIN 636. The basic load rating values are converted in accordance with Tables 1 and 2.

# Table 1 · Converting basic load ratings to DIN to basic load ratings as used in the Far East

Flat cage guidance system	Conversion factor
Ball cage	$C_{50000} = 1,26 \cdot C_{DIN}$
Cylindrical roller cage	$C_{50000} = 1,23 \cdot C_{DIN}$

### Table 2 · Converting basic load ratings as used in the Far East into basic load ratings to DIN

Flat cage guidance system	Conversion factor
Ball cage	$C_{DIN} = 0,79 \cdot C_{50000}$
Cylindrical roller cage	C <sub>DIN</sub> = <b>0,81</b> · C <sub>50000</sub>

C<sub>50000</sub>

Ν

Basic dynamic load rating C for basic rating life of 50000 m  $C_{\text{DIN}}$  N

Basic dynamic load rating C for basic rating life of 100000 m – definition in accordance with DIN 636.

# Dynamic load carrying capacity and life

The dynamic load carrying capacity of the flat cage guidance system is determined by the fatigue behaviour of the material. The life, i.e. the fatigue life, depends on:

- the load acting on the guidance system
- the traverse speed of the guidance system
- the statistical probability of the first appearance of failure.

# **Basic rating life**

The basic rating life is reached or exceeded by 90% of a sufficiently large group of apparently identical flat cage guidance systems before the first evidence of material fatigue occurs (*Calculation example*, page 24).

Cw

$$L = \left(\frac{C_{w}}{P}\right)^{p}$$
$$L_{h} = \frac{8,33 \cdot 10}{H \cdot n_{osc}}$$

$$L_{h} = \frac{1666}{\bar{v}} \cdot \left(\frac{C_{w}}{P}\right)$$

Т m Basic rating life in 100000 m L<sub>h</sub> h Basic rating life in hours Cw Ν Effective dynamic load rating (page 21) Ρ Ν Equivalent dynamic load . Life exponent: flat cage guidance systems based on rollers: p = 10/3flat cage guidance systems based on balls: p = 3Н mm Distance between ends of stroke min<sup>-1</sup>

 $\begin{array}{cc} n_{oSC} & min^{-1} \\ Number of return strokes per minute \\ \overline{v} & m/min \end{array}$ 

Equivalent dynamic velocity.

$$\triangle$$

According to DIN 636-3, the equivalent dynamic load should not exceed P = 0.5  $\cdot$   $C_w!$ 

# **Operating life**

The operating life is defined as the life actually achieved by a flat cage guidance system. The operating life may deviate from the calculated life (basic rating life).

Possible reasons are wear and/or fatigue due to:

- misalignment between the guideways
- contamination of the guidance system
- inadequate lubrication
- reciprocating motion with very small stroke length (false brinelling)
- vibration during stoppage (false brinelling).

Since there is a wide range of possible installation and operating conditions, it is not possible to calculate the operating life of a flat cage guidance system precisely in advance. The most reliable method of achieving a good estimate of the operating life is by comparison with similar applications.

# Equivalent load and speed

The life calculation formulae are based on the assumption that the load and speed remain constant. Non-constant operating conditions can be taken into consideration by means of equivalent operating values. These have the same effect on the life as the loads occurring in practice.

#### Equivalent dynamic load

#### General formula

$$P = \sqrt[p]{\left(\int_{0}^{T} |v(t) \cdot F^{p}(t)| q^{t}\right) \left(\int_{0}^{T} |v(t)| dt\right)}$$

Load varying in steps

$$P = p \sqrt{\frac{q_1 \cdot F_1^{p} + q_2 \cdot F_2^{p} + \dots + q_z \cdot F_z^{p}}{100}}$$

Load and speed varying in steps

$$P = p \sqrt{\frac{q_1 \cdot v_1 \cdot F_1^p + q_2 \cdot v_2 \cdot F_2^p + \dots + q_z \cdot v_z \cdot F_z^p}{q_1 \cdot v_1 + q_2 \cdot v_2 + \dots + q_z \cdot v_z}}$$

Equivalent dynamic speed General formula

$$\overline{v} = \frac{1}{T} \int_{0}^{T} |v(t)| dt$$

Speed varying in steps

$$\bar{V} = \frac{q_1 \cdot v_1 + q_2 \cdot v_2 + ... + q_z \cdot v_z}{100}$$

Ν Equivalent dynamic load Life exponent: flat cage guidance systems based on rollers: p = 10/3flat cage guidance systems based on balls: p = 3 % Proportion of total duration Ν F Variable load

m/min

Variable speed m/min

Equivalent dynamic velocity.

# Static load carrying capacity

### Permissible load

The static load carrying capacity of a flat cage guidance system is limited by

- the permissible load on the flat cage guidance system
- the load carrying capacity of the raceways
- the permissible load on the screw connections
- the permissible load on the adjacent construction.

required for the application!

# **Basic static load ratings**

The basic static load ratings (*dimension tables*) are the loads under which a permanent deformation of the raceways and rolling elements occurs which corresponds to 1/10 000 of the rolling element diameter.

### Static load safety factor

The static load safety factor  $S_{0}$  indicates the security with regard to permissible permanent deformation in the bearing without affecting the guidance accuracy and smooth running of the bearing.

$$S_0 = \frac{C_{0v}}{P_0}$$

S<sub>0</sub> Static load safety factor C<sub>0w</sub> Effective static load rating (page 21) Ν P<sub>0</sub>

Maximum equivalent static load.

If high demands are placed on accuracy and smoothness of running, the static load safety factor should not be less than  $S_0 = 3.$ 

 $S_0 < 3$  for tensile and moment loading, the screw nnections must be checked!

# Calculation of effective load ratings

The basic dynamic and static load ratings C and C<sub>0</sub> given in the dimension tables are valid for a cage with a theoretical reference length of 100 mm. It is thus possible to make a direct comparison of the load carrying capacity of flat cages of various types and sizes.

For the effective cage lengths, the effective dynamic and static load ratings C<sub>w</sub> and C<sub>Ow</sub> must be calculated using the formulae below.

Needle roller flat cages:

$$C_{w} = C \cdot \left(\frac{L_{K} - 2e + t}{100}\right)^{7/2}$$
$$C_{0w} = C_{0} \cdot \frac{L_{K} - 2e + t}{100}$$

Ball flat cages:

$$C_{w} = C \cdot \left(\frac{L_{K} - 2e + t}{100}\right)^{0},$$
$$C_{0w} = C_{0} \cdot \frac{L_{K} - 2e + t}{100}$$

С

Ν Basic dynamic load rating for cage length of 100 mm (dimension table)  $C_0$ Ν

Basic static load rating for cage length of 100 mm (dimension table) C<sub>w</sub> N Effective dynamic load rating C<sub>Ow</sub> N Effective static load rating L<sub>K</sub> mm Length of cage (Figure 1) mm Distance between first or last pocket centre and end of cage (Figure 1, dimension table)

mm

Pocket pitch in basic cage element (Figure 1, dimension table) 7

Number of rolling elements per row.



The formulae for the effective load ratings will only give reliable results if the cage length  $L_{K} = (Z-1)t + 2e$  is based on a whole number of rolling elements per row. Formula for checking Z:

 $Z = \frac{L_K - 2e}{t} + 1 = \text{whole number!}$ 



Figure 1 · Dimensions for calculating the effective load rating

# Factors influencing the load carrying capacity

The basic load ratings given in the dimension tables are only valid under certain conditions. Correction factors should be used to take account of:

- deviating raceway hardness
- offset loading.

### Hardness - correction factor

If flat cages are used on raceways with a surface hardness <670 HV (58 HRC), the basic load ratings must be multiplied by the hardness factor  $f_H$  or  $f_{H0}$  (Figure 2).

# Load direction – correction factor

The effective basic load ratings of double row angled flat cages are dependent on the load angle  $\alpha$  at which the load acts on the guidance system.

The basic load ratings in the dimension tables are valid on condition that the load acts symmetrical to the cage shanks  $(\alpha = 0^{\circ})$ . For other load directions, the effective load ratings can be calculated using the formulae below and Figure 3.

### Basic dynamic load rating

$$C_{W} = f_{\alpha} \cdot f_{H} \cdot C \cdot \left(\frac{L_{K} - 2e + t}{100}\right)$$

C<sub>w</sub> N Effective dynamic load rating

Dynamic load direction factor

f<sub>H</sub> Dynamic hardness factor N

С

Basic dynamic load rating for cage length of 100 mm (dimension table)

Life exponent

for needle roller and cylindrical roller flat cages: p = 7/9for ball flat cages: p = 0,7.

# Basic dynamic load rating

$$C_{0w} = f_{\alpha 0} \cdot f_{H0} \cdot C_0 \cdot \left(\frac{L_K - 2e}{100}\right)$$

C<sub>Ow</sub> N Effective static load rating f<sub>H0</sub> – Static hardness factor  $C_0$ Ν Basic static load rating for cage length of 100 mm (dimension table)



Figure 2 · Hardness factors for raceway



Figure 3 · Load direction factors for angled flat cages

Offset loading of guideways in open arrangement Flat cages always travel half the distance of the moving guideway. They are therefore not subjected to uniform load in most cases.

The basic load ratings given in the dimension tables are only valid for uniform load distribution and concentric loading. If the cages are subjected to offset loading, the load carrying capacity and life can be calculated on the basis of the equivalent static or dynamic cage load (Figure 4). If the defined operating limits are exceeded, the cages are only subjected to partial load. This impairs the load carrying capacity and rigidity of the guidance system.

Open arrangement (*Design of bearing arrangements*, page 40, Figure 1).

Offset loading of guideways in closed arrangement

Guidance systems in a closed arrangement can be subjected to additional load and tilting moments.

Calculation of the equivalent cage load is a highly complex task. It is carried out by the INA engineering service using appropriate calculation programs (*Data sheet*, page 26).

Closed arrangement (*Design of bearing arrangements*, page 40, Figure 2).



Figure 4 · Equivalent load for flat cages subjected to offset loading with open arrangement

# Calculation example

Given data	
Guideway	M 6035 and V 6035
Flat cage	HW 20
Basic dynamic load rating for a cage length of 100 mm	C = 40300 N
Basic static load rating for a cage length of 100 mm	C <sub>0</sub> = 139500 N
Operating load, acting concentrically on the guidance system (factors $f_{\alpha}$ , $f_{\alpha0}$ , $k_F$ , $k_{0F} = 1$ )	$F_{B} = 25000 \text{ N}$
Distance between ends of stroke	H = 200 mm
Number of return strokes per minute	$n_{\rm osc} = 18  {\rm min}^{-1}$
Cage length	$L_{K} = 500 \text{ mm}$
Required	

L and  $L_h$  S<sub>0</sub>

Basic rating life	
Static load safety factor	

# Calculation

Checking the number of rolling elements per row (t, e, *dimension table*, page 104)

$$Z = \frac{L_{K} - 2e}{t} + 1$$

$$Z = \frac{500 - 8}{5, 5} + 1 = 90$$
(rounded whole number)

Effective dynamic load rating  $\mathrm{C}_{\mathrm{W}}$ 

$$C_{w} = C \cdot \left(\frac{L_{K} - 2e + t}{100}\right)^{7/9} = 140000$$

$$C_{w} = 40300 \cdot \left(\frac{497, 5}{100}\right)^{7/9} = 140000$$

Basic rating life L

$$L = \left(\frac{C_{\rm w}}{P}\right)^{10^{\prime} 3}$$
$$L = \left(\frac{140000}{25000}\right)^{10^{\prime} 3} = 310 \cdot 10^5 \,\rm{r}$$

Basic rating life Lh

$$L_{h} = \frac{8.33 \cdot 10^{5}}{H \cdot n_{osc}} \cdot \left(\frac{C_{w}}{P}\right)^{10}$$
  
. 8.33 \cdot 10^{5} \cdot 310 \cdot cosc

$$L_{h} = \frac{8,33 \cdot 10^{5} \cdot 310}{200 \cdot 18} = 72000$$

Static load safety factor S<sub>0</sub>

$$C_{0w} = C_0 \cdot \frac{L_K - 2e + 1}{100}$$

$$C_{0w} = 133500 \cdot \frac{497, 5}{100} = 664000 |$$

$$S_0 = \frac{C_{0W}}{P_0}$$

$$S_0 = \frac{664000}{25000} = 26,6$$

Computer program

# Computer program

The calculation method on pages 21 to 24 is intended mainly for the preliminary selection of flat cage guidance systems. It allows only an approximate calculation of the equivalent bearing load, since:

the formulae for calculating the element loads assume a system supported at two points.

# Complex influencing factors

The system is not in fact supported at two points and cannot be calculated by simple means.

A precise calculation of the equivalent load requires precise knowledge of the internal load distribution. This means that the loads on the individual rolling elements must be known.

In principle, the internal load distribution could be taken into consideration by means of load factors. Due to the large number of possible load combinations, however, an impracticably large number of diagrams would be required.

In many applications, not only the load carrying capacity and life of the system but also the rigidity and displacement under combined load are important. The latter values can only be calculated by simple manual means for very simple load cases.

# INA calculation program

INA has therefore developed a calculation program that can be used to calculate the load carrying capacity and rigidity under any combined load. The program takes into consideration the non-linear deflection curves of the rolling elements. The adjacent construction is assumed to be rigid.

INA offers the calculation of linear guidance systems using this program as a service.



Figure 5  $\cdot$  Internal load distribution under combined load

Computer program

The INA computer program for linear guidance systems calculates values such as:

- the static load safety factor
- the basic rating life
- the displacement resulting from the elasticity of the bearing arrangement.

For calculation, the following data must be given for each load case in a co-ordinate system with a freely selectable origin (Figure 6):

- the geometry and position of the monorail guidance system including the position of the drive axis
- the external load components (any number) and the positions of their loading points in the co-ordinate system
- the components of shear-free moments
- the masses (any number) and the positions of their centres of gravity in the co-ordinate system
- the kinetic values
- the durations of particular steps.

### Data sheet

The geometry and loads can be easily described using the data sheet below (Figure 6):

- two masses and two loads are given in the data sheet as an example. The other masses and loads should be entered on the same basis
- if there are several planes, please complete a separate data sheet for each plane
- if there are planes or guideways at an angle to the planes of the co-ordinate system, please provide a diagram showing the positions of the planes
- the data must be given separately for each load case.

Calculation can also be carried out on the basis of a technical drawing containing the required dimensions and loads.

Guidano	ce geometr	у	Position	of guidanc	ce system	Kinetic	values		Duratior	n of particu	lar step
$L_{K}$		mm	$\mathbf{x}_{LF}$		mm	a <sub>max</sub>		m/s <sup>2</sup>	q		%
b <sub>1</sub>		mm	У <sub>LF</sub>		mm	v <sub>max</sub>		m/min	Shear-fr	ee momen	ts
b <sub>2</sub>		mm	$z_{\text{LF}}$		mm	V		m/min	$M_{X}$		Nm
h <sub>1</sub>		mm				Н		mm	My		Nm
h <sub>2</sub>		mm				n <sub>osc</sub>		min <sup>-1</sup>	$M_{z}$		Nm
Mass 1			Mass 2			Load 1			Load 2		
m <sub>1</sub>		kg	$m_2$		kg	F <sub>1 x</sub>		Ν	F <sub>2 x</sub>		Ν
						F <sub>1y</sub>		Ν	F <sub>2y</sub>		Ν
						F <sub>1z</sub>		Ν	F <sub>2z</sub>		Ν
Centre o	of gravity 1		Centre o	of gravity 2		Loading	point 1		Loading	point 2	
x <sub>m 1</sub>		mm	x <sub>m 2</sub>		mm	x <sub>F1</sub>		mm	x <sub>F2</sub>		mm
Y <sub>m 1</sub>		mm	Y <sub>m 2</sub>		mm	УF 1		mm	УF 2		mm
z <sub>m 1</sub>		mm	z <sub>m 2</sub>		mm	z <sub>F1</sub>		mm	z <sub>F2</sub>		mm







Figure 6 · Loading data for flat cage guidance systems

# Preload

Preload:

- increases the rigidity and guidance accuracy of flat cage guidance systems
- reduces the high loads on the rolling elements at the ends of cages under moment load; this increases the moment load carrying capacity of the guidance system
- affects the displacement resistance and the operating life.

### Preload value

Preload generally has a positive effect at a level of 0,02 to 0,03  $\cdot$  C<sub>0</sub>. The optimum preload can be determined using the calculation programs of the INA data service.

If the preload is too low or too high, it can lead to uncontrolled displacement of the flat cage (cage creep).

### Influence of the adjacent construction

If the rigidity is to be fully utilised, the adjacent construction must have sufficient rigidity and geometrical accuracy (*Design* of bearing arrangements). If the adjacent construction can be easily deformed or is not accurate, angular defects can occur between the raceways. The rolling elements may then be subjected to load at the ends only. In these cases, the rigidity of the system is not increased. Furthermore, the operating life may be reduced due to edge loads and cage creep may be induced.

# Setting the preload

The preload can be measured and set by various methods:

- by pressure screws with a defined tightening torque (Table 1, page 29)
- by the table displacement friction F<sub>RV</sub> (formula) when using pressure screws or guideways with an adjusting gib (page 29)
- by measuring the deformation of the adjacent construction once the correct deformation has been determined using the hydraulic adjustment rail (page 30).

$$\mathsf{F}_{\mathsf{RV}} = \frac{\mathsf{C}_{\mathsf{0W}}}{40000}$$

F<sub>RV</sub> N Table displacement friction

C<sub>0w</sub> N Effective static load rating

The formula is correct under the following conditions:

- preload 2,5% C<sub>0</sub>
- guidance system lubricated, no operating load
- movement of approx. 0,05 m/s.

# Pressure screws

Under low load ( $S_0 > 5$ ), the preload can be applied by pressure screws on the back face of the guideway. The pressure screws (Figure 1, Table 1) should:

- have an even contact surface (dowel screw to ISO 4 026, DIN 913)
- be arranged in the spaces between the fixing screws and at the ends of the guideways.

Table 1 · Sizes of pressure screws/ righterning torques	Table 1 ·	Sizes of	pressure	screws/	/Tighteni	ing torques	
---	-----------	----------	----------	---------	-----------	-------------	--

Guideway	Pressure screw		Tightening torque <sup>1</sup> )
	Size	Spacing mm	M <sub>A</sub> Nm
M / V 3015	M 4	40	0,34
M / V 4020/5025	M 6	80	1,2
M / V 4525	M 6	80	1
M / V 6035	M 8	100	2,9
M / V 6535	M 8	100	3,5
M / V 7040	M10	100	5,7
M / V 8050	M12	100	7,7
M / V 8550	M12	100	7,3

<sup>1</sup>) Preload = 2,5% C<sub>0</sub> (*dimension table*).



Figure 1 · Arrangement of pressure screws

### Guideways with adjusting gib

Under high loads (S $_0$  <5), or where high requirements are placed on rigidity, it is advantageous to use guideways ML with an adjusting gib. With these guideways, the preload is applied evenly over the whole length of the guidance system and precise adjustment is achieved.

### Hydraulic adjustment rail

This rail can be used to determine the deformation of the adjacent construction and the preload of M, ML and V guideways in closed arrangements. The deformation is specific to the particular design and must only be determined once for a machine type (Figure 2).

Hydraulic adjustment rails are of a modular construction and can be matched to practically any application. Such measurements are offered by INA as a service.

# Description of the principle

- At initial assembly, replace one guideway and one flat cage by the hydraulic adjustment rail.
- Apply the necessary preload using the hydraulic adjustment rail.
- Measure the deformation (deflection) of the adjacent construction under the preload.
- Set the preload of the completely reassembled guidance system in accordance with the measured deformation.



Figure 2  $\cdot$  Preload with hydraulic adjustment rail

# Friction



At all traverse speeds, flat cage guidance systems have a low friction coefficient. The friction curve is uniform and the starting friction is slight. As a result, flat cage guidance systems run without stick-slip.

# Friction coefficient

- The total friction consists of:
- rolling and sliding friction in rolling contacts
- Iubricant friction
- seal friction.

The friction coefficient is dependent on:

- the load
- the preload
- the traverse speed
- the lubricant
- the lubrication condition
- the temperature
- the installation accuracy.

The influence of these factors may be reciprocal, act in a single direction or counteract each other. The friction coefficient is characterised by the ratio of displacement force to normal load. Under normal conditions, the friction coefficient of rolling element guidance systems is between 0,001 and 0,004.

Fresh grease will cause a temporary increase in the friction coefficient. However, the friction coefficient returns to its original lower value after a short running-in period. This applies to both initial operation and regreasing.

# Seal friction

The total friction is increased by contact type seals.

The seal friction is at its greatest with new guidance systems.

During the running-in phase, the seal lip geometry becomes worn at particular locations and adapts to the guideway; the seal friction decreases as a result.

# **Displacement resistance**

The displacement resistance is defined as follows:

 $F_R = \mu \cdot F$ 

 F<sub>R</sub>
 N

 Displacement resistance

 F
 N

 Load on the guidance system

 µ

Friction coefficient.

The friction coefficient is dependent on the ratio C/P and is therefore a function of the load (Figure 1).



Figure 1 · Relationship between friction coefficient and load ratio C/P

# Rigidity

In flat cage guidance systems, the load-bearing rolling elements are needle rollers, cylindrical rollers or balls.

### Contact geometry

In guidance systems with needle or cylindrical rollers, the rolling elements are in line contact, while the rolling elements in guidance systems with balls are in four point contact. Due to the larger contact surface, guidance systems with needle or cylindrical rollers are significantly more rigid than guidance systems with balls (Figure 1).

Needle and cylindrical rollers have profiled ends, i.e. the outside surface of the roller curves has a curved surface running towards the ends. This reduces the edge stresses at the ends of the rolling elements. This has practically no effect on the load carrying capacity since the effective contact length between the roller and raceway is hardly reduced.

#### Elastic deformation

Flat cage guidance systems have extremely high rigidity. However, the operating load does lead to elastic deformation at the contact points. The deformation and rigidity are subject to the formulae on page 33.



This does not take account of elastic deformation of the adjacent construction and screw connections or of settling effects, etc. Since the adjacent construction is not completely rigid, the elastic deformation can in practice be somewhat higher.

In closed arrangements with M and V guideways, the rigidity of the guidance system can be further increased by preload (*Preload*, page 28).



Figure 1 · Rolling element type and elastic deformation - deflection (rigidity)

# Rigidity of flat cage guidance systems

The rigidity of a flat cage guidance system is determined by the ratio between the load and elastic deformation.

$$C_L = \frac{F}{\delta}$$

 $C_L$ N/µm Rigidity of flat cage guidance system F Ν Operating load  $\delta \qquad \mu m \\ \text{Elastic deformation at contact points.}$ 

# Calculation of rigidity

The elastic deformation depends on the load, number, length and geometry of the rolling elements.

Flat cage guidance systems with line contact

$$\delta = K \cdot \frac{(F \neq Z)^{0,838}}{L_{W}^{0,605}}$$
$$C_{L} = \frac{1}{K} \cdot F^{0,162} \cdot Z^{0,838} \cdot L_{W}^{0,605}$$

Flat cage guidance systems with four point contact

$$\delta = K \cdot \frac{(F/Z)^{2/3}}{D_w^{1/3}}$$
$$C_L = \frac{1}{K} \cdot F^{1/3} \cdot Z^{2/3} \cdot D_w^{1/3}$$

δ μm Elastic deformation at contact points. Convergence of two raceway planes Κ Factor for calculating elastic deformation, dependent on the product type (Table 1) F Ν Operating load Ζ Number of rolling elements per row L<sub>w</sub> mm Length of rolling element mm  $C_L$  N/ $\mu$ m Rigidity of flat cage guidance systems D<sub>w</sub> m Ball diameter. mm

Table 1	Tune feater fer	a a la ulatina	alaatia	defermention
Table T.	Type factor for	calculating	elastic	deformation

Guideway Type	Type factor K
F	0,092
	0,087
	0,049
Calculation example	M 6035 and V 6035

Guideway	M 6035 and V 6035
Flat cage	HW 20×500.
Operating load	25000 N
Number of rolling elements per row Z	90
Rolling element length L <sub>w</sub>	9,8
Type factor K (Table 1)	0,092.

Calculation of elastic deformation:

$$\delta = K \cdot \frac{(F/Z)^{0,838}}{L_{W}^{0,605}}$$

$$\delta = 0,092 \cdot \frac{(25000 / 90)^{0,838}}{9,8^{0,605}} = 2,6 \ \mu m$$

Calculation of rigidity:

$$C_{L} = \frac{25000}{2.6} = 9600 \,\text{N/} \,\mu\text{m}$$

# Lubrication

The performance capability of a machine is determined to a considerable extent by the flat cage guidance systems used. Lubrication constitutes an important component of the flat cage guidance system.

Lubricants (either grease or oil) perform the following functions:

- they reduce friction
- they minimise wear
- they prevent corrosion
- they give protection against contamination
- they increase the operating life of the guidance system.

#### Early selection

Lubrication is a design element! The lubricant and lubrication method must therefore be selected in the development phase of the machine. Experience has shown that considerable problems occur if the lubrication of the guidance system is only selected when the design is complete.

The lubricant requirement has an increasingly important role in guidance systems. A carefully considered lubrication concept is therefore indicative of a modern flat cage guidance system. The lubricant consumption should be as small as possible, especially in the case of oil lubrication.

It will be determined on the basis of technical and economic considerations whether oil or grease should be used and which lubrication method should be applied.

#### Advantages of grease lubrication

- Very little design work is required for relubrication devices if a central lubrication plant is not required
- Relubrication intervals up to one year
- Emergency running characteristics due to the thickener in the grease
- Good support to the seal.

#### Advantages of oil lubrication

- Heat dissipation
- Very good lubricant distribution
- Almost complete replacement of the lubricant when relubrication is carried out
- Contaminant particles are washed out.

Furthermore, oil lubrication is advisable where the adjacent machine elements are already supplied with oil.

### Ensuring correct function

Flat cage guidance systems must be relubricated at appropriate intervals. The length of the interval is essentially dependent on:

- the speed
- the load
- the temperature
- the stroke length
- the environmental conditions.

The shorter the lubrication intervals, the easier it is to justify substantial expenditure on lubrication devices on economic grounds. Where the intervals are long, lubrication by hand or using semi-automatic devices can be advantageous.

#### Environmental protection

Any lubrication method involves loss of lubricant. The lubricant consumed must be collected and disposed of by methods which help to protect the environment.



The handling and use of lubricants is governed by national regulations for environmental protection and health and safety at work as well as information from lubricant manufacturers. These specifications must be observed.

#### Flat cage guidance systems

Flat cage guidance systems require only small quantities of lubricant. They are supplied with a preservative as standard. The preservative is compatible with oils and greases.

The guidance systems operate almost exclusively under mixed friction conditions, especially at low speeds. Doped lubricants (type P to DIN 51502) should be used in preference.



Drilling oils or other coolant emulsions must not be used for lubrication. These have the effect of thinning the lubricating oils and can lead to corrosion in certain circumstances.

Lubricants with solid additives must not be used.

With M and V guideways, lubricant is generally introduced through the lateral gap between the upper and lower guideway. If this is not possible, for example if the guidance system is fitted in a vertical position, guideways with a relubrication facility can be supplied (special design).

# **Oil lubrication**

Lubricating oils CLP to DIN 51517 and HLP to DIN 51524 should be used in preference. For operating temperatures between 0 °C and +70 °C, the viscosity should be between ISO VG 32 and ISO VG 68. For low-temperature operation, oils to ISO VG 10 or ISO VG 22 should be used. Slideway oils CGLP up to ISO VG 220 may be used.

Oil impulse or oil drop lubrication is recommended. In highly contaminated environments, pneumatic oil lubrication is particularly advantageous. It creates a slight pressure in the guidance system, reinforcing the effectiveness of the seals.

The feed mechanism for the lubricating oil must be selected with reference to the installation position (Figure 1) such that all the rows of rolling elements are provided with lubricant.

### Compatibility and miscibility

Before lubricating oils are used, their behaviour with respect to plastics, elastomers, light metals and non-ferrous metals must be checked under dynamic conditions at operating temperature if no past experience or data from the manufacturer is available.

In general, oils with a mineral oil base and with the same classification are miscible with each other. However, the viscosities should be within one ISO VG class of each other.

Synthetic oils should always be checked for their miscibility and compatibility before use. If there is any doubt, please consult the INA engineering service.

### Initial operation

The raceway and cage of the flat cage guidance system should be oiled before initial operation and protected against solid and liquid contaminants.



Figure 1 · Installation position of flat cage guidance systems
### Grease lubrication

INA recommends lithium soap greases with a mineral oil base. The base oil viscosity should be between:

ISO VG 150 and ISO VG 220.

For high loads ( $S_0 < 8$ ), the following are absolutely necessary:

EP doped greases with a base oil viscosity of ISO VG 220. A lubricating grease KP2N-20 to DIN 51825 is recommended for initial greasing.

### Miscibility

Greases may be mixed providing the following preconditions are met:

- they have the same base oil type
- their thickener types must match
- they must have similar base oil viscosities (the difference must be no more than one ISO VG class)

their consistencies (NLGI class) must match.

If there is any doubt, please consult the INA engineering service.

### Storage

Experience shows that lubricating greases with a mineral oil base can be stored for up to 3 years. The following preconditions apply:

- the storage room is protected against outside influences
- the temperature is between 0 °C and +40 °C
- the relative humidity is less than 65%
- the lubricants are protected against chemical agents (vapours, gases, fluids).

After long storage periods, friction may be considerably higher for a short time compared to freshly greased flat cage guidance systems. The lubricity of the grease may also have deteriorated. It is the user's responsibility to follow the directions given by the lubricant manufacturer.

### Initial operation and initial grease quantity Without relubrication device

The initial grease quantity according to Table 1 should be introduced evenly from both sides into the cage pockets and distributed over the cage surfaces. A thin film of grease should also be applied to the guideway raceways.

The guidance system must be protected against solid and liquid contaminants before and during fitting.

### With relubrication device

Before fitting, a thin film of grease must be applied to the cage and raceways. The guidance system is then fitted and the feed pipe is filled with grease before the initial grease quantity according to Table 1 is introduced.

During greasing, guidance systems must be moved the whole stroke length several times to ensure uniform grease distribution.

Flat cage Series	Initial grease quantity g per 100 mm cage length <sup>1</sup> )
HW 10	0,2 to 0,6
HW 15 <sup>2</sup> ) / FFW 2025 / FF 2025 ZW	0,2 to 0,6
HW 20 <sup>2</sup> ) / FFW 2535 / FF 2535 ZW	0,2 to 1
HW 25 <sup>2</sup> ) / FFW 3045 / FF 3045 ZW	0,3 to 1,3
HW 30 <sup>2</sup> ) / FFW 3555 / FF 3555 ZW	0,3 to 2,1
HRW 08	0,2 to 0,6
HRW 50	0,2 to 1,5
HRW 70	0,2 to 3,5
HRW 100	0,2 to 6,6
H 10 <sup>2</sup> )	0,1 to 0,3
H 15 <sup>2</sup> )	0,1 to 0,5
H 20 <sup>2</sup> )	0,2 to 0,7
H 25 <sup>2</sup> )	0,2 to 1,1
FF 2010	0,1 to 0,3
FF 2515	0,1 to 0,5
FF 3020	0,2 to 0,7
FF 3525	0,2 to 1,1

#### Table 1 · Initial grease quantity for flat cages – guide values

 For high speeds, the lower value should be selected; for low speeds, the quantity should be close to the higher value.

<sup>2</sup>) For cages with damping, use 80% of the stated value.

### Relubrication

The relubrication quantity is approximately 50% of the initial grease quantity. Relubrication should be carried out with several partial quantities at shorter intervals in preference to a single regreasing at the end of the relubrication interval.

#### Relubrication interval

The relubrication interval and quantity can only be determined precisely under operating conditions since it is not possible to calculate all the influences in advance. An observation period of adequate length must be allowed.

The relubrication interval  $t_{fR}$  should be no more than 1 year even if the formula gives a value greater than this:

$$t_{fR} = t_f \cdot K_P \cdot K_W \cdot K_U$$

h

 $t_{fR} \qquad h \\ Guide \mbox{ value for relubrication interval in hours}$ h

tf

Basic lubrication interval in hours (Figure 2)

 $K_{\text{P}},\,K_{\text{W}},\,K_{\text{U}}$  Correction factors for load, stroke and environmental influences (page 38).

### Basic lubrication interval

The basic lubrication interval t<sub>f</sub> (Figure 2) is valid under the following conditions:

- bearing temperature t <+70 °C</p>
- load ratio  $C_0/P = 20$
- Iubrication with high quality lithium soap grease
- no disruptive environmental influences
- stroke ratio between 1 and 10.

The speed parameter is defined as follows:

Speed parameter:  $\frac{60}{\overline{v}}$  · K<sub>LF</sub>

v m/min Equivalent dynamic speed K<sub>IF</sub>

Bearing factor (Table 2).



Figure 2 · Calculation of basic lubrication interval

### Table 2 · Bearing factor KIF

Flat cage	HW, HRW, FFW	H, HR, FF
Bearing factor K <sub>LF</sub>	1,5	1

### Load correction factor ${\sf K}_{\sf P}$

The correction factor  $K_P$  takes into consideration the greater strain on the lubricating grease at a load ratio of  $C_0/P < 20$ .



The values given in Figure 3 are valid only for high quality lithium soap grease. The preload must be taken into consideration.

### Stroke ratio correction factor $K_W$

The correction factor  $K_{\rm W}$  (Figure 4) takes into consideration the raceway length to be provided with grease. It is dependent on the stroke ratio.

If the stroke ratio is <1 or >10, the relubrication interval must be shortened in order to reduce possible fretting corrosion.

If the stroke is very short, the relubrication interval  $t_{fR}$  may be shorter than the calculated value. In such cases, the use of special lubricating greases is recommended. Please consult the INA engineering service.

The stroke ratio is defined as follows:

Stroke ratio:	$\frac{L_{K}}{H}$
---------------	-------------------

L<sub>K</sub> mm Length of cage H mm

Distance between ends of stroke.

### Environmental correction factor K<sub>U</sub>

The correction factor  $K_U$  (Table 3) takes into consideration the effect of oscillations, vibration (leading to fretting corrosion) and shocks. These subject the grease to additional strain.

All calculations are invalid if cooling lubricants or moisture penetrate the bearing.

### Lubricating grease operating life

If a linear system cannot be relubricated, the operating life of the lubricating grease is then the decisive factor. In most applications, the guide value is:

 $t_{fG} = 2 \cdot t_{fR}$ 

 $t_{fG} \qquad h \\ \text{Guide value for grease operating life in hours}$ 

 $t_{fR}$  h Guide value for relubrication interval in hours.









Table 3 · Environmenta	correction factor K <sub>II</sub>
------------------------	-----------------------------------

Environmental influences	K <sub>U</sub>
slight	1
moderate	0,8
severe	0,5

## Sealing Operating limits



### Sealing

The type of sealing or shielding is of decisive importance in ensuring problem-free operation and a long operating life of flat cage guidance systems.

Depending on the operating conditions and requirements, the following sealing arrangements can be used:

- wipers (*Accessories*, page 90)
- sealing strips
- a complete solution for M and V guideways with conventional wipers and integral sealing strips (suffix ..ZZ, ..PP)
- seals in the adjacent construction.

For most applications, wipers will be sufficient to keep the raceways clean.

The wipers must be in contact with raceways over the whole stroke length of the system.

If the guideways are subjected to severe contamination or aggressive media, special measures must be used.

### **Operating limits**

#### Acceleration

The permissible acceleration of flat cage guidance systems depends on:

- the load
  - the lubrication
  - the accuracy achieved in fitting.

For high accelerations, HYDREL flat cages made from light metal are particularly suitable. These cages:

- have low mass and high strength
- are suitable for accelerations up to 250 m/s<sup>2</sup>
- are suitable for speeds up to 15 m/s.

For accelerations of more than 100 m/s<sup>2</sup>, one guideway should be extended by the amount of the stroke length and both end pieces should be fixed to the shorter guideway. The cage will therefore be guided between the raceways in all positions. If wipers are used, an end piece should be fitted between the wiper and guideway.

#### Operating temperatures

Guideways without wipers in conjunction with metal flat cages are suitable for continuous temperatures up to +150 °C if appropriate lubrication is used.

Guideways which operate continuously at higher temperatures must be heat stabilised. In such cases, please consult the INA engineering service.

Needle roller and cylindrical roller flat cages made from plastic can be used at temperatures up to +120  $^\circ\text{C}.$ 

Guideways with wipers must not be subjected to operating temperatures above +100 °C.

## **Design of bearing arrangements**

### Installation arrangements

Guideways can be fitted in open or closed arrangements.

### Open arrangements

The open arrangement is a locating/non-locating bearing arrangement. It comprises (Figure 1):

- M and V guideways with the associated angled flat cage
- J and S guideways with the associated flat cage.

This arrangement:

- is mainly used for applications with loads acting concentric and vertical to the guidance plane and a large guidance base
- is very easy to install.

### Closed arrangement M/V

The closed arrangement is a locating/locating bearing arrangement. It comprises (Figure 2):

two M and two V guideways with the associated angled flat cages.

This arrangement:

- is mainly used for applications with all types of load directions and moment loads
- allows a small guidance system base
- allows all types of operating positions
- can be preloaded (*Preload*, page 28).

### Closed arrangement LUE

The closed arrangement LUE is a locating/non-locating bearing arrangement. It comprises (Figure 3):

- one M and one V guideway
- one J and one S guideway
- the associated flat cages
- counterstay parts matched to the guidance systems.
- This arrangement:
- is mainly used for applications with all types of load directions and moment loads and very high accuracy requirements
- is intrinsically preloaded
- can compensate for thermal expansion of the adjacent construction without distortion.



Figure 1 · Open arrangement M/V, J/S



Figure 2 · Closed arrangement M/V



Figure 3 · Closed arrangement LUE

### Calculation of length

When selecting the dimensions of guideways and cages, the technical requirements or design features can be decisive. In both cases, the stroke H is an important factor.

### Calculation in accordance with technical requirements Cage length

The load carrying capacity and rigidity of the guidance system are determined by the cage size and cage length.

For concentric loading, the size and length of the flat cages can theoretically be selected within wide limits. As a guide value for determining the cage length  $L_K$ , the following principles have proved appropriate in both technical and economic terms:

- for an open arrangement
- cage length L<sub>K</sub> ≧1,5 H
- for a closed arrangement
  - cage length  $L_K ≥ H$

For offset loading and moment loads, the largest possible cage lengths and distances between the guideways should be selected. This gives more uniform load distribution.

### Guideway length

For the guideway length L, the cage length  ${\rm L}_{\rm K}$  and stroke H are decisive. It must be taken into consideration that:

- for kinematic reasons, the flat cage always travels half the distance of the moving guideway (Figure 4)
- the whole length of the cage remains between the two guideways irrespective of the position of the moving guideway
- the wipers must remain in contact with the raceway (Figure 5).

Guideway without wipers (Figure 4).

$$L = L_{K} + \frac{H}{2}$$

Guideway length

L<sub>K</sub> mm Cage length

H mm

Distance between end points of stroke.

Guideways with wipers (Figure 5).

$$L_{A} = L_{K} + 3 \cdot \frac{H}{2} + 10 \,\text{mm}$$

L<sub>A</sub> mm Length of guideway with allowance for wipers.



Figure 4 · Movement of guideway and cage



Figure 5 · Length of guideway with wipers

Calculation in accordance with design features

The maximum possible guideway length L,  $L_A$  and the stroke H are often predetermined by the design. The possible cage length  $L_K$  is therefore already defined.

### Cage length

Kinematically possible cage length in guidance systems without wipers (Figure 6).

$$L_{K} = L - \frac{H}{2}$$

Kinematically possible cage length in guidance systems with wipers (Figure 7).

$$L_{K} = L_{A} - 3 \cdot \frac{H}{2} - 10 \text{ mm}$$

L mm Guideway length

L<sub>A</sub> mm Guideway length

L<sub>K</sub> mm Cage length

H mm

Distance between ends of stroke.



If a guidance system is designed for long strokes such that the whole length of the flat cage does not remain between the guideways, the formulae are not valid!

In such cases, single-piece metal flat cages should be used and raceway lead areas should be provided on the shorter guideway (suffix E2). If necessary, cages with positive control can be used or, alternatively, it should be checked whether a monorail guidance system with rolling element recirculation would be more suitable!

### End pieces / wipers

End pieces or wipers retain the cage in its nominal position at the ends of the stroke.

Two end pieces or wipers should be fitted per cage. If this is not possible, the function of the end pieces must be fulfilled by elements of the adjacent construction.



Under special operating conditions, for example where the stroke length changes but remains constant over the long term or where extreme loads are applied at the ends of the stroke, the normal end pieces can no longer fulfil the cage positioning function. Such applications can be fulfilled by the use of integral positive control of the cage (series MVZ, page 62).



Figure  $6 \cdot \text{Cage}$  length of guideways without wipers



Figure 7 · Cage length of guideways with wipers



Figure 8 · Impermissible arrangement of end pieces or wipers



### Location of guideways

The guideways are fixed to the adjacent construction using fixing screws. The possible guideway hole types are shown in Figure 9 and the *dimension tables*. Hole patterns are shown on page 44.

Guideways of series M and V of standard lengths held in stock have holes of type 15. Insert nuts ESM can be used to allow screw mounting of these guideways on the basis of hole type 03 (Figure 10).

The insert nuts must be ordered separately (*Accessories*, page 93). They are included loose in the delivery and must be fixed in the counterbore by means of adhesive (Figure 10).



Figure 9  $\cdot$  Hole types



Figure 10  $\cdot$  Location of M and V guideway from one side

### Hole patterns

Unless stated otherwise, guideways are supplied with a symmetrical hole pattern. On request, an asymmetrical hole pattern is also possible where  $C_5 \ge C_{5\text{min}}$  and  $C_6 \ge C_{6\text{min}}$ . For a symmetrical hole pattern,  $C_5 = C_6$  (Figure 11a).

For an asymmetrical hole pattern,  $C_5 \neq C_6$  (Figure 11b).

 $\triangle$ 

Note the position of the spacing  $C_5$ ! Definition of the position of  $C_5$ : see Figure 12. Suffix LA ( $C_5/C_6$ )

### Calculation of hole patterns

The number of pitches between holes is the nearest whole number equivalent to

 $n = \frac{L - (2 \cdot C_{5min})}{C_4}$ 

Dimensions C<sub>5</sub> and C<sub>6</sub>

$$C_5 + C_6 = L - n \cdot C_4$$

Guideways with symmetrical hole pattern

$$C_5 = C_6 = \frac{1}{2} \cdot (L - n \cdot C_4)$$

mm

Number of holes

x = n + 1

L mm Guideway length

C<sub>4</sub>

Hole pitch

 $C_{5,}\,C_{6}\,$  mm Distance between start or end of guideway and nearest hole

 $\rm C_{5\ min}, \rm C_{6\ min}\ mm$  Minimum value for  $\rm C_{5}$  and  $\rm C_{6}$  (dimension table)

n

Maximum possible number of hole pitches

x – Number of holes.



Figure 11 · Symmetrical (a) and asymmetrical (b) hole pattern for guideway with one row of holes



Figure 12  $\cdot$  Position of distance between first hole and start of guideway C<sub>5</sub>



### Accuracy of adjacent components

The correct function of flat cage guidance systems is decisively determined by the accuracy of the adjacent surfaces. The requirements for geometrical and positional accuracy increase with those for accuracy and smooth running.

### Parallelism and perpendicularity

Any parallelism defects in the adjacent surfaces should not be greater than those in the corresponding guideways. The adjacent and rear face surfaces of the adjacent components for M and V guideways must be machined perpendicular to each other.

 $\angle$  The deviation must not exceed ±0,3 mrad!

### Height variation of adjacent surfaces

In order to ensure uniform load distribution over the length of the rolling elements, the height variation  $\Delta h$  must not be exceeded (Figure 13 and 14 and formulae).

In an open arrangement, the height variation  $\Delta h$  can be matched to the non-locating bearing side using an intermediate layer or set to the locating bearing side using the adjusting gib guideway ML (Figure 14).

Permissible height variation for needle roller flat cages

 $\Delta h < 0, 1 \cdot b$ 

b mm Distance between guidance systems.

Permissible height variation for cylindrical roller flat cages

 $\Delta h < 0.3 \cdot b$ 

### Surface roughness

In order to achieve the required geometrical accuracy, the necessary measurement basis as well as good location of the guideways, the surfaces should be precision machined.

In order to avoid location defects, the holes must be carefully deburred.



Figure 13 · Height variation in closed arrangement



Figure 14 · Adjustment of the height variation in open arrangement

## Fitting

Guideways are high precision machine elements. These products must be handled very carefully before and during fitting. Their trouble-free operation depends largely on the care taken during fitting.

### Delivered condition and storage

Guideways

- are supplied with a preservative and are packed in anticorrosion paper.
- parts dimensionally matched to each other are packed together. If the parts must be packed in several units due to weight, these are marked accordingly.
- Flat cages
  - are supplied with a preservative and packed with anticorrosion protection.

#### Storage

- Packaging must be kept closed
- Parts must be stored in dry, clean rooms with the temperature as constant as possible
- The maximum relative humidity is 65%.

### Removal from packaging

Perspiration from handling leads to corrosion. Hands must be kept clean and dry; protective gloves should be worn if necessary.

#### Guideways

- Guideways should only be removed from their packaging immediately before fitting
- They should be lightly oiled in order to prevent corrosion during fitting
- Guideways matched to each other are supplied as sets. Attention must be paid to the marking:
  - parts with the same set number must be fitted in the same guidance system
  - where the arrangement of guideways within a set having the same set number must be defined more specifically, this is indicated using letters (Figure 1).

### Flat cages

When unpacking and fitting metal cages, especially longer cages, it must be ensured that they are not bent.



Figure 1 · Marking of single and multi/piece guideways

### **Closed arrangements**

Guideways packed in pairs and marked with the same number must only be fitted in the same guidance unit. The locating faces of the guideways are not marked and can be identified by the larger edge break.

Guideways must be clamped against the back locating face before the fixing screws are fully tightened.

### Fitting of closed arrangements

 $\triangle$  The sequence of work steps must be adhered to.

- Screw the guideway pair ① rigidly in place (Figure 2)
- Check the parallelism of the guideway pair (Figure 3)
- Locate the opposing guideway <sup>(2)</sup> (Figure 4)
- Locate the adjustment guideway ③ (Figure 4). In order to allow adjustment of the guideway, do not fully tighten the fixing screws
- Push the guidance system in longitudinally (Figure 4)
- Push the cages between the guideways and position precisely
- Preload the adjustment guideway ③ using pressure screws ④ (Figure 4) or an adjusting gib (ML guideway, Figure 5 to 7) to approximately twice the required value and loosen again, in order to preempt settling effects
- Set the preload to the required value
- Tighten the fixing screws fully
- Fit the wipers and/or end pieces.

### Preloading using pressure screws

- Tighten the pressure screws to the required tightening torque in two stages
- Secure the pressure screws by means of a locknut or screw lock.



The movable part must be moved such that each pressure screw is supported by the flat cage when the screws are fully tightened.



Figure 2 · Fitting the guideway pair to the table



Figure 3 · Checking the parallelism



Figure 4 · Fitting of closed arrangements

Setting the preload on guideways with an adjusting gib Setting

- Push the unhardened, ground gib under the guideway ML and set the guidance system free from clearance (Figure 5)
  - the gib protrudes approximately 30 mm from each end of the guideway.
- Shorten the gib on the adjustment side of the guideway such that it is recessed about 3 mm from the end face of the guideway (Figure 5).
- Shorten the gib on the other side such that it is flush with the end of the guideway.
- Drive in the gib using a copper drift and light hammer blows in order to preload the system (Figure 6)
  - moving the gib by 1 mm changes the adjustment by 15  $\mu\text{m}.$
- Check the preload.
- After setting, fix the position of the gib using the hexagonal socket screw on the end face of the guideway (Figure 7).

### **Open arrangements**

Fitting should be carried out by a procedure similar to that for *Closed arrangements*. However, it is not necessary to adhere strictly to the sequence of steps.



Figure 5 · Inserting and shortening the gib



Figure 6 · Preloading the system



Figure 7 · Fixing the setting device

## Product range



HYDREL/EGIS M and V guideways with angled needle roller flat cage or angled cylindrical roller flat cage

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### Features

### Flat cage guidance systems

### consist of:

- guideways of type M and V
- angled flat cages with a large number of rolling elements
- are used as linear locating bearings
- are highly suitable for limited stroke lengths
- have high rigidity and load carrying capacity whilst requiring very little space
- have low, uniform friction
- have high, consistent accuracy throughout their operating life
- can be supplemented by accessories
- are also available with sealing strips (*dimension table*).

### Guideways M, V

- are made from through hardened steel with a minimum hardness of 670 HV
- have precision ground raceways and locating faces
- are produced with various hole types
- have threaded holes on the end faces for fitting end pieces or wipers.

### Angled needle roller and angled cylindrical roller flat cages

- are double row cages with the shanks at right angles to each other
- are made from metal or plastic
  - metal cages are lightweight components with high strength. They are suitable for more demanding conditions such as high acceleration, high temperatures and where the ends of the cage are exposed at some point in operation
  - plastic cages are an economical solution for less demanding operating conditions.

### Accessories

- end pieces prevent the cage creeping out of the load zone
- wipers protect the raceways against contamination under normal operating conditions
- insert nuts allow the conversion of holes of type 15 to threaded holes.



- guideways M, V
- angled needle roller flat cages FFW, HW or HGW or angled cylindrical roller flat cages HRW

### HYDREL/EGIS guideways



- for angled needle roller and angled cylindrical roller flat cages standard lengths from 100 to 1000 mm in steps of 100 mm; hole type 15 for fixing
- available in any custom length up to the maximum length
- hole types 03, 10 and 15







### 

- In a closed arrangement, two V guideways and two M guideways with identical dimensions A<sub>1</sub> and A<sub>2</sub> should always be used in order to ensure correct function. The guideways are matched accordingly and are marked with the same set number.
- Guideway pairs should be fitted in accordance with the set numbers (Figure 1). This ensures that the installation width is within tolerance.
- Guideways with a restricted tolerance of ± 0,005 for dimensions A<sub>1</sub> and A<sub>2</sub> (suffix US) can be used in any combination if normal requirements apply.
   Where accuracy requirements are higher, they can be matched in accordance with the marking on the guideway:

Page

marking US-1:	deviation $A_1$ and $A_2$	+0,005
		0
marking US-2:	deviation $A_1$ and $A_2$	0
		- 0,005.

doutotion A and A

### Further information

orling LIC 1.

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### Accuracy

### Quality, positional and profile tolerances

Guideways are available in three quality grades (Figure 2):

- Q2
  - For exacting requirements in high precision machines. This grade should only be used if the adjacent construction can be produced to equally high accuracy.
- Q6
- Conforms to the requirements of precision table guidance systems in machine tool construction.
- Q10
  - Standard quality grade. For requirements in general machine building
- Geometrical and positional tolerances (Figure 2).
- Profile tolerances (Figure 1).











Tolerances for hole pitches

The positional tolerance of the hole pattern  $\varnothing$  X is 1‰ of the guideway length L (Figure 3).

If the guideway length is less than 80% of the maximum length given in the dimension table, it can be fitted to a predrilled hole pattern; a positional tolerance of  $\varnothing$  0,2 must be achieved in this case.

If longer guideways are to fulfil this requirement, they must be ordered using the suffix P (positional tolerance).

## Sp.

### Special designs

### Guideways

- with different cross-sections, fixing holes, hole pitches and lengths
- guidance system heights  $A_1$  or  $A_2$  with restricted tolerance  $\pm 0,005$  mm
  - suffix US
- anti-corrosion protection by thin layer chromium plating
  suffix DSV
- raceway lead chamfers at ends of guideway
- suffix E2without holes on end faces
  - suffix E1
- needle roller flat cage with friction damping
  - series HGW.

## Amm

### Ordering example and ordering designation

For four machines:

M and V guideways in standard length for closed arrangement, stroke 90 mm, end pieces with wipers, angled flat cage made from metal. Profile size 5025 Hole type 15 Length of V guideways 500 mm Length of M guideways 400 mm Quality grade Q6

Insert nuts ESM for V guideway.

Angled flat cage HW and end pieces with wipers

Ordering designation:

8 off V 5025/15×500 Q6 8 off M 5025/15×400 Q6 8 off HW 16×355 16 off EAM 5025 48 off ESM M6 (Figure 4).



Figure 3 · Hole pattern of guideway



Figure 4 · Ordering example, ordering designation

HYDREL/EGIS guideways with angled needle roller flat cage or angled cylindrical roller flat cage

Series M V

M guideways also available with sealing strips – gap or lip seal – (only in conjunction with V guideways). Suffix for M guideways with gap seals: ...ZZ Suffix for M guideways with lip seals: ...PP





Cylindrical roller flat cage

172 600

Dimension	table · Dime	nsions ir	n mm											
Designation		Dimen	sions				Fixing holes	Holes in end faces						
	H -0,2	A _0,1	A <sub>3</sub>	h <sub>2</sub>	h <sub>3</sub>	K <sub>1</sub> For screws DIN 912-8.8	h <sub>1</sub>	a <sub>1</sub>	t min.	C <sub>4</sub> <sup>1</sup> )	C <sub>5</sub> <sup>2</sup> ) min.	C <sub>6</sub> <sup>2</sup> ) min.	К <sub>3</sub>	
M 3015	-	30	15	15	15,5	-	M4	5,5	10,5	15	40	15	15	M3
-	V 3015	30	15	15	-	10,5	M4	5,5	10,5	15	40	15	15	M3
M 4020	-	40	20	20	22,5	-	M6	7,5	13,2	20	80	15	15	M5
-	V 4020	40	20	20	-	13,5	M6	7,5	13,2	20	80	15	15	M5
M 4525	-	45	25	25	22,5	-	M6	7,5	18,2	15	80	20	20	M6
-	V 4525	45	25	25	-	14	M6	7,5	18,2	15	80	20	20	M6
M 5025	-	50	25	25	28	-	M6	10	18,2	15	80	20	20	M6
-	V 5025	50	25	25	-	17	M6	10	18,2	15	80	20	20	M6
M 6035	-	60	35	35	35	-	M8	11	26	20	100	20	20	M6
-	V 6035	60	35	35	-	20	M8	11	26	20	100	20	20	M6
M 6535	-	65	35	35	33	-	M8	11	26	20	100	20	20	M6
-	V 6535	65	35	35	-	20	M8	11	26	20	100	20	20	M6
M 7040	-	70	40	40	40	-	M10	13	29	25	100	20	20	M6
-	V 7040	70	40	40	-	24	M10	13	29	25	100	20	20	M6
M 8050	-	80	50	50	45	-	M12	14	37	30	100	20	20	M6
-	V 8050	80	50	50	-	26	M12	14	37	30	100	20	20	M6
M 8550	-	85	50	50	42	-	M12	14	37	30	100	20	20	M6
-	V 8550	85	50	50	-	26	M12	14	37	30	100	20	20	M6

^) M/V 3015: For L = 100 mm to 109 mm, C\_4 = 35 mm (3 holes). Other sizes: For L < C\_4 + C\_5  $_{min.}$  + C\_6  $_{min.}$  C\_4 = 50 mm.

<sup>2</sup>) C<sub>5</sub> and C<sub>6</sub> are dependent on the guideway length and must be identical at both ends of a guideway: C<sub>5</sub> and C<sub>6</sub> =  $\frac{L - \Sigma C_4}{2}$ ; C<sub>5 min.</sub> and C<sub>6 min.</sub> are minimum values.

<sup>3</sup>) Guideways longer than the stated maximum length are available as multi-piece units. The total length of a multi-piece unit must be stated in the order. Single-piece extra-long guideways may be available by agreement.

<sup>4</sup>) End pieces: see page 88; wipers: see page 90; insert nuts: see page 93.

<sup>5</sup>) Hole pattern for M/V 3015 =  $15 + 2 \times 35 + 15$ . Hole pattern for other sizes = 25 + 50 + 25.



Hole type 15

Hole type 10







Hole type 03

172 602

M and V

Hole types

Available standard lengths											Custom lengths	Appropriate	e angled flat	Accessories <sup>4</sup> )			
100 <sup>5</sup> )	150	200	300	400	500	600	700	800	900	1000	up to L <sup>3</sup> ) max.	Plastic needle roller cage	Cylindrical roller cage	Metal needle roller cage	End pieces	End pieces with wipers	Insert nuts
•	•	•	•	•	•	•					600	-	-	HW 10	EM 3015	EAM 3015	ESM M4
•	•	•	•	•	•	•					600	-	-	HW 10	EV 3015	EAV 3015	ESM M4
•	•	•	•	•	•	•					1000	FFW 2025	-	HW 15	EM 4020	EAM 4020	ESM M6
•	•	•	•	•	•	•					1000	FFW 2025	-	HW 15	EV 4020	EAV 4020	ESM M6
•		•	•	•	•	•	•	•	•	•	1000	-	HRW 50	-	EM 4525	EAM 4525	ESM M6
•		•	•	•	•	•	•	•	•	•	1000	-	HRW 50	-	EV 4525	EAV 4525	ESM M6
•		•	•	•	•	•	•	•	•	•	1300	FFW 2025	-	HW 16	EM 5025	EAM 5025	ESM M6
•		•	•	•	•	•	•	•	•	•	1300	FFW 2025	-	HW 16	EV 5025	EAV 5025	ESM M6
		•	•	•	•	•	•	•	•	•	1300	FFW 2535	-	HW 20	EM 6035	EAM 6035	ESM M8
		•	•	•	•	•	•	•	•	•	1300	FFW 2535	-	HW 20	EV 6035	EAV 6035	ESM M8
		•	•	•	•	•	•	•	•	•	1300	-	HRW 70	-	EM 6535	EAM 6535	ESM M8
		•	•	•	•	•	•	•	•	•	1300	-	HRW 70	-	EV 6535	EAV 6535	ESM M8
		•	•	•	•	•	•	•	•	•	1300	FFW 3045	-	HW 25	EM 7040	EAM 7040	ESM M10
		•	•	•	•	•	•	•	•	•	1300	FFW 3045	-	HW 25	EV 7040	EAV 7040	ESM M10
			•	•	•	•	•	•	•	•	1300	FFW 3555	-	HW 30	EM 8050	EAM 8050	ESM M12
			•	•	•	•	•	•	•	•	1300	FFW 3555	-	HW 30	EV 8050	EAV 8050	ESM M12
			•	•	•	•	•	•	•	•	1300	-	HRW 100	-	EM 8550	EAM 8550	ESM M12
			•	•	•	•	•	•	•	•	1300	-	HRW 100	-	EV 8550	EAV 8550	ESM M12

Series	a <sub>min</sub>
M and V 3015	0,7×45°
M and V 4020 to M and V 8550	1,3×45°



Chamfer dimensions



Design with sealing strips

HYDREL/EGIS ML and V guideways with adjusting gib and angled needle roller flat cage

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## Features

### Flat cage guidance systems

- consist of:
  - guideways of type ML with adjusting gib and guideways of type V
  - angled flat cages with a large number of rolling elements
- are used as linear locating bearings
- are preloaded using the adjusting gib
- the gib applies the preload evenly over the whole length of the guidance system
- are extremely rigid and support loads very uniformly due to the preload
- are highly suitable for limited stroke lengths
- have high rigidity and load carrying capacity whilst requiring very little space
- have low, uniform friction
- have high, consistent accuracy throughout their operating life
- can be supplemented by accessories
- are also available with sealing strips (*dimension table*).

### Guideways ML, V

- are made from through hardened steel with a minimum hardness of 670 HV (the adjusting gib is unhardened)
- have precision ground raceways and locating faces
- are produced with various hole types
- have threaded holes on the end faces for fitting end pieces or wipers.

### Angled needle roller flat cages

- are double row cages with the shanks at right angles to each other
- are made from metal or plastic
  - metal cages are high strength lightweight components. They are suitable for demanding conditions such as high acceleration, high temperatures and where the cage ends are exposed at some point in operation
  - plastic cages are an economical solution for less demanding operating conditions.

### Accessories

- end pieces prevent the cage creeping out of the load zone
- wipers protect the raceways against contamination under normal operating conditions.
- insert nuts allow the conversion of holes of type 15 to threaded holes.



- basic design comprising:
  - guideways ML with adjusting gib and guideway V.
    The pitch of the gib surface is 1,5%. When the gib is moved by 1 mm, the height changes by 15 µm
  - angled needle roller flat cages FFW, HW or HGW

### HYDREL/EGIS guideways



- with adjusting gib for preloading the system
- for angled needle roller flat cages
- custom lengths according to dimension table
- hole type 03 and 15





HYDREL/EGIS ML and V guideways with adjusting gib and angled needle roller flat cage



### Design and safety guidelines

- In a closed arrangement, two V guideways and one ML and one M guideway with identical dimensions A<sub>1</sub> and A<sub>2</sub> should always be used in order to ensure correct function. The guideways are matched accordingly and are marked with the same set number (suffix 2SX for ML and M guideway).
- Guideway pairs should be fitted in accordance with the set numbers (Figure 1). This ensures that the installation width is within tolerance.
- Guideways with a restricted tolerance of ± 0,005 for dimensions A1 and A2 (suffix US) can be used in any combination if normal requirements apply. Where accuracy requirements are higher, they can be matched in accordance with the marking on the guideway:

Marking US-1: deviation A1 and A2 +0,005 0

Marking US-2: deviation A<sub>1</sub> and A<sub>2</sub> 0

-0,005.

### Further information

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### Accuracy

Quality, positional and profile tolerances

Guideways are available in two guality grades (Figure 2): Q6

- Conforms to the requirements of precision table guidance systems in machine tool design.
- Q10
  - Standard quality grade for requirements in general machine building.
- Geometrical and positional tolerances (Figure 2).
- Profile tolerances (Figure 1).







Figure 2 · Positional accuracy and quality of guideways

Tolerances for hole pitches

The positional tolerance of the hole pattern  $\varnothing$  X is 1‰ of the guideway length L (Figure 3).

If the guideway length is less than 80% of the maximum length given in the dimension table, it can be fitted to a predrilled hole pattern; a positional tolerance of  $\emptyset$  0,2 must be achieved in this case.

If longer guideways are to fulfil this requirement, they must be ordered using the suffix P (positional tolerance).



### Special designs

### Guideways

- with different cross-sections, fixing holes, hole pitches and lengths
- guidance system heights A<sub>1</sub> or A<sub>2</sub> with restricted tolerance ±0,005 mm
  - suffix US
- anti-corrosion protection by thin layer chromium plating - suffix DSV
- raceway lead chamfers at ends of guideway - suffix E2
- without holes on end faces
  - suffix E1
- needle roller flat cage with friction damping
  - series HGW.

## AMAA

### Ordering example and ordering designation

For four machines:

ML and V guideways and M and V guideways in custom length for closed arrangement

Stroke length Profile width Hole type Length of ML and M guideways Length of V guideways Quality grade

150 mm 25 mm 15, counterbore on right 450 mm 600 mm

Q6.

Ordering designation:

4 off ML 6025/15 L×450 Q6 2SX 4 off M  $\,$  5025/15  $\,$   $\times450$  Q6 2SX  $\,$ 8 off V 5025/15 ×600 Q6 8 off HW 16×375 8 off EM 5025 8 off EML 25 (Figure 4).



Figure 3 · Hole pattern of guideway



Figure 4 · Ordering example, ordering designation



HYDREL/EGIS guideways with adjusting gib and angled needle roller flat cage Series MI

Series ML

ML guideways also available with sealing strips – gap or lip seal – (only in conjunction with V guideways).

Suffix for ML guideways with gap seals: ..ZZ Suffix for ML guideways with lip seals: ...PP



Dimension table · Dimensions in mm														
Designation	Dimens	ions						Fixing holes						
	H <sup>1</sup> )	A -0,1	A <sub>3</sub> -0,2	h <sub>2</sub> <sup>1</sup> )	h <sub>4</sub>	h <sub>5</sub>	а	K <sub>1</sub> For screws DIN 912-8.8	h <sub>1</sub>	a <sub>1</sub>	t min.	C <sub>4</sub> <sup>2</sup> )	C <sub>5</sub> <sup>3</sup> ) min.	C <sub>6</sub> min.
ML 5020	50	20	20	32,5	5,5	15	15	M6	17,5	13,2	20	80	30	15
ML 5520	55	20	20	37,5	6	20	15	M6	22,5	13,2	20	80	30	15
ML 5525	55	25	25	32,5	2,5	11,5	20	M6	15	18,2	15	80	30	20
ML 6025	60	25	25	37,5	3,5	16,5	20	M6	20	18,2	15	80	30	20
ML 6525	65	25	25	42,5	5	21,5	20	M6	25	18,2	15	80	30	20
ML 7025	70	25	25	47,5	6,5	26,5	20	M6	30	18,2	15	80	30	20
ML 7035	70	35	35	45	3	15,5	25	M8	21	26	20	100	32	20
ML 8035	80	35	35	55	5	25,5	25	M8	31	26	20	100	32	20
ML 8040	80	40	40	50	3	16	30	M10	23	29	25	100	32	20
ML 9040	90	40	40	60	5	26	30	M10	33	29	25	100	32	20
ML 9050	90	50	50	55	3	15,5	40	M12	24	37	30	100	32	20
ML 10050	100	50	50	65	5	25,5	40	M12	34	37	30	100	32	20

<sup>1</sup>) The various guideway heights H and  $h_2$  of a profile are determined by the gib heights dependent on the gib length. Setting range for H: –0,5 to +0,3 mm.

<sup>2</sup>) For length L < C<sub>4</sub> + C<sub>5 min.</sub> + C<sub>6 min.</sub> C<sub>4</sub> = 50 mm.

<sup>3</sup>) C<sub>5</sub> is always on the adjusting gib side.

<sup>4</sup>) The dimensions C<sub>5</sub> and C<sub>6</sub> are dependent on the guideway length:  $C_6 + C_5 = L - \sum C_4$ ; C<sub>6 min.</sub> and C<sub>5 min.</sub> are minimum values; for  $L - \sum C_4 \ge 2 C_5 \min C_6 = C_5$ .

<sup>5</sup>) End pieces: see page 88; wipers: see page 90; insert nuts: see page 93.



MI	and	V
	ana	v

Hole types

Holes in end faces	Design length	Appropriate guideway	Appropriate ang	led flat cages	Accessories <sup>5</sup> )			
К <sub>3</sub>	L <sup>4</sup> )		Plastic	Metal	End pieces	End pieces with wipers	Insert nuts	
M4	100- 300	V 4020	FFW 2025	HW 15	EML 20	EAML 20	ESM M6	
M4	301- 600	V 4020	FFW 2025	HW 15	EML 20	EAML 20	ESM M6	
M5	100- 250	V 5025	FFW 2025	HW 16	EML 25	EAML 25	ESM M6	
M5	251- 500	V 5025	FFW 2025	HW 16	EML 25	EAML 25	ESM M6	
M5	501- 750	V 5025	FFW 2025	HW 16	EML 25	EAML 25	ESM M6	
M5	751–1000	V 5025	FFW 2025	HW 16	EML 25	EAML 25	ESM M6	
M6	100- 500	V 6035	FFW 2535	HW 20	EML 35	EAML 35	ESM M8	
M6	501-1000	V 6035	FFW 2535	HW 20	EML 35	EAML 35	ESM M8	
M6	100- 500	V 7040	FFW 3045	HW 25	EML 40	EAML 40	ESM M10	
M6	501-1000	V 7040	FFW 3045	HW 25	EML 40	EAML 40	ESM M10	
M6	100- 500	V 8050	FFW 3555	HW 30	EML 50	EAML 50	ESM M12	
M6	501-1000	V 8050	FFW 3555	HW 30	EML 50	EAML 50	ESM M12	



Design with sealing strips



HYDREL/EGIS M and V guideways with integral toothed rack for positive control of angled needle roller flat cage

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### Features

### Flat cage guidance systems

### consist of:

- guideways of type M and V with an integral toothed rack
- angled flat cages with an integral gear and a large number of rolling elements
- are used where there is a risk of cage creep, for example: - in designs with non-uniform rigidity in the guidance area
  - if the cage does not regularly move to the end positions
- are used as linear locating bearings
- have high rigidity and load carrying capacity whilst requiring very little space
- have low, uniform friction
- have high, consistent accuracy throughout their operating life
- are only supplied in sets (designation MVZ)
  - two M and two V guideways, two needle roller flat cages and a positive control device comprising a rack and pinion
- can be supplemented by accessories
- are also available with sealing strips (*dimension table*).

### Guideways with positive cage control

- are made from through hardened tool steel with a minimum HYDREL/EGIS guideways hardness of 670 HV (the toothed rack is unhardened)
- have precision ground raceways and locating faces have the same mounting dimensions
- as guideways M/ML and V
- are produced with various hole types
- have threaded holes on the end faces for fitting wipers.

### Angled needle roller flat cages

- are subjected to positive control by means of an integral gear
- are double row cages with the shanks at right angles to each other
- are made from metal.

### Accessories

- wipers protect the raceways against contamination under normal operating conditions
- insert nuts allow the conversion of holes of type 15 to threaded holes.



- basic design comprising:
- guideway M with integral toothed rack and guideway V with integral toothed rack
- angled needle roller flat cage HW with integral gear



- M guideway and V guideway with integral toothed rack
- custom length according to dimension table
- hole types 03, 10 and 15





HYDREL/EGIS M and V guideways with integral toothed rack for positive control of angled needle roller flat cage



### Design and safety guidelines

The guideways and cages of MVZ guidance systems are supplied in sets. The parts are matched and marked on the basis of the dimensions A<sub>1</sub> and A<sub>2</sub> and the cages.

### Fitting

- Guideways must be fitted in sets. This must be carried out in accordance with the set number.
- During fitting, it must be ensured that the guideway and cage are correctly positioned relative to each other in order to prevent the cage colliding with the wipers or points on the adjacent construction.
  - Note the marking x-x indicating the correct relative position of the guideways and cages in the central stroke position (Figure 1).

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## Quality, positional and profile tolerances

Guideways are available in three quality grades (Figure 2):

Q2

For exacting requirements in high precision machines. This grade should only be used if the adjacent construction can be produced to equally high accuracy.

Q6

Conforms to the requirements of precision table guidance systems in machine tool design.

Q10

Standard quality grade for requirements in general machine building.

- Geometrical and positional tolerances (Figure 2).
- Profile tolerances (Figure 1).



Figure 1 · Profile tolerances, marking



Figure  $2 \cdot \text{Positional}$  accuracy and quality of guideways

Tolerances for hole pitches

The positional tolerance of the hole pattern  $\varnothing$  X is 1‰ of the guideway length L (Figure 3).

If the guideway length is less than 80% of the maximum length given in the dimension table, it can be fitted to a predrilled hole pattern; a positional tolerance of  $\varnothing$  0,2 must be achieved in this case.

If longer guideways are to fulfil this requirement, they must be ordered using the suffix P (positional tolerance).



### Special designs

### Guideways

- with different cross-sections, fixing holes, hole pitches and lengths
- anti-corrosion protection by thin layer chromium plating
  suffix DSV
- raceway lead chamfers at ends of guideway
  suffix E2
- without holes on end faces
  - suffix E1.



### Ordering example and ordering designation

4 sets of M and V guideways with integral positive cage control, closed arrangement

Profile size		4020					
M guideway	S	400 mm, hole type 03					
V guideways	5	600 mm, hole type 15					
Cage length		305 mm					
Stroke		190 mm					
End wipers of	on M guideway						
Quality grad	е	Q10.					
Ordering des	signation:						
4 sets MVZ	4020×400/600/	305 Q10					
each compr	ising:						
2 off M 4020/03×400 Q10 2 off V 4020/15×600 Q10							
2 off HW 15×305							
4 off EAM 4020 (Figure 4).							



Figure 3 · Hole pattern of guideway



Figure 4 · Ordering example, ordering designation



HYDREL/EGIS guideways with integral toothed rack for positive control of angled needle roller flat cage

Series MVZ

M guideways also available with sealing strips – gap or lip seal – (only in conjunction with V guideways).

Suffix for M guideways with gap seals: ...ZZ Suffix for M guideways with lip seals: ...PP



Dimension table - Dimensions in mm																		
Designation		Dimens	sions				Fixing holes							Holes in end faces				
		H -0,2	A -0,1	A <sub>3</sub> -0,2	h <sub>2</sub>	h <sub>3</sub>	K <sub>1</sub> For screws DIN 912-8.8	h <sub>1</sub>	a <sub>1</sub>	t	C <sub>4</sub> <sup>1</sup> )	C <sub>5</sub> ²) min.	C <sub>6</sub> ²) min.	K <sub>3</sub>				
M 3015	-	30	15	15	15,5	-	M4	5,5	10,5	15	40	15	15	M3				
-	V 3015	30	15	15	-	10,5	M4	5,5	10,5	15	40	15	15	M3				
M 4020	-	40	20	20	22,5	-	M6	7,5	13,2	20	80	15	15	M5				
-	V 4020	40	20	20	-	13,5	M6	7,5	13,2	20	80	15	15	M5				
M 5025	-	50	25	25	28	-	M6	10	18,2	15	80	20	20	M6				
-	V 5025	50	25	25	-	17	M6	10	18,2	15	80	20	20	M6				
M 6035	-	60	35	35	35	-	M8	11	26	20	100	20	20	M6				
-	V 6035	60	35	35	-	20	M8	11	26	20	100	20	20	M6				
M 7040	-	70	40	40	40	-	M10	13	29	35	100	20	20	M6				
-	V 7040	70	40	40	-	24	M10	13	29	35	100	20	20	M6				
M 8050	-	80	50	50	45	-	M12	14	37	30	100	20	20	M6				
-	V 8050	80	50	50	-	26	M12	14	37	30	100	20	20	M6				

 $^{\rm 1})\,$  M/V 3015: For L = 100 mm to 109 mm, C\_4 = 35 mm (3 holes). Other sizes: For L < C\_4 + C\_5 min. + C\_6 min. C\_4 = 50 mm.

<sup>2</sup>) C<sub>5</sub> and C<sub>6</sub> are dependent on the guideway length and must be identical at both ends of a guideway: C<sub>5</sub> and C<sub>6</sub> =  $\frac{L - \Sigma C_4}{2}$ ; C<sub>5 min.</sub> and C<sub>6 min.</sub> are minimum values.

<sup>3</sup>) Extra-long guideways available by agreement.

<sup>4</sup>) End pieces with wipers: see page 90; insert nuts: see page 93.

<sup>5</sup>) Hole pattern for M/V 3015 =  $15 + 2 \times 35 + 15$ . Hole pattern for other sizes = 25 + 50 + 25.



Hole type 15

Hole type 10

ĺΖ







Hole type 03

172 602

MVZ

Hole types

Availab	le stand	dard ler	ngths								Custom lengths up to	Appropriate angled flat	Accessories <sup>4</sup> )	
100 <sup>5</sup> )	150	200	300	400	500	600	700	800	900	1000	L <sup>3</sup> ) cages		End pieces with wipers	Insert nuts
•	•	•	•	•	•	•					600	HW 10	EAM 3015	ESM M4
•	•	•	•	•	•	•					600	HW 10	EAV 3015	ESM M4
•	•	•	•	•	•	•					1000	HW 15	EAM 4020	ESM M6
•	•	•	•	•	•	•					1000	HW 15	EAV 4020	ESM M6
•		•	•	•	•	•	•	•	•	•	1300	HW 16	EAM 5025	ESM M6
•		•	•	•	•	•	•	•	•	•	1300	HW 16	EAV 5025	ESM M6
		•	•	•	•	•	•	•	•	•	1300	HW 20	EAM 6035	ESM M8
		•	•	•	•	•	•	•	•	•	1300	HW 20	EAV 6035	ESM M8
		•	•	•	•	•	•	•	•	•	1300	HW 25	EAM 7040	ESM M10
		•	•	•	•	•	•	•	•	•	1300	HW 25	EAV 7040	ESM M10
			•	•	•	•	•	•	•	•	1300	HW 30	EAM 8050	ESM M12
			•	•	•	•	•	•	•	•	1300	HW 30	EAV 8050	ESM M12

Series	a <sub>min</sub>
M and V 3015	0,7×45°
M and V 4020 to M and V 8050	1,3×45°



Chamfer dimensions



Design with sealing strips



## Flat guidance systems

# HYDREL/EGIS M guideways with sliding layer and V guideways

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### Features

### Flat guidance systems

- consist of
  - guideways of type M with a plain sliding layer and guideways of type V as a mating raceway
- are used especially
  - for supporting static or pulsating loads
  - where lubrication is not possible or only initial lubrication can be provided (plain sliding material LB1, *Design and safety guidelines*, page 70).
     However, they can also be used with oil or grease lubrication
  - where an increased friction level is required in order to provide damping
- are used as linear locating bearings
- have hardly any stick-slip.

### Guideways M with plain sliding layer

- comprise a guideway made from unhardened steel with a plain sliding layer attached by adhesive
- are available with various plain sliding materials
  - Turcite® B, suffix LB1
  - Permaglide® P21, suffix LP21
- have the same mounting dimensions as guideways M and V with flat cages
- are produced with various hole types
- have threaded holes on the end faces for fitting wipers.

### Accessories

insert nuts allow the conversion of holes of type 15 to threaded holes.



- guideway M with plain sliding layer and guideway V

### HYDREL/EGIS guideway



- guideway without plain sliding layer, dimensionally identical to design for flat cages
- standard and custom lengths according to dimension table hole types 03, 10 and 15



### Flat guidance systems

HYDREL/EGIS M guideways with sliding layer and V guideways



### Design and safety guidelines

Guide values for selection of plain sliding material

	Turcite <sup>®</sup> B	Permaglide <sup>®</sup> P21
Suffix	LB1	LP21
Maximum specific load		
p <sub>max</sub> static [N/mm <sup>2</sup> ]	6	250
$p \cdot v max$ [N/mm <sup>2</sup> · m/s]	1	3
Maximum temperature [°C]	-40 to +80	-40 to +110
Coefficient of friction, dry	0,15 to 0,26	not suitable
Coefficient of friction, lubricated	0,04 to 0,08	0,02 to 0,2

- In a closed arrangement, two V guideways and two M guideways with identical dimensions A<sub>1</sub> and A<sub>2</sub> should always be used in order to ensure correct function. The guideways are matched accordingly and are marked with the same set number.
- Guideway pairs should be fitted in accordance with the set numbers (Figure 1). This ensures that the installation width is within tolerance.



Plain guidance systems must not be set free from clearance or preloaded in a rigid environment, since even slight thermal expansion can lead to uncontrolled friction conditions.

The screw connections should be checked.

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### Quality, positional and profile tolerances

Guideways are available in two quality grades (Figure 2):

- Conforms to the requirements of precision table guidance systems in machine tool design.
- Q10 Standard quality grade for requirements in general machine building.
- Geometrical and positional tolerances (Figure 2).
- Profile tolerances (Figure 1).



Figure 1 · Profile tolerances, marking



Figure 2  $\cdot\,$  Positional accuracy and quality of guideways

Tolerances for hole pitches

The positional tolerance of the hole pattern  $\varnothing$  X is 1‰ of the guideway length L (Figure 3).

If the guideway length is less than 80% of the maximum length given in the dimension table, it can be fitted to a predrilled hole pattern; a positional tolerance of  $\varnothing$  0,2 must be achieved in this case.

If longer guideways are to fulfil this requirement, they must be ordered using the suffix P (positional tolerance).



### Special designs

### Guideways

- with different cross-sections, fixing holes, hole pitches and lengths
- without holes on end faces
  - suffix E1.



Figure  $3 \cdot$  Hole pattern of guideway





### Ordering example and ordering designation

For four machines:

M guideways with plain sliding layer in standard length and V guideways in standard length, for closed arrangement, stroke 100 mm

Profile size	5025
Hole type	15
Length of V guideways	500 mm
Length of M guideways with plain sliding layer	400 mm
Quality grade	Q6
Plain sliding layer	LB1.
Ordering designation:	

8 off M 5025/15×400 Q6 LB1 8 off V 5025/15×500 Q6 (Figure 4). Intended for four sets.



Figure 4 · Ordering example, ordering designation
### Flat guidance systems

HYDREL/EGIS guideways with plain sliding layer Series M

V



M with plain sliding layer and V

Dimension	Dimension table · Dimensions in mm												
Designation		Dimensio	ons				Fixing holes						
		H -0,2	A -0,1	A <sub>3</sub> -0,2	h <sub>2</sub>	h <sub>3</sub>	K <sub>1</sub> For screws DIN 912-8.8	h <sub>1</sub>	a <sub>1</sub>	t min.	C <sub>4</sub> <sup>1</sup> )	C <sub>5</sub> <sup>2</sup> ) min.	C <sub>6</sub> <sup>2</sup> ) min.
M 3015	-	30	15	15	15,5	-	M4	5,5	10,5	15	40	15	15
- '	V 3015	30	15	15	-	10,5	M4	5,5	10,5	15	40	15	15
M 4020	-	40	20	20	22,5	-	M6	7,5	13,2	20	80	15	15
- '	V 4020	40	20	20	-	13,5	M6	7,5	13,2	20	80	15	15
M 5025	-	50	25	25	28	-	M6	10	18,2	15	80	20	20
- '	V 5025	50	25	25	-	17	M6	10	18,2	15	80	20	20
M 6035	-	60	35	35	35	-	M8	11	26	20	100	20	20
- '	V 6035	60	35	35	-	20	M8	11	26	20	100	20	20
M 7040	-	70	40	40	40	-	M10	13	29	25	100	20	20
- '	V 7040	70	40	40	-	24	M10	13	29	25	100	20	20
M 8050	-	80	50	50	45	-	M12	14	37	30	100	20	20
- ''	V 8050	80	50	50	-	26	M12	14	37	30	100	20	20

 $^{\rm 1})\,$  M/V 3015: For L = 100 mm to 109 mm, C\_4 = 35 mm (3 holes). Other sizes: For L < C\_4 + C\_5 min. + C\_6 min. C\_4 = 50 mm.

<sup>2</sup>) C<sub>5</sub> and C<sub>6</sub> are dependent on the guideway length and must be identical at both ends of a guideway: C<sub>5</sub> and C<sub>6</sub> =  $\frac{L - \Sigma C_4}{2}$ ; C<sub>5 min</sub> and C<sub>6 min</sub> are minimum values.

<sup>3</sup>) Guideways longer than the stated maximum length are available as multi-piece units. The total length of a multi-piece unit must be stated in the order.

<sup>4</sup>) End pieces with wipers: see page 90; insert nuts: see page 93.

<sup>5</sup>) Hole pattern for M/V  $3015 = 15 + 2 \times 35 + 15$ .

Hole pattern for other sizes = 25 + 50 + 25.

#### Maximum static load <sup>6</sup>)

Designation	F <sub>0 perm</sub>	F <sub>0 perm</sub>
	Turcite <sup>®</sup> B	Permaglide <sup>®</sup> P21
M 3015	3 600	150 000
V 3015	-	-
M 4020	6 600	275 000
V 4020	-	-
M 5025	8 400	350 000
V 5025	-	-
M 6035	12000	500 000
V 6035	-	-
M 7040	13 800	575 000
V 7040	-	-
M 8050	16 200	675 000
V 8050	-	-

<sup>6</sup>) For a guideway length of 100 mm in *Load direction* according to the figure.









M with plain sliding layer and V

Hole types

Holes in Standard lengths end faces						Design lengths Accessories <sup>4</sup> ) up to								
K <sub>3</sub>	1005)	150	200	300	400	500	600	700	800	900	1000	L <sup>3</sup> )	End pieces with wipers	Insert nuts
140	100 )	150	200	300	400	300	000	700	000	900	1000	111dA.	E 4 4 4 4 4 4	5014144
M3	•	•	•	•	•	•	•					600	EAM 3015	ESIM M4
M3	•	•	•	•	•	•	•					600	-	ESM M4
M5	•	•	•	•	•	•	•					1000	EAM 4020	ESM M6
M5	•	•	•	•	•	•	•					1000	-	ESM M6
M6	•		•	•	•	•	•	•	•	•	•	1 300	EAM 5025	ESM M6
M6	•		•	•	•	•	•	•	•	•	•	1300	-	ESM M6
M6			•	•	•	•	•	•	•	•	•	1 300	EAM 6035	ESM M8
M6			•	•	•	•	•	•	•	•	•	1300	-	ESM M8
M6			•	•	•	•	•	•	•	•	•	1 300	EAM 7040	ESM M10
M6			•	•	•	•	•	•	•	•	•	1300	-	ESM M10
M6				•	•	•	•	•	•	•	•	1 300	EAM 8050	ESM M12
M6				•	•	•	•	•	•	•	•	1 300	-	ESM M12

Series	a <sub>min</sub>
M and V 3015	0,7×45°
M and V 4020 to M and V 8050	1,3×45°



Chamfer dimensions



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HYDREL/EGIS J and S guideways with needle roller flat cage

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	Features

### **Features**

#### Flat cage guidance systems

- consist of:
  - quideways of type J and S
  - flat cages with a large number of rolling elements
- are used as linear non-locating bearings
- are highly suitable for limited stroke lengths
- have high rigidity and load carrying capacity whilst requiring very little space
- have low, uniform friction
- have high, consistent accuracy throughout their operating life
- can be supplemented by accessories.

#### Guideways J, S

- are made from through hardened steel with a minimum hardness of 670 HV
- have precision ground raceways and locating faces
- the J guideway guides the flat cage between ribs
- are produced with various hole types
- have threaded holes on the end faces for fitting end pieces or wipers.

#### Needle roller flat cages

- are single or double row cages depending on the housing
- are made from metal or plastic
  - metal cages are lightweight components with high strength. They are suitable for more demanding conditions such as high acceleration, high temperatures and where the ends of the cage are exposed at some point in operation.

#### Accessories

- end pieces prevent the cage creeping out of the load zone
- wipers protect the the raceways against contamination under normal operating conditions.
- insert nuts allow the conversion of holes of type 15 to threaded holes.

# Flat cage guidance systems 3 173 595

- basic design comprising:
- guideways J, S
- needle roller flat cages FF or H
- high or flat design

#### HYDREL/EGIS guideways



- for needle roller flat cages
- flat and high designs
- available in any custom length up to the maximum length given in the dimension table
- hole type 03 and 15



HYDREL/EGIS J and S guideways with needle roller flat cage



#### Design and safety guidelines

- When using J and S guideways in conjunction with M and V guideways in an open arrangement, the heights must be carefully matched in order to prevent edge loads (page 45, Figure 14).
- Matched sets of combinations of M/V and J/S guideways in an open arrangement (suffix 4SX) allows installation without the need for a shim or an ML guideway. It must be ensured that the nominal value for the height difference of the adjacent parts conforms to the tolerance  $\Delta h$  (page 45).

#### Fitting

6

Parts supplied in sets are matched to each other and must be fitted in sets in accordance with the set number. This must be carried out in accordance with the set number.

#### Further information

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#### Quality and positional tolerances

Guideways are available in three quality grades (Figure 2):

Q2

For exacting requirements in high precision machines. This grade should only be used if the adjacent construction can be produced to equally high accuracy.

Q6

Conforms to the requirements of precision table guidance systems in machine tool design.

Q10

Standard quality grade for requirements in general machine building.

- Geometrical and positional tolerances (Figure 2).
- Profile tolerances (Figure 1).







Figure  $2\cdot$  Positional accuracy and quality of guideways

Tolerances for hole pitches

The positional tolerance of the hole pattern  $\varnothing$  X is 1‰ of the guideway length L (Figure 3).

If the guideway length is less than 80% of the maximum length given in the dimension table, it can be fitted to a predrilled hole pattern; a positional tolerance of  $\varnothing$  0,2 must be achieved in this case.

If longer guideways are to fulfil this requirement, they must be ordered using the suffix P (positional tolerance).

### Sp.

### Special designs

#### Guideways

- with different cross-sections, fixing holes, hole pitches and lengths
- anti-corrosion protection by thin layer chromium plating
  suffix DSV
- raceway lead chamfers at ends of guideway
  suffix E2
- without holes on end faces
  - suffix E1
- matching in sets for combinations of M/V and J/S guideways
  suffix 4SX
- needle roller flat cage with friction damping
  - series HGW, HG.



### ${\Bbb X}$ Ordering example and ordering designation

For five machines:

J/S guideways and M/V guideways in custom length, for open arrangement, stroke length 200 mm, matched in sets (height difference = 25 mm)

J and S profile size4025M and V profile size5025Hole type15Length of guideways750 mmQuality gradeQ10.

Ordering designation:

4025/15×750 Q10 4SX 5 off J 4025/15×750 Q10 4SX 5 off S 5 off V 5025/15×750 Q10 4SX 5025/15×750 Q10 4SX 5 off M 5 off H 15×650 5 off HW 16×650 10 off EJ 40 10 off EM 5025 (Figure 4).



Figure 3 · Hole pattern of guideway



Figure 4 · Ordering example, ordering designation



HYDREL/EGIS guideways with needle roller flat cage Flat design Series J S



J and S

Dimension tab	Dimension table · Dimensions in mm										
Designation		Dimensior	าร			Fixing holes					
		H 0,2	A -0,2	h <sub>2</sub>	h <sub>3</sub>	K <sub>1</sub> For screws DIN 912-8.8	а	h <sub>4</sub>	C <sub>4</sub> <sup>1</sup> )	C <sub>5</sub> min.	C <sub>6</sub> min.
J 3525	-	25	35	11,8	-	M5	22	6	80	15	15
-	S 3525	25	35	-	13	M5	22	-	80	15	15
J 4025	-	25	40	12,3	-	M5	28	6,5	80	15	15
-	S 4025	25	40	-	12,5	M5	28	-	80	15	15
J 5030	-	30	50	14,8	-	M6	35	8	100	15	15
-	S 5030	30	50	-	15	M6	35	-	100	15	15
J 5530	-	30	55	15,3	-	M6	40	8,5	100	15	15
-	S 5530	30	55	-	14,5	M6	40	-	100	15	15

1) For length L < C\_4 + 2  $\cdot$  C\_5 min. C\_4 = 50 mm.

<sup>2</sup>) Guideways longer than the stated maximum length are available as multi-piece units. The total length of a multi-piece unit must be stated in the order.

<sup>3</sup>) Associated flat cages: see page 94.

<sup>4</sup>) End pieces: see page 88; wipers: see page 90.



Holes in end faces	Custom length	Appropriate flat cag	ges <sup>3</sup> )	Accessories <sup>4</sup> )				
К <sub>3</sub>	L <sup>2</sup> )			End pieces	End pieces with wipers	Insert nuts		
M5	80–1000	FF 2010	H 10	EJ 35	EAJ 35	ESM M5 D10		
-	80–1000	FF 2010	H 10	-	-	ESM M5 D10		
M5	80–1000	FF 2515	H 15	EJ 40	EAJ 40	ESM M5 D10		
-	80–1000	FF 2515	H 15	-	-	ESM M5 D10		
 M6	80–1000	FF 3020	H 20	EJ 50	EAJ 50	ESM M6		
-	80–1000	FF 3020	H 20	-	-	ESM M6		
 M6	80–1000	FF 3525	H 25	EJ 55	EAJ 55	ESM M6		
-	80–1000	FF 3525	H 25	-	-	ESM M6		



Chamfer dimensions



HYDREL/EGIS guideways with needle roller flat cage High design Series J



Dimension ta	Dimension table · Dimensions in mm											
Designation		Dimensions				Fixing holes						
		Н	A	h <sub>2</sub>	h <sub>3</sub>	K <sub>1</sub> For screws DIN 912-8.8	h <sub>1</sub>	a <sub>1</sub>	t min.	C <sub>4</sub> <sup>1</sup> )	C <sub>5</sub> <sup>2</sup> ) min.	$C_6^2$ ) min.
J 5025	-	50	25	27,7	-	M6	10	18,2	15	80	20	20
-	S 5025	50	25	-	22	M6	10	18,2	15	80	20	20
J 6035	-	60	35	34,7	1	M8	11	26	20	100	20	20
-	S 6035	60	35	-	25	M8	11	26	20	100	20	20
J 7040	-	70	40	39,7	I	M10	13	29	25	100	20	20
-	S 7040	70	40	-	30	M10	13	29	25	100	20	20
J 8050	-	80	50	44,7	-	M12	14	37	30	100	20	20
-	S 8050	80	50	-	35	M12	14	37	30	100	20	20

<sup>1</sup>) For length L < C<sub>4</sub> + C<sub>5 min.</sub> + C<sub>6 min.</sub> C<sub>4</sub> = 50 mm.

<sup>2</sup>) C<sub>5</sub> and C<sub>6</sub> are dependent on the guideway length and must be identical at both ends of a guideway: C<sub>5</sub> and C<sub>6</sub> =  $\frac{L - \Sigma C_4}{2}$ ; C<sub>5 min</sub>. and C<sub>6 min</sub>. are minimum values.

<sup>3</sup>) Guideways longer than the stated maximum length are available as multi-piece units. The total length of a multi-piece unit must be stated in the order.

4) Associated flat cages: see page 94.

<sup>5</sup>) End pieces: see page 88; wipers: see page 90.



Holes in end faces	Custom lengths up to	Appropriate flat cages <sup>4</sup> )	Accessories <sup>5</sup> )					
K <sub>3</sub>	L <sup>3</sup> ) max.		End pieces	End pieces with Wipers	Insert nuts			
M6	800	H 15, FF 2515	EJ 5025	EAJ 5025	ESM M6			
M6	800	H 15, FF 2515	-	-	ESM M6			
M6	1000	H 24 ZW	EJ 6035	EAJ 6035	ESM M8			
M6	1000	H 24 ZW	-	-	ESM M8			
M6	1200	H 34 ZW	EJ 7040	EAJ 7040	ESM M10			
M6	1200	H 34 ZW	-	-	ESM M10			
M6	1200	H 44 ZW	EJ 8050	EAJ 8050	ESM M12			
M6	1200	H 44 ZW	-	-	ESM M12			



Chamfer dimensions



HYDREL/EGIS L counterstay system with needle roller and cylindrical roller flat cages Locating/non-locating bearing units

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### Features

#### HYDREL/EGIS L counterstay systems

- are complete units comprising:
  - a locating bearing arrangement comprising guideways of type M and V
  - a non-locating bearing arrangement comprising guideways of type J and S
  - needle roller and cylindrical roller flat cages
  - an L counterstay in each case which preloads the two guidance systems
- are used as linear locating/non-locating bearing units
  - due to the subdivision into locating and non-locating bearing arrangements, the system is not distorted by thermal expansion
- are used where guidance systems must have high accuracy, rigidity and load carrying capacity and the adjacent construction must not be distorted by preload forces or thermal expansion
  - the highest load carrying capacity and rigidity are achieved when the principal load acts in load direction I or II (*dimension table*, page 86)
- have the highest accuracy of any rolling element type linear guidance system
- are easy to fit. No adjustment work is required after fitting
  - the ready-to-fit guidance system is fitted using the specified tightening torques. This gives a precise preload
- have particularly low friction
- can only be replaced in their entirety
  - all parts of the system are matched to each other
- can be supplemented by accessories.

#### Guideways

correspond to the standard range of M, V, J and S guideways in grade Q2.

#### Needle roller and cylindrical roller flat cages

guide the rolling elements exactly parallel to the axis and retain them at a specified distance from each other.

#### LU counterstay arrangements

comprise an unhardened spacer and a hardened counterstay

#### Accessories

- $\hfill$  end pieces prevent the cage creeping out of the load zone
  - end pieces should be selected in accordance with the guideway arrangement.

#### HYDREL/EGIS L counterstay system



#### HYDREL/EGIS guideways



- guideways M, V, J, S
- hole type 15 for fixing



HYDREL/EGIS L counterstay system with needle roller and cylindrical roller flat cages

Locating/non-locating bearing units



#### Design and safety guidelines

LUE counterstay systems are supplied in sets (Figure 1). The parts are matched to each other in terms of dimensions and preload and are marked with set numbers.

#### Fitting

- The parts must be fitted as sets (Figure 1). This must be carried out in accordance with the set number.
- The fixing screws for the counterstays must be tightened to the specified tightening torque.



#### Quality and positional tolerances

Guideways are supplied in quality grade Q2 (Figure 2):

- For exacting requirements in high precision machines.
- Geometrical and positional tolerances (Figure 2).

#### Tolerances for hole pitches

The positional tolerance of the hole pattern  $\varnothing$  X is 1‰ of the guideway length L (Figure 3).

If the guideway length is less than 80% of the maximum length given in the dimension table, it can be fitted to a predrilled hole pattern; the positional tolerance of  $\emptyset$  0,2 applies in this case.

#### Further information

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	Lubrication	
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### Special designs

Available by agreement:

Needle roller and cylindrical roller flat cages

- flat cages with friction damping
- series HG, HGW
- anti-corrosion protection by thin layer chromium plating - suffix DSV.



Figure 1 · Marking of sets for LUE unit



Figure 2 · Positional accuracy and quality of guideways



Figure 3 · Hole pattern of guideway

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#### Ordering example and ordering designation

For four machines:LUE units to custom lengthProfile size6035Counterstay with hole type10Length of V and S guideways800 mmLength of M and J guideways620 mmStroke180 mmEnd pieces.180 mm

Ordering designation: 8 sets LUE 6035×8001) each comprising: LU 6035/10×800 2 off Counterstay 6035/15×620 Q2 1 off Guideway Μ 1 off Guideway V 6035/15×800 Q2 1 off Guideway J 6035/15×620 Q2 1 off Guideway S 6035/15×800 Q2 1 off Angled flat cage HW 20 G1×530 1 off Flat cage 24 ZW G1×530 Н 2 off Flat cages ΒF 5015 G1×530 4 off End pieces ELU 6035 2 off End pieces ΕM 6035 EJ 6035 (Figure 4). 2 off End pieces

<sup>1</sup>) Length of longest part.



Figure 4 · Ordering example, ordering designation



HYDREL/EGIS L counterstay system with needle roller and cylindrical roller flat cages Series LUE



L counterstay system

Dimension table · Dimensions in mm														
Designation	Dimensions			Mountin	Mounting dimensions									
Unit	H 0 -0,2	H <sub>1</sub>	A <sub>1</sub>	A <sub>2</sub>	A <sub>3</sub>	A <sub>4</sub>	K <sub>1</sub> <sup>1</sup> )	K <sub>2</sub> <sup>1</sup> )	C <sub>4</sub>	C <sub>5</sub> min.	C <sub>6</sub> min.	L <sup>2</sup> ) max.	H <sub>2</sub>	H <sub>3</sub>
LUE 5025	50	62	45	30	20	31	M 6	Μ6	50	20	20	800	27,5	17
LUE 6035	60	77	60	40	25	42	M 8	M 8	50	20	20	1000	34,5	20
LUE 7040	70	89	65	40	25	47	M10	M 8	50	20	20	1000	39,5	24
LUE 8050	80	100	86	51	36	61	M12	M12	50	20	20	1000	44,5	26

<sup>1</sup>) Screws of grade 10.9 should be used.

<sup>2</sup>) L >L<sub>max</sub> by agreement.

<sup>3</sup>) For a theoretical cage length of 100 mm in the load direction according to the figure. Calculation of basic load ratings for effective cage lengths and calculation of safety factors and life: see page 18ff.

<sup>4</sup>) For a theoretical cage length of 100 mm in the load direction according to the figure. Calculation of limit loads for effective cage lengths:

 $F_{w\,I,\,II,\,III} = F_{I,\,II,\,III} \cdot \frac{L_K - 2e + t}{100} \text{ where } Z = \frac{L_K - 2e}{100} + 1 = \text{whole number}.$ 



Main load directions

<sup>5</sup>) Limited by system preload.

<sup>6</sup>) Limited by load carrying capacity/friction locking effect of fixing screws.

Tightening torque for fixing screws <sup>7</sup> )						
for screws K <sub>1</sub> , K <sub>2</sub> Grade 10.9	Tightening torque Nm					
M 6	12					
M 8	29					
M10	58					
M12	101					

<sup>7</sup>) Coefficient of friction  $\mu = 0, 1$ .



Load carrying capacity									
Basic dynamic load ratings <sup>3</sup> )			Limiting loads <sup>4</sup> )						
CI	CII	CIII	F <sub>I perm</sub> <sup>5</sup> )	F <sub>II perm</sub> <sup>5</sup> )	F <sub>III perm</sub> <sup>6</sup> )	F <sub>IV perm</sub> <sup>6</sup> )			
Ν	Ν	Ν	Ν	Ν	Ν	Ν			
26200	35800	21600	13840	15630	1 200	7500			
40300	37000	69900	38690	58 620	1 500	10 000			
62900	57000	69900	42500	61720	2500	16000			
82700	88900	69900	43150	69540	4000	23 000			



LUE unit

Components of LUE unit								
Unit desig- nation	Counterstay	Guideway	ys	Cages				
	1	M/V ② ③	J/S ④⑤	6	1	8		
LUE	LU 5025/10	5025/15	5025/15	HW 15	H 15	H 10		
5025	LU 5025/03	5025/15	5025/15	HW 15	H 15	H 10		
LUE	LU 6035/10	6035/15	6035/15	HW 20	H 24 ZW	BF 5015		
6035	LU 6035/03	6035/15	6035/15	HW 20	H 24 ZW	BF 5015		
LUE	LU 7040/10	7040/15	7040/15	HW 25	H 34 ZW	BF 5015		
7040	LU 7040/03	7040/15	7040/15	HW 25	H 34 ZW	BF 5015		
LUE	LU 8050/10	8050/15	8050/15	HW 30	H 44 ZW	BF 5015		
8050	LU 8050/03	8050/15	8050/15	HW 30	H 44 ZW	BF 5015		



End pieces for HYDREL/EGIS guideways

Series EV EM EML

EJ



Dimension table · Dime	nsions in mm				
Guideway	Designation <sup>1</sup> )				
					_
	EV	EM	EML	EJ	
V 3015	EV 3015	-	-	-	
M 3015	-	EM 3015	-	-	
V 4020	EV 4020	-	-	-	
M 4020	-	EM 4020	-	-	
ML 5020, ML 5520	-	-	EML 20	-	
V 4525	EV 4525	-	-	-	
M 4525	-	EM 4525	-	-	
V 5025	EV 5025	-	-	-	
M 5025	-	EM 5025	-	-	
ML 5525 to ML 7025	-	-	EML 25	-	
V 6035	EV 6035	-	-	-	
M 6035	-	EM 6035	-	-	
ML 7035, ML 8035	-	-	EML 35	-	
V 6535	EV 6535	-	-	-	
M 6535	-	EM 6535	-	-	
V 7040	EV 7040	-	-	-	
M 7040	-	EM 7040	-	-	
ML 8040, ML 9040	-	-	EML 40	-	
V 8050	EV 8050	-	-	-	
M 8050	-	EM 8050	-	-	
ML 9050, ML 10050	-	-	EML 50	-	
V 8550	EV 8550	-	-	-	
M 8550	-	EM 8550	-	-	
J 3525	-	-	-	EJ 35	
J 4025	-	-	-	EJ 40	
J 5030	-	-	-	EJ 50	
J 5530	-	-	-	EJ 55	
J 5025	-	-	-	EJ 5025	
J 6035	-	-	-	EJ 6035	
J 7040	-	-	-	EJ 7040	
J 8050	-	-	-	EJ 8050	

<sup>1</sup>) The end pieces are supplied with fixing screws to ISO 7380.

<sup>2</sup>) Profile size 3015 = 2 mm.



Dimensions					Fixing screws
A <sub>1</sub>	H <sub>1</sub>			C <sub>8</sub>	K <sub>1</sub>
				max.	
14	12,6	-	-	5	-
14	-	16,7	-	5	-
19	15,6	-	-	6	-
19	-	23	-	6	-
19	-	12	-	6,5	-
24	18,5	-	-	6,5	-
24	-	26,5	-	6,5	-
24	19,1	-	-	6,5	-
24	-	30	-	6,5	-
24	-	15	-	6,5	-
34	22,8	-	-	6,5	-
34	-	35	-	6,5	-
34	-	23	-	6,5	-
34	27,5	-	-	6,5	-
34	-	40,5	-	6,5	-
39	27,5	-	-	6,5	-
39	-	40	-	6,5	-
39	-	28,5	-	6,5	-
49	30,3	-	-	6,5	-
49	-	45	-	6,5	-
49	-	35	-	6,5	-
49	37,5	-	-	6,5	-
49	-	54,5	-	6,5	-
34	-	-	11	6	-
39	-	-	12	6	-
49	-	-	14	6,5	-
54	-	-	15	6,5	-
24	-	-	16,6	6,5	-
34	-	-	17	6,5	-
39	-	-	16,8	6,5	-
49	-	-	18,2	6,5	-



Wipers for HYDREL/EGIS guideways

Series EAV EAM EAML EAJ



Dimension table · Dimensions	s in mm				
Guideway	Designation <sup>1</sup> )				
-					
	EAV	EAM	EAML	EAJ	
V 3015	EAV 3015	-	-	-	
M 3015	-	EAM 3015	-	-	
V 4020	EAV 4020	-	-	-	
M 4020	-	EAM 4020	-	-	
ML 5020, ML 5520	-	-	EAML 20	-	
V 4525	EAV 4525	-	-	-	
M 4525	-	EAM 4525	-	-	
V 5025	EAV 5025	-	-	-	
M 5025	-	EAM 5025	-	-	
ML 5525 to ML 7025	-	-	EAML 25	-	
V 6035	EAV 6035	-	-	-	
M 6035	-	EAM 6035	-	-	
ML 7035, ML 8035	-	-	EAML 35	-	
V 6535	EAV 6535	-	-	-	
M 6535	-	EAM 6535	-	-	
V 7040	EAV 7040	-	-	-	
M 7040	-	EAM 7040	-	-	
ML 8040, ML 9040	-	-	EAML 40	-	
V 8050	EAV 8050	-	-	_	
M 8050	-	EAM 8050	-	-	
ML 9050, ML 10050	-	-	EAML 50	-	
V 8550	EAV 8550	-	-	-	
M 8550	-	EAM 8550	-	-	
J 3525	-	-	-	EAJ 35	
J 4025	-	-	-	EAJ 40	
J 5030	-	-	-	EAJ 50	
J 5530	-	-	-	EAJ 55	
J 5025	-	-	-	EAJ 5025	
J 6035	-	-	-	EAJ 6035	
J 7040	-	-	-	EAJ 7040	
J 8050	-	-	-	EAJ 8050	

 $^{\rm 1}\!\!$  ) The end pieces are supplied with fixing screws to ISO 7380.

<sup>2</sup>) Profile size 3015 = 4 mm.



Dimensions	nensions							
A <sub>1</sub>	H <sub>1</sub>	H <sub>1</sub>						
				max.				
14	13,4	-	-	6				
14	-	18,6	-	6				
19	16,7	-	-	8				
19	-	25,7	-	8				
19	-	14	-	8				
24	21,2	-	-	8,5				
24	-	30,2	-	8,5				
24	19,9	-	-	8,5				
24	-	32,2	-	8,5				
24	-	18,4	-	8,5				
34	23,7	-	-	8,5				
34	-	39,2	-	8,5				
34	-	25,7	-	8,5				
34	30,2	-	-	8,5				
34	-	44,2	-	8,5				
39	28,4	-	-	8,5				
39	-	45,2	-	8,5				
39	-	31,2	-	8,5				
49	31,2	-	-	8,5				
49	-	53,2	-	8,5				
49	-	39,6	-	8,5				
49	40,2	-	-	8,5				
49	-	58,2	-	8,5				
34	-	-	11,6	8,5				
39	-	-	12,3	8,5				
49	-	-	14,3	8,5				
54	-	-	14,8	8,5				
24	-	-	16,9	8,5				
34	-	-	17,3	8,5				
39	-	-	17,1	8,5				
49	-	-	18,7	8,5				



End pieces for HYDREL/EGIS L counterstay system

Series ELU EMLU EJLU





ELU

Dimension table · Dimen	nsions in mm					
Designation	Designation <sup>1</sup> )			Dimensions		
Unit				A <sub>1</sub>	H <sub>1</sub>	C <sub>8</sub>
	ELU	EMLU	EJLU			max.
LUE 5025	ELU 5025	-	-	28,6	11,3	6,5
	-	EMLU 5025	1	24	31,1	6,5
	-	-	EJLU 5025	24	29,2	6,5
LUE 6035	ELU 6035	-	1	38,6	13,8	6,5
	-	EMLU 6035	-	34	40,5	6,5
	-	-	EJLU 6035	34	39	6,5
LUE 7040	ELU 7040	-	-	38,6	15,8	6,5
	-	EMLU 7040	-	39	46,5	6,5
	-	-	EJLU 7040	39	44	6,5
LUE 8050	ELU 8050	-	1	49,6	18,8	6,5
	-	EMLU 8050	-	49	53,5	6,5
	-	-	EJLU 8050	49	49	6,5

<sup>1</sup>) The end pieces are supplied with fixing screws to ISO 7380.



Insert nuts for guideways Series ESM



ESM

Dimension table · Dimen	nsions in mm			
Designation <sup>1</sup> ) Dimensions			Appropriate guideway	
	М	D <sup>-0,05</sup> -0,1	h	
ESM M4	M4	8,5	4,3	M 3015, V 3015
ESM M5 / D10	M5	10	5,5	J 3525, S 3525, J 4025, S 4025
ESM M6	M6	11,5	6,5	M 4020, V 4020, M 4525, V 4525, M 5025, V 5025, J 5025, S 5025, J 5030, S 5030, J 5530, S 5530, ML20, ML25
ESM M8	M8	15	8,5	M 6035, V 6035, M 6535, V 6535, J 6035, S 6035, ML35
ESM M10	M10	18,5	10,5	M 7040, V 7040, J 7040, S 7040, ML40
ESM M12	M12	20	12,5	M 8050, V 8050, J 8050, S 8050, M 8550, V 8550, ML50
	-	-	-	•

 Insert nuts must be ordered separately. They are included loose in the delivery and must be fixed in the counterbore by means of adhesive (page 43, Figure 10).

Insert nuts ESM for screws to grade 8.8.



Needle roller flat cages

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<b>⊦⊘</b> +	Accuracy	98
Sp.	Special designs	99
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### Features

#### Flat cages

- consist of:
  - a basic cage made from metal or plastic depending on the series
  - a large number of rolling elements guided precisely in pockets
- have a low section height
- have high load carrying capacity
- have high rigidity
- require hardened and ground surfaces for use as raceways
  are supplied as individual parts and in combination with
- are supplied as individual parts and in combination with guideways.

#### Needle roller flat cages

- have needle rollers as rolling elements
  - the needle rollers have profiled ends, i.e. the outside surface of the roller has a curved surface running towards the ends. This reduces the edge stresses at the ends of the rolling elements
- have the highest rigidity of any flat cage type
- are available as flat cages or angled flat cages.

 Needle roller flat cages

 Image: Second sec

for temperatures up to +120 °C





Cylindrical roller flat cages Ball flat cages

A	F	age
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<b>F</b> @+	Accuracy	98
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### Features

#### Cylindrical roller flat cages

- have cylindrical rollers as rolling elements
  - the needle rollers have profiled ends, i.e. the outside surface of the roller has a curved surface running towards the ends. This reduces the edge stresses at the ends of the rolling elements
- have a load carrying capacity similar to that of needle roller flat cages
- have a slightly lower rigidity than needle roller flat cages
- are available as flat cages or angled flat cages.

#### Ball flat cages HB

- have balls as rolling elements arranged in several rows in two point contact with the raceways
- have a high static load carrying capacity due to the large number of small balls
- have the lowest friction of any flat cage type.







Cylindrical roller flat cages Ball flat cages



#### Design and safety guidelines

The raceways for the cages must be produced in accordance with the requirements for rolling bearing raceways; alternatively, HYDREL/EGIS guideways must be used.

Design of raceways

- HYDREL/EGIS guideways are suitable as raceways. If these guideways cannot be used, machine parts with appropriate characteristics can be used as raceways instead. It must be ensured that:
  - the raceway roughness does not exceed Ra 0,3
  - the raceway hardness is at least 670 HV (58 HRC). If the hardness is below this value, the basic load ratings (dimension table) must be multiplied by the hardness factors (Load carrying capacity and life, Figure 2, page 22).
- The cage must be in the correct longitudinal position before preloading and initial operation.
- The cages must not be deformed during unpacking and fitting.
- The cages must be protected against contamination during fitting.

#### Further information

ner	information	Page
	Load carrying capacity and life	18
	Preload	
	Friction	
	Rigidity	
	Lubrication	
	Sealing, operating limits	
	Design of bearing arrangements	
	Fitting	46



#### Accuracy

- Needle rollers in quality grade G2 in accordance with DIN 5402-3
- Cylindrical rollers in quality grade GN in accordance with DIN 5402-1
- Balls in quality grade in accordance with DIN 5401-1 - G5 for flat cages of series HB
- Excerpt from DIN 5402: see table.

Rolling elements	to DIN		Roundness µm	Sort tolerance µm
Needle rollers	DIN 5402-3	3 G2	1	2
		G1 <sup>1</sup> )	0,5	1
Cylindrical rollers	DIN 5402-1	GN	1	2
		G1	0,5	1
Balls	DIN 5401	G20	0,5	2
		G5	0,13	1

<sup>1</sup>) Not included in DIN 5402.



### Special designs

Available by agreement:

- cages made from different materials
  - steel, suffix F
  - brass, suffix MS
- higher accuracy of rolling elements
- cages with anti-corrosion protection
- suffix BKcoating for reduced friction
  - suffix BR
- cages bent into circular shapes.



#### Crdering example and ordering designation

Two H needle roller flat cages Rolling element diameter 3 mm Cage width 20 mm Cage length 400 mm Ordering designation: 2 off H 20×400 (Figure 1). Two HW angled needle roller flat cages for closed arrangement, steel cage

Rolling element diameter3 mmCage shank length25 mmCage length280 mmOrdering designation:

2 off HW 25×280 F (Figure 2).



Figure 1 · Ordering example, ordering designation



Figure 2 · Ordering example, ordering designation



Single row Series FF H BF HR



Dimension	table · Dim	nensions ir	n mm														
Rolling	Designatio	n		Mass for	Dimens	sions											
element diameter					L <sub>K</sub> = 1000 mm	A <sub>1</sub>	D <sub>W</sub>	L <sub>w</sub>	t	е	а	L <sub>K</sub> <sup>3</sup> )	l <sub>e</sub>	Z <sub>e</sub> <sup>1</sup> )			
	FF	Н	BF	HR	g							max.					
2	FF 2010	-	-	-	46	10	2	6,8	-	-	2	-	32	7			
	-	H 10	-	-	63	10	2	6,8	4,5	3,5	-	3 000	-	-			
2,5	FF 2515	-	-	-	84	15	2,5	9,8	-	-	2,5	-	45	8			
	-	H 15	-	-	120	15	2,5	9,8	5	3,5	-	3 000	-	-			
3	FF 3020	-	-	-	148	20	3	13,8	-	-	3	-	60	9			
	-	H 20	-	-	202	20	3	13,8	6	4,5	-	3 000	-	-			
	-	-	BF 3020	-	342	20	3	15,8	6	4,5	-	2 000	-	-			
3,5	FF 3525	-	-	-	221	25	3,5	17,8	-	-	3	-	75	10			
	-	H 25	-	-	294	25	3,5	17,8	7	5	-	3 000	-	-			
5	-	-	-	HR 50	105	10,5	5	5	10	6,5	-	3 000	-	-			
	-	-	BF 5015	-	375	15	5	11,8	8	5,5	-	2 000	-	-			
	-	-	BF 5023	-	530	23	5	19,8	8	5,5	-	2 000	-	-			
	-	-	BF 5032	-	722	32	5	27,8	8	5,5	-	2 000	-	-			
7	-	-	-	HR 70	295	17	7	10	13	8,5	-	3 000	-	-			
	-	-	BF 7028	-	875	28	7	24	11	7,5	-	2 000	-	-			
	-	-	BF 7035	-	1080	35	7	30	11	7,5	-	2 000	-	-			
10	-	-	-	HR 100	598	24	10	14	17	10	-	3 000	-	-			
12	-	-	BF 12022	-	1220	22	12	18	16	10	-	2 000	-	-			
	-	-	BF 12040	-	1970	40	12	36	16	10	-	2 000	-	-			

<sup>1</sup>) Number of rolling elements in one cage element.

 <sup>2</sup>) For a theoretical cage length of 100 mm in the load direction according to the figure. Calculation of basic load ratings for effective cage lengths and calculation of safety factors and life: see page 18ff.

<sup>3</sup>) Length tolerance:  $+0/-1 \times t$ .



Basic load ratings <sup>2</sup> )		Mounting dimensi	ons		Appropriate guideways
С	C <sub>0</sub>	C <sub>a</sub> C <sub>a1</sub> H <sub>a</sub>		H <sub>a</sub>	
Ν	Ν		min.		
21 300	61 900	10,3+0,2	7	1,7	J 3525, S 3525
21 600	62 800	10,3+0,2	7	1,7	J 3525, S 3525
32 700	92 300	15,3+0,2	10	2,2	J 4025, S 4025, J 5025, S 5025
35800	103 800	15,3+0,2	10	2,2	J 4025, S 4025, J 5025, S 5025
47 800	133 200	20,4+0,2	14	2,7	J 5030, S 5030
51 900	148 000	20,4+0,2	14	2,7	J 5030, S 5030
57 800	170 100	20,4+0,2	16	2,7	J 5030, S 5030
64 700	177 300	25,4+0,2	18	3,2	J 5530, S 5530
68 200	190 000	25,4+0,2	18	3,2	J 5530, S 5530
29 400	50 800	10,9+0,2	5	3,4	-
69 900	154 700	15,3+0,2	12	4,6	-
106 400	265 100	23,4+0,2	20	4,6	-
139500	375 600	32,5 <sup>+0,3</sup>	28	4,6	-
65 800	114 200	17,4+0,2	10	4,8	-
150 800	331 800	28,4+0,2	24	6,5	-
179800	416 200	35,6+0,3	30	6,5	-
109 900	174 200	24,4+0,2	14	6,5	-
178 800	288 300	22,4+0,2	18	11	-
447 600	938 600	40,5 <sup>+0,3</sup>	36	11	-







Double row Series FF..ZW H..ZW HR..ZW



Dimension	table · Dimension	ns in mm												
Rolling	Designation		Mass for	Dimensions										
element diameter				L <sub>K</sub> = 1000 mm	A <sub>1</sub>	A <sub>2</sub>	D <sub>w</sub>	Lw	t	е	а	L <sub>K</sub> <sup>3</sup> )	I <sub>e</sub>	Z <sub>e</sub> <sup>1</sup> )
	FFZW	HZW	HRZW	g								max.		
2	-	H 19 ZW <sup>4</sup> )	-	219	19,2	8	2	4,8	4	3	-	1000	-	-
	FF 2025 ZW	-	-	94	25	10	2	6,8	-	-	2	-	32	7
	-	H 24 ZW	-	138	24	10,5	2	6,8	4,5	3,5	-	3 000	-	-
2,5	FF 2535 ZW	-	-	182	35	15	2,5	9,8	-	-	2,4	-	45	8
	-	H 34 ZW	-	239	33,5	14,3	2,5	9,8	5,5	4	-	3 000	-	-
3	FF 3045 ZW	-	-	315	45	20	3	13,8	-	-	3	-	60	9
	-	H 44 ZW	-	408	44	19	3	13,8	6	4,5	-	3 000	-	-
3,5	FF 3555 ZW	-	-	464	55	25	3,5	17,8	-	-	3,2	-	75	10
	-	H 55 ZW	-	598	55	24	3,5	17,8	7	5	-	3 000	-	-
5	-	-	HR 50 ZW	215	24	10,5	5	5	10	6,5	-	3 000	-	-
7	-	-	HR 70 ZW	602	40	17	7	10	13	8,5	-	3 000	-	-
10	-	-	HR 100 ZW	1233	55	24	10	14	17	10	-	3 0 0 0	-	-

<sup>1</sup>) Number of rolling elements per row in one cage element.

<sup>2</sup>) For a theoretical cage length of 100 mm in the load direction according to the figure. Calculation of basic load ratings for effective cage lengths and calculation of safety factors and life: see page 18ff.

<sup>3</sup>) Length tolerance:  $+0/-1 \times t$ .

4) Steel basic cage.



FF 2025 ZW





H..ZW

HR..ZW



Mounting dimensions





# Angled flat cages

Series FFW HW HRW



FFW

Dimension table · Dimensions in mm																				
Rolling	Designation			Mass for	Dimen	sions														
element diameter				L <sub>K</sub> = 1000 mm	A <sub>1</sub>	A <sub>2</sub>	D <sub>w</sub>	Lw	t	е	а	L <sub>K</sub> <sup>5</sup> )	l <sub>e</sub>	Z <sub>e</sub> <sup>1</sup> )						
	FFW	HW	HRW	g								max.								
2	-	HW 10 <sup>3</sup> )	-	219	10	8	2	4,8	4	3	-	1 0 0 0	-	-						
	FFW 2025	-	-	94	15	10	2	6,8	-	-	2	-	32	7						
	-	HW 15 <sup>4</sup> )	-	138	14	10,5	2	6,8	4,5	3,5	-	3 0 0 0	-	-						
	-	HW 16	-	190	16	13,5	2	8,8	3,8	2,8	-	3 0 0 0	-	-						
2,5	FFW 2535	-	-	182	20,5	15	2,5	9,8	-	-	2,4	-	45	8						
	-	HW 20 <sup>4</sup> )	-	239	20	14,3	2,5	9,8	5,5	4	-	3 0 0 0	-	-						
3	FFW 3045	-	-	315	26	20	3	13,8	-	-	3	-	60	9						
	-	HW 25	-	408	25	19	3	13,8	6	4,5	-	3 0 0 0	-	-						
3,5	FFW 3555	-	-	464	31,5	25	3,5	17,8	-	-	3,2	-	75	10						
	-	HW 30	-	598	30	24	3,5	17,8	7	5	-	3 0 0 0	-	-						
5	-	-	HRW 50	215	15,5	10,5	5	5	10	6,5	-	3 0 0 0	-	-						
7	-	-	HRW 70	602	25	17	7	10	13	8,5	-	3 0 0 0	-	-						
10	-	-	HRW 100	1233	34	24	10	14	17	10	-	3 0 0 0	-	-						

<sup>1</sup>) Number of rolling elements per row in one cage element.

 <sup>2</sup>) For a theoretical cage length of 100 mm in the load direction according to the figure. Calculation of basic load ratings for effective cage lengths and calculation of safety factors and life: see page 18ff.

<sup>3</sup>) Steel basic cage.

<sup>4</sup>) Higher basic load ratings by agreement.

<sup>5</sup>) Length tolerance:  $+0/-1 \times t$ .





HRW

Basic load ratings <sup>2</sup> )		Appropriate guideways
C	C <sub>0</sub>	
Ν	N	
21 400	68 700	M 3015, V 3015
25 900	87 500	M 4020, V 4020, ML 5020, ML 5520, M 5025, V 5025, ML 5525 to ML 7025
26 200	88 900	M 4020, V 4020, ML 5020, ML 5520, M 5025, V 5025, ML 5525 to ML 7025
36 900	138 100	M 5025, V 5025, ML 5525 bis ML 7025
39 600	130 500	M 6035, V 6035, ML 7035, ML 8035
40 300	133 500	M 6035, V 6035, ML 7035, ML 8035
57 900	188 400	M 7040, V 7040, ML 8040, ML 9040
62 900	209 400	M 7040, V 7040, ML 8040, ML 9040
78 400	250 800	M 8050, V 8050, ML 9050, ML 10050
82 700	268 700	M 8050, V 8050, ML 9050, ML 10050
35 700	71 900	M 4525, V 4525
79700	161 500	M 6535, V 6535
133 200	246 300	M 8550, V 8550







### Ball flat cages

Series HB



ΗB

Dimensio	Dimension table · Dimensions in mm													
Ball	Designation	Mass for L <sub>K</sub> = 1000 mm g	Dimens	ions				Basic load	d ratings <sup>1</sup> )	Appropriate guideway				
diameter			A <sub>1</sub>	D <sub>w</sub>	e	t/2	L <sub>K</sub> max.	dyn. C N	stat. C <sub>0</sub> N					
2,5	HB 2515	95	15	2,5	4,5	3	3 000	3330	3040	J 4025, S 4025, J 5025, S 5025				
3	HB 3020	167	20	3	4	3,5	3 000	5350	5000	J 5030, S 5030				
3	HB 3023	187	23	3	5,5	3,5	3 000	5350	5000	-				
4	HB 4025	250	25	4	5	5	3 000	7630	6220	J 5530, S 5530				

 For a theoretical cage length of 100 mm in the load direction according to the figure. Calculation of basic load ratings for effective cage lengths and calculation of safety factors and life: see page 18ff.



Load direction


# Needle roller flat cages with friction damping

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Ex	Features108
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<b>+ O</b> +	Accuracy
Sp.	Special designs
Amar	Ordering example and ordering designation109
	Dimension table

## Features

#### Needle roller flat cages with friction damping

- consist of:
  - a large number of rolling elements
  - special metal intermediate elements which are arranged in a row by means of tie rods. The assembly is preloaded longitudinally by spring elements. This preload leads to the frictional damping force between the rolling elements and intermediate elements in the direction of movement
- are used where table systems are designed for very high positional accuracy and the table must not oscillate about its nominal position
- prevent internally or externally induced oscillations in the direction of movement or rapidly dissipate these oscillations
- leiminate the need for additional damping elements
  - they have the advantage that the damping force acts directly on the moving parts
  - they prevent hysteresis effects due to the use of transmission elements
- are free from stick slip even at very low speeds
  - due to the minimal surface pressure on the sliding surfaces of the rolling elements and the intermediate elements
- have a damping force RS of 0,1‰ of the basic static load rating C<sub>0</sub> as standard – related to one row of rolling elements.







#### Design and safety guidelines

#### Design of raceways

- M, ML, V, J and S guideways are suitable as raceways. If these guideways cannot be used, machine parts with appropriate characteristics can be used as raceways instead. It must be ensured that:
  - the raceway roughness does not exceed  $R_a 0.3$
  - the raceway hardness is at least 670 HV (58 HRC).
     If the hardness is below this value, the basic load ratings (*dimension table*) must be multiplied by the hardness factors (*Load carrying capacity and life*, Figure 2, page 22).

#### Cage length





Cages with friction damping must always be located over their whole length between the guideways. If this is not the case, please consult the INA engineering service.

The information on cage length given in the section *Design of bearing arrangements*, page 40 must be noted!

#### Damping force and preload

- If a damping force other than the standard value is required, this must be stated in the order. The permissible range is between 0 and 2 RS.
- In order to make full use of the damping force, the guidance system must be subjected to a minimal preload
  - preload approx. 0,005 C<sub>0</sub>.

#### Total friction

The total frictional torque consists of the rolling friction due to the load and the damping force independent of the load. If the cage is correctly installed, the coefficient of rolling friction  $\mu_R$  is approximately 0,001.

#### $\Sigma R = RS + \mu_R \cdot F$

 $\begin{array}{ccc} \Sigma R & N \\ Total frictional force \\ RS & N \\ Damping force \\ \mu_R & - \\ Coefficient of rolling friction \\ F & N \\ Load on the guidance system. \end{array}$ 

#### Operating temperature

 Cages with friction damping can be used at temperatures up to +150 °C.



#### 

Needle rollers in quality grade G2 in accordance with DIN 5402-3.



## Special designs

- Differing damping forces RS – suffix ..RS
- Special designs by agreement
- Needle rollers in higher quality grades.



#### Crdering example and ordering designation

Needle roller flat cage for a flat cage guidance system with M and V guideways:

Guideway sizes M 4020 and V 4020 Angled needle roller flat cage with friction damping HGW 15 Cage length 400 mm.

Ordering designation:

1 off HGW 15×400 (Figure 1).







## Needle roller flat cages with friction damping

Series HG HGW



НG

Dimension table	$\mathbf{e} \cdot \mathbf{D}$ imensions in	mm								
Needle roller	Designation		Mass for	Dimensions						
diameter				A <sub>1</sub>	A <sub>2</sub>	D <sub>w</sub>	L <sub>w</sub>	t		
	HG	HGW	g							
2	HG 10	-	130	10	-	2	6,3	4,5		
	-	HGW 15	265	13,5	10	2	6,3	4,5		
2,5	HG 15	-	230	15	-	2,5	9,8	5		
	-	HGW 20	470	19,5	15	2,5	9,8	5		
3	HG 20	-	375	20	-	3	13,8	6		
	-	HGW 25	760	25	20	3	13,8	6		
3,5	HG 25	-	560	25	-	3,5	17,8	7		
	-	HGW 30	1150	30,5	25	3,5	17,8	7		

 For a theoretical cage length of 100 mm in the load direction according to the figure. Calculation of basic load ratings for effective cage lengths and calculation of safety factors and life: see page 18ff.

<sup>2</sup>) For a cage length of 100 mm. Calculation of damping force for effective cage lengths  $RS_w = RS \cdot \frac{L_K}{100}$ .







Basic load ratings <sup>1</sup> )		Standard	Appropriate guideway				
	dyn. C	stat. damping force RS <sup>2</sup> )					
	N	Ν	N				
	18 300	50 800	4,5	J 3525, S 3525			
	21 900	70 500	9	M 4020, M 5025, ML 5020, ML 5520, ML 5525 to ML 7025, V 4020, V 5025			
	31 700	88 700	8	J 4025, S 4025, J 5025, S 5025			
	38000	123 800	16	M 6035, ML 7035, ML 8035, V 6035			
	47 700	132 800	11	J 5030, S 5030			
	57 200	185 500	22	M 7040, ML 8040, ML 9040, V 7040			
	61 300	165 700	14	J 5530, S 5530			
	73800	232 100	28	M 8050, ML 9050, ML 10050, V 8050			





## Order form

# Flat cage guidance systems

Closed arrangement M/V, ML/V

# M/V guidance system

ML/V guidance system



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set/sets
000/0010
mm
stamp and signature

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+41/71/466 66 90

## Order form

# Flat cage guidance systems

Open arrangement M/V, J/S

#### M/V guidance system



J/S guidance system



Company stamp and signature

Quantity	Designation	n (see dimen	sion tables)		Hole pitc $C_5$ + i $\times$	ches (see $C_4 + C_6$	dimension ta	bles)		
	Guideway Guideway Guideway Guideway or Flat cage or End piece End piece End piece Or Wiper Wiper or	J S M ML FF H FFW HW/HRW EJ EV EM EM EM EM EAV EAM EAM			Grade Intended Stroke of Commer Delivery o	+ + + + + + for for guidance nts date	X X X X    	+		mm
					Date			Co	ompany stam	np and signature

### HYDREL AG

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# Zwischentitel



# Linear recirculating roller guidance systems

Linear recirculating roller guidance systems comprise a bearing arrangement system for linear motion with unlimited stroke. Since the guidance elements have a versatile range of possible arrangements, linear recirculating roller guidance systems are suitable for numerous applications in general machine building, especially for linear guidance systems in machine tools where high guidance and positioning accuracies are required over long traverse distances.

Linear recirculating roller guidance systems consist of:

- INA linear recirculating roller bearings with cylindrical rollers as rolling elements and
- HYDREL guideways with up to four raceways for the bearings. In combination with the bearings, the guideways give high precision linear guidance systems.

Linear recirculating roller guidance systems:

- have very high load carrying capacity
- can be preloaded by means of adjusting gibs
- have extremely high rigidity under preload
- have high accuracy throughout their operating life
- have consistently low friction
- can be used to achieve compact designs matched to the specific application
- are suitable for locating/locating and locating/non-locating bearing arrangements
- can in a locating/locating bearing arrangement support loads from all directions and moments about all axes.
- The range of recirculating guidance systems is complemented by:
- INA adjusting gibs for precise setting of preload
- INA setting devices for measuring the deformation of the adjacent construction where preload forces must be supported
- HYDREL adapters. These adapters
  - provide a simple means of installing the linear recirculating roller bearings in 45° arrangements
  - are suitable for both open and closed arrangements.

INA-Schaeffler KG Linear Technology Division Homburg (Saar)

HYDREL AG Romanshorn (Switzerland)

# Product range

Overview/selection scheme

	Characteristic	Accuracy			Load	Rigidity	Friction	Travel velocity
					carrying			
		Q2	Q6	Q10	1 5			max
Guidance syste	m							
RUS / RUSKS	Community M							50 m/min to 100 m/min <sup>2)</sup>
PR								120 m/min
	mann							
RUSW								100 m/min
UG, UGN, UGS, UGSN	C C C C C C C C C C C C C C C C C C C		•					
UZ, UZN, UZS, UZSN								
UV	0							
UFA, UFB, UFK								
APUV, AUV, AUVL, AV, AVL								
<ul> <li>Standard desi</li> <li><sup>1</sup>) Without seals:</li> </ul>	ign. : −40 °C to +120 °C.			<ul> <li>Optional.</li> <li><sup>2</sup>) Depender</li> </ul>	nt on series.			

1) Without seals: –40 °C to +120 °C.

Operating temperature	Raceways	Relubrication via end piece	Accessories			Features
	Quantity		Adjusting gib	Setting device	Wipers	See page
-30 °C to +90 °C¹)		(RUSKS)				174
-40 °C to +120 °C						175
−30 °C to +90 °C						175
	4					190 191
	3					<u>190</u> 191
	2					191
	1					191
						202

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#### Appendix

Addresses

# **Product index**

sorted alphanumerically

Туре	Description
AK	End face wiper for adapters, for screw mounting, steel support element, seal lips made from vulcanised NBR plastic
AL	Longitudinal wiper for adapters, for screw mounting, light metal support element, seal lips made from vulcanised NBR plastic
APUV	Adapter for use with UV guideway, screw mounting through locating face, with dowel hole, mainly for open arrangement systems
AUV	Adapter for use with UV guideway, screw mounting through lateral locating face, with dowel hole, mainly for closed arrangement systems
AUVL	Adapter for use with UV guideway, screw mounting through lateral locating face, with adjusting gib, mainly for closed arrangement systems
AV	Adapter for use with V guideway, screw mounting through lateral locating face, mainly for closed arrangement systems
AVL	Adapter for use with V guideway, screw mounting through lateral locating face, with adjusting gib, mainly for closed arrangement systems
EUS	Setting device for determining the preload dimension for linear recirculating roller bearings
HS	Hollow filling screw for filling slot in guideways of series UGN, UGSN, UZN, UZSN
PR	Linear recirculating roller bearing, full complement, inch dimensions
RUS	Linear recirculating roller bearing, spacers between cylindrical rollers, metric dimensions
RUSKS	Linear recirculating roller bearing, spacers between cylindrical rollers, relubrication facility via end pieces, metric dimensions
RUSW	Linear recirculating roller bearing designed as angled unit, roller recirculation channels arranged at right angles to each other, roller chain preloaded, chain axially guided by metal plates, relubrication facility via end pieces, sealed on all sides
	Type AK AL APUV AUV AUVL AV AVL EUS HS HS RUS RUS.KS RUS.KS

Page	Туре	Description
191	UFA	Guideway with one raceway, recesses for adhesive bonding to adjacent construction
191	UFB	Guideway with one raceway, through holes and counterbores for fixing screws
191	UFK	Guideway with one raceway, for clamping to adjacent construction by means of clamping strip UKB
190	UG	Guideway, rectangular cross-section, with four raceways, through holes and counterbores for fixing screws
190	UGN	Guideway, rectangular cross-section, with four raceways, milled continuous slot for square steel bar in order to support high lateral loads, through holes and counterbores for fixing screws
191	UGS	Guideway, rectangular cross-section, with four raceways, threaded blind holes
191	UGSN	Guideway, rectangular cross-section, with four raceways, milled continuous slot for square steel bar in order to support high lateral loads, threaded blind holes
201	UKB	Clamping strip for locating guideways UFK
191	UV	Guideway, trapezoidal cross-section, with two raceways arranged at an angle of 45° to the mounting face, through holes and counterbores for fixing screws
191	UZ	Guideway, rectangular cross-section, with three raceways, through holes and counterbores for fixing screws
191	UZN	Guideway, rectangular cross-section, with three raceways, milled continuous slot for square steel bar in order to support high lateral loads, through holes and counterbores for fixing screws
190	UZS	Guideway, rectangular cross-section, with three raceways, threaded blind holes
190	UZSN	Guideway, rectangular cross-section, with three raceways, milled continuous slot for square steel bar in order to support high lateral loads, threaded blind holes
176	VUS	Adjusting gib for linear recirculating roller bearings RUS, metric dimensions
176	VUSZ	Adjusting gib for linear recirculating roller bearings PR, inch dimensions

# Index of suffixes

Sorting criteria: A–Z

Suffix	Definition
KL 02	Guideway UFA with adhesive bonding gap 0,02 mm
L	Asymmetrical hole pattern for guideways of series UG – left handed design
LA/	Indication of end distances if these are outside the limit values in the dimension table and/or are asymmetrical. Indication of $C_5/C_6$ by: $C_5$ first distance $C_6$ last distance
QH	Guideways UG, UZ with raceway accuracy only in relation to dimension H. Lateral surfaces preground only
Q2	Accuracy class, high precision
Q6	Accuracy class, precision
Q10	Accuracy class, normal
R	Asymmetrical hole pattern for guideways of series UG – right handed design
S	Linear recirculating roller bearings or guideways matched and sorted together
VGS	Preground guideways, all surfaces with grinding allowance
VQ	Preground guideways, mounting face finish ground, other surfaces with grinding allowance
Х	Non-identical guideway designs in guideways matched and sorted together

# Symbols and units

Unless stated otherwise in the text, the values used in this catalogue have the following symbols, units and definitions.

A	cm <sup>2</sup>	Total piston surface
C.	N	Basic dynamic load rating
Cu	N	Effective dynamic load rating at reduced hardness
С <sub>Н</sub>	N/um	Rigidity of guidance system
C <sub>2</sub>	N	Basic static load rating
Cou	N	Effective static load rating at reduced bardness
C <sub>0H</sub>	mm	Effective saddle plate length
C (	mm	Hole nitch
$C_4$	mm	Distance between start/end of quideway and nearest hole
65, 06 f.,	_	Dynamic hardness factor
'H fou	_	Static hardness factor
'OH F	N	Operating load
F.	N	Variable load
ri Es	N	Operating load
ч В Бъ	N	Displacement resistance
	N	Displacement resistance of table
F	N	Calculated preload
'v H	mm	Stroke
k-	_	Load correction factor under misalignment
K. K. K.		Correction factors for load stroke and environmental
rsp, rsw, rsu		influences
K <sub>LF</sub>	_	Bearing factor
L	m	Basic rating life in 100 000 m displacement of guideway or
		of movable table
L	mm	Length of guideway
L <sub>h</sub>	h	Basic rating life in hours
n	-	Maximum possible number of hole pitches
n <sub>osc</sub>	min <sup>-1</sup>	Frequency of reciprocating motion
р	-	Life exponent
р	bar	Pressure required
Р	Ν	Equivalent dynamic load
P <sub>0</sub>	Ν	Maximum equivalent static load
q <sub>i</sub>	%	Proportion of total duration
S <sub>0</sub>	-	Static load safety factor
t <sub>f</sub>	h	Basic lubrication interval in hours
t <sub>fG</sub>	h	Guide value for lubricant operating life in hours
t <sub>fR</sub>	h	Guide value for relubrication interval in hours
V	m/min	Mean traverse speed
Vi	m/min	Variable speed
V	m/min	Equivalent dynamic speed
Х	-	Number of holes
$\delta_{B}$	μm	Deflection under operating load
μ	-	Coefficient of friction

## Load carrying capacity and life

The size of the linear recirculating roller guidance system required is dependent on the demands made on its:

- load carrying capacity
- life
- operational reliability.

The load carrying capacity is described using:

- $\blacksquare$  the basic static load rating C\_0 according to the dimension table and the
- basic dynamic load rating C according to the dimension table.

#### Basic load ratings according to DIN and to Japanese industrial practice

The calculation of basic load ratings for linear recirculating roller guidance systems is based on DIN 636/ISO 281.

Basic dynamic load ratings in accordance with DIN 636 are calculated on a basic rating life of 100 000 m. Suppliers from the Far East often give basic load ratings based on a rating life of 50 000 m. This results in basic load ratings that may appear to be more than 20% higher than those according to DIN 636. Table 1 and Table 2 show how the basic load ratings should be converted.

Table 1 · Conversion of basic load ratings according to DIN into basic load ratings according to Japanese industrial practice

Linear recirculating roller bearing Designation	Conversion factor	
RUS, RUSKS, PR, RUSW	$C_{50\ 000} = 1,23 \cdot C_{DIN}$	

Table 2 · Conversion of basic load ratings according to Japanese industrial practice to basic load ratings according to DIN

Linear recirculating roller bearing	Conversion factor
Designation	
RUS, RUSKS, PR, RUSW	$C_{DIN} = 0.81 \cdot C_{50,000}$

#### Dynamic load carrying capacity and life

The dynamic load carrying capacity of the linear recirculating roller guidance system is determined by the fatigue behaviour of the material.

- The dynamic load carrying capacity is described in terms of:
- the basic dynamic load rating (dimension table)
- the basic rating life.

The rating life as a fatigue period depends on:

the load acting on the guidance system

- the traverse speed of the guidance system
- the statistical probability of the first appearance of failure.

#### **Basic rating life**

The basic rating life is reached or exceeded by 90% of a sufficiently large group of apparently identical linear recirculating roller guidance systems before the first evidence of material fatigue occurs (Calculation example, page 131).

The basic rating life is determined using the following formulae. The life calculation formulae are based on the assumption that the load and speed remain constant. If the load and speed are not constant, see Equivalent load and speed, page 127.

$$L = \left(\frac{C}{P}\right)^{10/3}$$
  

$$h = \frac{8,33 \cdot 10^5}{H \cdot n_{osc}} \cdot \left(\frac{C}{P}\right)^{10/3}$$
  

$$-h = \frac{1666}{\bar{v}} \cdot \left(\frac{C}{P}\right)^{10/3}$$

Basic rating life in 100 000 m displacement of guideway or of movable table

\_h Basic rating life in hours

Ν

Basic dynamic load rating according to dimension table

Ν Equivalent dynamic load.



C

Ρ

Н

According to DIN 636, the equivalent dynamic load should not exceed  $P = 0.5 \cdot C!$ mm

Single stroke length of reciprocating motion

min<sup>-1</sup> n<sub>osc</sub> Frequency of reciprocating motion

m/min Mean travel speed equivalent dynamic speed.

#### Equivalent load and speed

Non-constant operating conditions can be taken into consideration by means of equivalent operating values. These have the same effect on the life as the loads occurring in practice.

#### Equivalent dynamic load

#### General formula

$$P = \Pr\left(\int_{0}^{T} |v(t) \cdot F^{p}(t)| dt\right) - \left(\int_{0}^{T} |v(t)| dt\right)$$

Load varying in steps

$$P = p \sqrt{\frac{q_1 \cdot F_1^p + q_2 \cdot F_2^p + ... + q_z \cdot F_z^p}{100}}$$

Load and speed varying in steps

$$P = p \sqrt{\frac{q_1 \cdot v_1 \cdot F_1^p + q_2 \cdot v_2 \cdot F_2^p + \dots + q_z \cdot v_z}{q_1 \cdot v_1 + q_2 \cdot v_2 + \dots + q_z \cdot v_z}}$$

Equivalent dynamic speed General formula

$$\bar{v} = \frac{1}{T} \int_{0}^{T} |v(t)| c$$

Speed varying in steps

$$\bar{v} = \frac{q_1 \cdot v_1 + q_2 \cdot v_2 + ... + q_z \cdot}{100}$$

P N Equivalent dynamic load

 $\begin{array}{ccc} p & - \\ Life exponent = 10/3 \\ q_i & \% \\ Proportion of total duration \\ F_i & N \\ Variable load \\ V_i & m/min \\ Variable speed \\ \overline{v} & m/min \\ Equivalent dynamic speed. \end{array}$ 

#### **Operating life**

The operating life is defined as the life actually achieved by a linear recirculating roller guidance system. The operating life may deviate from the calculated life (basic rating life). Possible reasons are wear and/or fatigue due to:

 misalignment between the guideways and guidance elements

- contamination of the guidance system
- inadequate lubrication
- excessive preload
- reciprocating motion with very small stroke length (false brinelling)
- vibration during stoppage (false brinelling).

Since there is a wide range of possible installation and operating conditions, it is not possible to calculate the operating life of a linear recirculating roller guidance system precisely in advance. The most reliable method of achieving a good estimate of the operating life is by comparison with similar applications.

#### Static load carrying capacity

#### Basic static load rating

The basic static load rating (*dimension table*) is the load under which a permanent deformation of the raceways and rolling elements occurs which corresponds to 1/10 000 of the rolling element diameter.

#### Static load carrying capacity

The static load carrying capacity is limited by:

- the permissible loading of the linear recirculating roller bearings
- the hardness of the raceways
- the permissible load on the screw connections
- the permissible load on the adjacent construction.

Attention must also be paid to the static load safety factor required for the application!

Check the frictional location of the fixing screws if lateral forces are present!

#### Static load safety factor

The static load safety factor  $S_0$  indicates the security with regard to permissible permanent deformation in the rolling contact without affecting the guidance accuracy and smooth running of the bearing.

$$S_0 = \frac{C_0}{P_0}$$

S<sub>0</sub> – Static load safety factor

 $\begin{array}{cc} C_0 & N\\ Basic static load rating according to dimension table \end{array}$ 

P<sub>0</sub> N Maximum equivalent static load.

If high demands are placed on accuracy and smoothness of running, for example in machine tools, the static load safety factor should not be less than  $S_0 = 3$ .

#### Factors influencing the load carrying capacity

The basic load ratings (dimension tables) are only valid under certain conditions. Correction factors must be applied in order to take account of:

- deviating raceway hardness
- misalignment.

#### Hardness factor

The basic load ratings in the dimension tables are defined for a raceway hardness of  $\geq$ 670 HV ( $\geq$ 58 HRC) with the fine structure characteristic of rolling bearing parts. If the raceway hardness is lower, the basic load rating is reduced to the value  $C_{\text{H}} \text{ or } C_{\text{OH}}.$  The reduced basic load ratings can be calculated using the correction factor formula.



The hardness factors are only valid for rolling bearing steels or similar alloy steels with corresponding purity and structure! The correction factors must not be used for other materials such as cast and non-ferrous metals!

Correction factor formula for effective dynamic load rating at reduced hardness

 $C_H = f_H \cdot C$ 

Correction factor formula for effective static load rating at reduced hardness

 $C_{OH} = f_{OH} \cdot C$ 

 $C_{H,} \ C_{0H} \qquad N$  Effective dynamic or static load rating at reduced hardness

f<sub>H</sub>, f<sub>OH</sub> Dynamic or static hardness factor (Figure 1)

C, C\_0  $$\rm N$$  Basic dynamic or static load rating according to dimension table.



Figure 1 · Dynamic and static hardness factors

#### Reduction in life due to misalignment

The life calculation formulae (page 126) are based on the assumption that the guidance elements are positioned correctly. If misalignment occurs, for example due to elastic deformation of the counterstay, the rolling elements are subjected to non-uniform load along the contact line.

Correction factor formula for load under misalignment

 $P = k_F \cdot F$ 

P N Equivalent dynamic load

k<sub>F</sub> – Correction factor for load under misalignment (Figure 2)

F N Bearing load.



Figure 2 · Correction factor for load under misalignment



## Calculation example

RUS 26102				
C = 95000 N				
$C_0 = 75000$ N				
F <sub>B</sub> = 15000 N				
H = 500 mm				
$n_{\rm OSC} = 8 \ {\rm min^{-1}}$				
$F_V = 8000$ N				
Calculate the load on the linear recirculating roller bearings from the preload diagram (Figure 3). Deduct the operating load $F_B$ and read off $F_1$ , $F_2$ on the Y axis:				
$1 F_1 = 20000 N$				
$2 F_2 = 5000 N$				
L and L <sub>h</sub>				

 $S_0$ 

Basic rating life	
Static load safety factor	

#### Ν 1 30 000 F<sub>1</sub> 20 000 Operating load F<sub>B</sub> 15 000 FB F<sub>V</sub> F<sub>2</sub> 5 000 10 000 (2) 0 30 μm 40 20 10 δΒ Deflection $\delta$ > Elastic deformation of linear roller bearing Elastic deformation of counterstay 126 057

40 000

Figure 3 · Preload diagram for counterstay guidance system with two linear recirculating roller bearings RUS 26 102

## Life calculation

$$L = \left(\frac{C}{P}\right)^{10/3}$$

For P, use the equivalent load  $F_1$ .

$$L = \left(\frac{95\ 000}{20\ 000}\right)^{10/3} = 180 \cdot 10^5 \text{ m}$$
  
$$\cdot_{h} = \frac{8.33 \cdot 10^5}{\text{H} \cdot \text{n}_{\text{osc}}} \cdot \left(\frac{\text{C}}{\text{P}}\right)^{10/3}$$
  
$$\cdot_{h} = \frac{8.33 \cdot 10^5 \cdot 180}{500 \cdot 8} = 37\ 500\ \text{h}$$

The life is sufficient for the more highly load bearing.

#### Calculation of static load safety factor

$$S_0 = \frac{C_0}{F_1}$$

$$S_0 = \frac{75\,000}{20\,000} = 3,75$$

The static load safety factor is sufficient.

## Preload

#### Preload

- increases the rigidity and guidance accuracy of the guidance system
- gives favourable load distribution under moment loading
- prevents clearance occurring in the guidance system in all operating conditions
- prevents slippage of the cylindrical rollers, thus ensuring correct rolling of the rolling elements
- affects the displacement resistance and the life of the guidance system.

#### Influence of the adjacent construction

In order to ensure optimum conversion of preload into rigidity, the adjacent construction must be rigid and geometrically accurate (see *Design of bearing arrangements*, page 148).

#### Preload - guide value

As a guide value, the preload force may be taken as between 10% and 20% of the basic dynamic load rating C (*dimension table*) of the linear recirculating roller bearing used.

If the preload is too low, the rigidity of the system will be reduced and the guidance system may lift under load! If the preload is too high, the life is reduced and the friction is increased!

#### Calculation of preload force

In order to calculate the preload force precisely, the following must be known:

- the deflection of the adjacent construction
- the operating load (see *Rigidity*, page 136).

#### Setting the preload

The preload can be set by means of:

- adjusting gibs VUS / VUSZ
- shims
- pressure screws.

#### Setting preload by means of adjusting gibs

These elements can be used to set the preload easily and precisely to the required preload dimension (Figure 1). The gibs transmit the preload uniformly over the whole length of the linear recirculating roller bearing.

#### Shims

Shims are preground, economical design elements. They are finish ground to the calculated preload dimension when the guidance system is fitted.

Further information on the production of shims can be obtained from INA.

#### Pressure screws

Guidance systems with linear recirculating roller bearings RUSW can be preloaded by means of pressure screws (Figure 2).

- The pressure screws should:
- have an even contact surface (dowel screw to DIN 913, ISO 4 026)
- be arranged between the fixing screws.

For pressure screws M6, the tightening torque  $M_{\text{A}}$  is 1,1 Nm (preload 0,1  $\cdot$  C).

#### Calculation of preload using the setting device EUS

The setting device EUS can be used to measure the deformation of the adjacent construction under preload forces. This deformation, when added to the deflection of the linear recirculating roller bearing under the preload force  $F_V$  (see deflection curves, page 138 and page 139), gives the required preload dimension.

The core of the setting device is the setting block with a pressure piston. This block has the same dimensions as the linear recirculating roller bearing to be fitted.

The setting block can be connected via a pressure pipe and an intermediate distributor block and integral manometer to a conventional grease gun.



Figure 1 · Preloading the guidance system using an adjusting gib



Measuring the preload dimension using the setting device EUS and setting the preload (Figure 3)

- Fit the setting device EUS ① in place of the linear recirculating roller bearing.
- Connect the grease gun (5) to the distributor block with the manometer (3) and pressure pipe (4).
- Position a dial gauge (i) at a suitable point in order to measure the deformation.
- Calculate the required pressure using the formula.
- Apply pressure using a conventional grease gun continuously until the required pressure is reached on the manometer 3.
- Read of and note the deformation travel on the dial gauge 6.
- Determine the deflection of the linear recirculating roller bearing under the preload force F<sub>V</sub> (deflection curves, page 138 and page 139).
- Add the deflection of the adjacent construction to that of the linear recirculating roller bearing to give the required preload dimension
- Detach the setting device and fit the linear recirculating roller bearing.
- Set the adjusting gib to the preload dimension or finish grind the shim to the preload dimension.

#### Required pressure

=	$\frac{F_v}{10\cdot A_K}$			

p bar Required pressure

р

F<sub>v</sub> N Calculated preload force

A<sub>K</sub> cm<sup>2</sup> Total piston area (*dimension table*)

#### Influence of preload on displacement resistance

 $\mathsf{F}_{RV} = \mu \cdot \Sigma \; \mathsf{F}_V$ 

F<sub>RV</sub> N Displacement resistance of table

\_

Ν

Coefficient of friction (*Friction*, page 135, Table 1)

#### F<sub>V</sub> Preload.



 $$\sum$$  Influences due to lubrication or seals and the mass of the table are not taken into consideration!



Figure 3 · Measuring the deformation travel using the setting device EUS / setting the preload dimension

## Friction



Linear recirculating roller guidance systems have a consistently low coefficient of friction throughout their operating life. They are therefore free from stick-slip in contrast to plain guidance systems. The displacement force of linear recirculating roller guidance systems is only about 2% up to a maximum of 10% of the displacement force of plain guidance systems (Figure 1).

Since the rolling elements are guided in parallel, the coefficient of friction is very low (Table 1)!

Due to the lower displacement resistance, linear recirculating roller guidance systems have:

- lower drive power requirements
- reduced deformation in the elastic machine parts
- higher positional accuracy.

The friction is temporarily increased by fresh grease at initial operation and during regreasing. However, the friction coefficient returns to its original lower value after a short running-in period.

#### Seal friction

In linear recirculating roller guidance systems with wipers, the seal friction is at its highest with new guidance systems.

During the running-in phase, the geometry of the seal lips gradually becomes matched to the profile of the guideway. As a result, the seal friction decreases again.

#### **Displacement resistance - calculation**

The displacement resistance of linear recirculating roller bearings can be approximated using the following formula:

 $F_R = F \cdot \mu$ 

 F<sub>R</sub>
 N

 Displacement resistance

 F
 N

 Operating load of linear recirculating roller bearing

μ – Coefficient of friction (Table 1).

#### Table 1 · Coefficient of friction

Linear recirculating roller guidance system Series	Coefficient of friction $\mu$
RUS	0,003
RUSKS	0,0025
RUSW	0,0035
PR	0,004

The values given in Table 1 are only valid if the required accuracy is achieved and if the lubrication is appropriate to the application!



Figure 1 · Comparison of displacement force – linear recirculating roller guidance system/plain guidance system

# Rigidity

#### Rigidity of the guidance system

If a linear recirculating roller bearing is subjected to an operating load F<sub>B</sub>, it undergoes elastic deformation of a magnitude  $\delta_B$  (Figure 1). The measured deflection curves of linear roller bearings are shown in Figure 7, page 138 and Figure 8, page 139.

The rigidity  $c_S$  of the guidance system is derived from the ratio between the operating load and deflection (formula).



 $c_S = N/\mu m$ Rigidity of the guidance system

F<sub>B</sub> N Operating load

 $\delta_B$   $\mu$ m Deflection under operating load.

The formula does not take into consideration the elastic deformation of the adjacent construction, screw connections, settling effects, etc! Since the adjacent construction is not completely rigid, the deformation of the complete structure can be higher in practice!

## Elastic deformation of linear recirculating roller bearings acting in opposition

## Linear recirculating bearings installed clearance-free without preload

Where two linear recirculating roller bearings set clearance-free without preload act in opposition to each other, only one linear roller bearing is subjected to load and elastically deformed. The linear roller bearing without load has clearance corresponding to the deflection of the load bearing (deflection curve, Figure 2).

## Linear recirculating bearings installed clearance-free with preload

If two linear recirculating roller bearings acting in opposition are subjected to a preload force  $F_V$  of magnitude  $\delta_{V_i}$  this gives the preload diagram in Figure 3.

If the system is subjected to an operating load  $\mathsf{F}_B$ , it undergoes deformation of magnitude  $\delta_B$  (Figure 3). The linear roller bearing  $\mathsf{R}_1$  is deformed by  $\delta_1$  and the linear roller bearing  $\mathsf{R}_2$  by  $\delta_2$ . The system remains clearance-free up to the maximum operating load  $\mathsf{F}_B$  max. In this range, the rigidity remains approximately twice that of the individual linear recirculating roller bearing.



Figure 1 · Deflection curve of a linear roller bearing



Figure 2 · Deflection curve of linear roller bearings without preload



Figure 3 · Preload diagram of linear roller bearings with preload

Influence of the adjacent construction on rigidity for linear recirculating roller bearings acting in opposition

Since the counterstay and the screw connections are elastic, the deflection curve becomes shallower in the opposing direction.

The deflection characteristics of the counterstay can be determined by measuring its elastic deformation, e.g.:

with the setting device EUS (*Preload*, page 134).

The deflection curve for the counterstay (Figure 4, red) is derived from adding together the elastic deformation of the linear roller bearing  $R_2$  and the counterstay (Figure 4, red). The deflection curve for the linear roller bearing  $R_1$  is shown

in green (Figure 4).

With operating loads in the opposite direction (F<sub>B II</sub>) – e.g. due to moment loads – it is possible that even under relatively small loads (F<sub>B II</sub> > F<sub>B II max</sub>) clearance will occur on the linear roller bearing R<sub>1</sub>. This clearance can be prevented by increased preload or higher rigidity of the counterstay (Figure 5).

 $\Sigma$  If the rigidity is to be fully utilised, the adjacent construction must have sufficient rigidity and geometrical

accuracy (*Design of bearing arrangements*, page 148).

#### Calculation example

#### Given data

Linear recirculating roller bearing	RUS 26102
Operating load at most highly loaded point	$F_{B} = 15\ 000\ N$
Preload force	$F_V = 8000N$
Deflection under operating load (derived from Figure 6)	δ <sub>B</sub> = 10 μm

Reference to Figure 6:

- $\blacksquare\,$  the deflection curve for the linear roller bearing intersects the curve for the counterstay at the point for the preload  $F_V$
- the operating load F<sub>B</sub> between the deflection curves is deducted
- The deflection  $\delta_B$  is derived from the distance between the intersection of the deflection curves for the bearing and counterstay and the points at which the operating load F<sub>B</sub> is in contact with the deflection curves.

#### Required

Rigidity of the guidance system in the load direction.

Calculation

 $F_{B}$ 

$$c_{\rm S} = \frac{15000 \,\text{N}}{\delta_{\rm B}}$$
  
 $c_{\rm S} = \frac{15000 \,\text{N}}{10 \,\mu\text{m}} = 1500 \,\text{N}/\mu\text{m}$ 



Figure 4 · Linear recirculating roller bearing preloaded, with counterstay











Figure 7 · Measured deflection curve for linear recirculating roller bearings RUS





Figure 8 · Measured deflection curve for linear recirculating roller bearings PR

## Lubrication

Lubrication is an important element for the reliable function and long operating life of the linear recirculating roller guidance system.

Lubricants (either grease or oil) perform the following functions:

- they reduce friction
- they minimise wear
- they prevent corrosion
- they give protection against contamination
- they increase the operating life of the guidance system.

#### Lubrication as a design element

Lubrication is a design element. The lubricant and lubrication method must therefore be selected in the development phase of the machine. If lubrication of the guidance system is only determined once the design is complete, experience shows that considerable problems are likely to occur.

The lubricant requirement has an increasingly important role. The lubricant consumption should be as small as possible, especially in the case of oil lubrication. A carefully considered lubrication concept is therefore indicative of a modern linear guidance system.

#### Environmental protection

Any lubrication method involves loss of lubricant. The lubricant consumed must be collected and disposed of by methods which help to protect the environment.



The handling and use of lubricants is governed by

national regulations for environmental protection and health and safety at work as well as information from lubricant manufacturers. These specifications must be observed.

#### **Doped lubricants**

Linear recirculating roller bearings operate almost exclusively under mixed friction conditions, especially at low speeds. Doped oils and greases (type P to DIN 51502) should be used in preference.

#### **Oil lubrication**

#### Advantages of oil lubrication

Oil:

- dissipates heat from the bearing
- is distributed more evenly in the lubrication area than grease
- is replaced almost completely when relubrication is carried out
- carries contaminant particles out of the bearing.

Furthermore, oil lubrication is advisable where the adjacent machine elements are already supplied with oil.

#### Oils

Lubricating oils CLP to DIN 51517 and HLP to DIN 51524 should be used in preference.

For operating temperatures between 0 °C and +70 °C, the viscosity should be between ISO VG 32 and ISO VG 68.

For low-temperature operation, oils to ISO VG 10 or ISO VG 22 should be used.

Slideway oils CGLP can also be used up to ISO VG 220.



Do not use drilling oils or other coolant emulsions for lubrication! These have the effect of thinning the lubricants and can lead to corrosion in certain circumstances

The preservative applied to linear roller bearings is compatible with the oils.

#### Lubrication methods

Oil impulse or oil drop lubrication is recommended.

In highly contaminated environments, pneumatic oil lubrication is particularly advantageous. This generates a slight excess pressure within the guidance system. The effectiveness of the seals is reinforced as a result.

Note the installation position of the guidance system (Figure 1). The lubricant feed arrangement should be selected such that all the moving parts of the linear roller bearings are supplied with oil:

- e.g. in the return zone of the linear roller bearings (Figure 2) or
- the oil pipe should, in place of the lubrication nipple, lead directly into the end piece (Figure 3).

#### Compatibility and miscibility

Before lubricating oils are used, their behaviour with respect to plastics, elastomers, light metals and non-ferrous metals must be checked under dynamic conditions at operating temperature if no past experience or data from the manufacturer is available.

In general, oils with a mineral oil base and with the same classification are miscible with each other. However, the viscosities should be within one ISO VG class of each other.

If synthetic oils are to be used, their miscibility and compatibility must always be checked.

If in any doubt, please consult the INA engineering service.



Figure 1 · Installation position of linear recirculating roller bearings



Figure  $2 \cdot \text{Oil}$  feed through the adjusting gib



Figure 3 · Oil feed through the end piece

#### Initial operation / minimum oil quantities

 $\bigwedge$  Protect the bearing and guideway system against solid and fluid contaminants!

Linear recirculating roller bearings and guideways should be oiled before initial operation. During this process, linear roller bearings should be moved several times consecutively without load by at least four times the length of the bearing.

The minimum oil quantities according to Table 1 are valid under the following standardised conditions:

- 100% operating duration
- $\Box C_0 / P = 8$
- 🔲 v = 0,8 m/s
- stroke = 500 mm to 1000 mm.

 $\hfill \hfill \hfill$ 

Minimum oil quantity Q<sub>min</sub> – Table 1:

- the minimum oil quantity is measured such that the oil ducts, rolling elements and raceways will be supplied with sufficient quantities of lubricant.
- Oil impulse lubrication Table 1:
- the oil impulse quantity applies when the recirculating lubrication system is connected to a central lubrication system. It is recommended that the stated quantity should be spread over several impulses.

Pneumatic oil lubrication:

Pneumatic oil lubrication may allow smaller quantities of oil to be used than oil impulse lubrication. It is not possible to state definitive quantities since these are essentially dependent on the design of the central lubrication system. The necessary oil quantity must therefore be determined under operating conditions.

#### Table 1 · Minimum oil quantities – guide values

Linear recirculating roller bearing Series	$\begin{array}{l} \mbox{Minimum oil quantity} \\ \mbox{for initial operation} \\ \mbox{Q}_{min} \\ \mbox{cm}^3 \end{array}$	Oil impulse quantity Q <sub>imp</sub> cm <sup>3</sup> /h
RUS 19 069	0,35 to 0,5	0,25
RUS 19 105	0,35 to 0,5	0,25
RUS 26 086	0,35 to 0,5	0,25
RUS 26 102	0,35 to 0,5	0,25
RUS 26 126	0,6 to 0,8	0,5
RUS 38 134	0,6 to 0,8	0,5
RUS 38 206	1,5 to 2	1
RUS 65 210	0,8 to 1,2	1
RUS 85 280	2,8 to 3	2
PR 14 032	0,25 to 0,4	0,25
PR 14 044	0,25 to 0,4	0,25
PR 14 061	0,25 to 0,4	0,25
PR 14 089	0,6 to 0,8	0,5
PR 14 135	0,8 to 1,2	1
PR 14 182	2,5 to 2,8	2
RUSW 4020	0,35 to 0,5	0,25



#### **Grease lubrication**

#### Advantages of grease lubrication

Grease:

- requires very little design work on relubrication devices if a central lubrication system is not used
- allows relubrication intervals of up to one year
- has emergency running characteristics due to the thickener in the lubricant
- gives additional support to the seals.

#### Greases

Lithium soap greases with a mineral oil base should be used in preference. The base oil viscosity should be between:

ISO VG 150 and ISO VG 220.

For high loads (S $_0$  <8), the following are absolutely necessary:

EP doped greases with a base oil viscosity of ISO VG 220. A lubricating grease KP2N-20 to DIN 51825 is recommended for initial greasing.

 $\angle$  Lubricants with solid additives must not be used. The preservative applied to linear roller bearings is compatible with greases having a mineral oil base.

#### Miscibility

Greases may be mixed providing the following preconditions are met:

- they have the same base oil type
- their thickener types must match
- they must have similar base oil viscosities (the difference must be no more than one ISO VG class)
- their consistencies (NLGI class) must match.

If in any doubt, please consult the INA engineering service.

#### Storage

Experience shows that lubricating greases having a mineral oil base can be stored for up to three years. The following preconditions apply:

- the storage room is protected against outside influences
- the temperature is between 0 °C and +40 °C
- the relative humidity is less than 65%
- the lubricants are protected against chemical agents (vapours, gases, fluids).

After long storage periods, the frictional torque may temporarily be higher than that of freshly greased linear recirculating roller bearings. The lubricity of the grease may also have deteriorated.

It is the user's responsibility to follow the directions given by the lubricant manufacturer.
#### Initial operation / initial grease quantity

 $\bigwedge$  Protect the bearing and guideway system against solid and fluid contaminants!

Linear recirculating roller bearings and guideways should be greased before initial operation. In order to ensure uniform distribution of the grease in the bearing, the linear roller bearings should be moved several times consecutively without load by at least four times the length of the bearing! Regreasing should be carried out continuously during this time!

The initial grease quantities are listed in Table 2.

On very long guidance systems, the guideways should be coated with lubricant before initial operation so that the grease reservoir from initial greasing is not used up prematurely.

If the guidance system is not connected to a central lubrication system, the linear roller bearings should be charged with the initial grease quantity before fitting.

Where central lubrication systems are used, the linear roller bearings should be charged with the initial grease quantity and the feed pipes filled with grease.

#### Relubrication

Linear recirculating roller bearings can be relubricated via the rolling element return zone or via lubrication nipples (Figure 4).

Relubrication should be carried out with several partial quantities at shorter intervals in preference to a single regreasing at the end of the relubrication interval.

During relubrication, linear roller bearings should be moved several times without load by at least four times the length of the bearing.

The relubrication quantities are listed in Table 2.



Figure 4 · Relubrication of linear recirculating roller bearings

Table 2 ·	Initial grease	and r	relubrication	quantities -	guide
	values				-

Linear recirculating roller bearing	Initial grease quantity	Relubrication quantity	
Series	g	g	
RUS 19 069	2,5	0,75	
RUS 19 105	3,5	1,05	
RUS 26 086	7	2	
RUS 26 102	7,5	2,2	
RUS 26 126	8	2,4	
RUS 38 134	18	5,4	
RUS 38 206	25	7,5	
RUS 65 210	26	8,6	
RUS 85 280	27	9	
PR 14 032	1	0,3	
PR 14 044	2	0,6	
PR 14 061	7	2,1	
PR 14 089	15	4,5	
PR 14 135	16	5,2	
PR 14 182	25	8,3	
RUSW 4020	2,5	0,75	

#### Relubrication interval

Linear recirculating roller bearings must be relubricated at appropriate intervals. The relubrication interval is essentially dependent on:

- the speed
- the load
- the temperature
- the stroke length
- the environmental conditions.

The relubrication interval and relubrication quantity can only be determined under operating conditions, since it is not possible to quantify all the influencing factors in advance. An observation period of adequate length must be allowed.

The relubrication interval  $t_{fR}$  should be no more than 1 year even if the formula gives a value greater than this.

$$t_{fR} = tf \cdot K_p \cdot K_w \cdot K_u$$

h

 $t_{fR} \qquad h \\ Guide \ value \ for \ relubrication \ interval \ in \ hours$ 

te

Basic lubrication interval in hours (Figure 5)

 $K_{p},\,K_{w},\,K_{u}$  \_ Correction factors for load, stroke and environmental influences (page 146).

#### Basic lubrication interval

The basic lubrication interval  $t_{\rm f}$  is valid under the following conditions (Figure 5):

- bearing temperature <+70 °C
- load ratio  $C_0/P = 20$
- Iubrication with high quality lithium soap grease
- no disruptive environmental influences
- stroke ratio between 1 and 10.

The speed parameter is defined as follows:

Speed parameter:  $\frac{60}{\bar{v}} \cdot K_{LF}$ 

v m/min Mean traverse speed

K<sub>LF</sub> – Bearing factor



Figure 5 · Determining the basic lubrication interval

#### Table 3 $\cdot$ Bearing factor $K_{LF}$

Linear guidance system	RUS RUSW	PR
Bearing factor K <sub>LF</sub>	1,5	1

#### Load correction factor $\ensuremath{\mathsf{K}_{\mathsf{p}}}$

The correction factor  $K_p$  takes into consideration the greater strain on the lubricating grease at a load ratio of  $C_0/P < 20$ .

The values given in Figure 6 are valid only for high quality lithium soap grease. The preload must be taken into consideration.

#### Stroke ratio correction factor K<sub>w</sub>

The correction factor  $K_w$  (Figure 7) takes into consideration the raceway length to be provided with grease. It is dependent on the stroke ratio.

If the stroke ratio is <1 or >10, the relubrication interval must be shortened in order to reduce possible fretting corrosion.

If the stroke is very short, the relubrication interval  $t_{fR}$  may be shorter than the calculated value. In such cases, the use of special lubricating greases is recommended. Please consult the INA engineering service.

The stroke ratio is defined as follows:

Stroke ratio:  $\frac{C_1 \cdot 10}{H}$ 

C1 mm Effective saddle plate length C1 (*dimension table*)

H mm Stroke length.

#### Environmental correction factor K<sub>u</sub>

The correction factor  $K_{\rm u}$  (Table 4) takes into consideration the effect of oscillations, vibration (leading to fretting corrosion) and shocks. These subject the grease to additional strain.

All calculations are invalid if cooling lubricants or moisture penetrate the system.

#### Lubricating grease operating life

If a linear system cannot be relubricated, the operating life of the lubricating grease is then the decisive factor. In most applications, the guide value is:

 $t_{fG} = 2 \cdot t_{fR}$ .

 $t_{fG} \qquad h \\ Guide \ value \ for \ grease \ operating \ life$ 







Figure 7  $\cdot$  Stroke ratio correction factor K<sub>w</sub>

correction	factor	Ku
	correction	correction factor

Environmental influences	K <sub>u</sub>
Slight	1
Moderate	0,8
Severe	0,5

## Sealing Operating limits

#### Sealing

The type of sealing or shielding is of decisive importance in ensuring problem-free operation and a long operating life of linear recirculating roller bearings.

Linear recirculating roller bearings RUS have interchangeable, elastic, double lip wipers on the end pieces. The wipers ensure that:

no contaminants enter the bearing

no lubricant escapes from the bearing.

In most applications, linear recirculating roller bearings are protected reliably against contamination by the wipers and the narrow gap between the saddle plate and raceway. In special cases, additional measures may be taken to cover the raceway.

If linear recirculating roller bearings are exposed to severe contamination (e.g. swarf, grinding dust, etc.) or aggressive media, and in the case of linear roller bearings PR, separate raceway wipers should be fitted!

#### **Operating limits**

Linear recirculating roller bearings are suitable for numerous applications. In appropriate arrangements, they can support loads from all directions and moments about all axes. They also have high load carrying capacity while requiring little space.

#### Traverse speed

Table 1 · Maximum traverse speed

The maximum traverse speeds of guidance systems with linear recirculating roller bearings are shown in Table 1.

· · · · · · · · · · · · · · · · · · ·	
Linear recirculating roller bearing	Traverse speed v <sub>max</sub> m/min
PR	120
RUS 19, RUS 19KS, RUSW 4020	100
RUS 26, RUS 26KS	80
RUS 38, RUS 38KS	60
RUS 65, RUS 85	50



If a guidance system is to be designed for very high speeds, please consult the INA engineering service.

#### Acceleration

If very high accelerations (shocks) occur and the linear roller bearings are fixed to the moving part (e.g. in a ram guidance system), additional support must be provided to the linear roller bearings. The inertia forces must not be supported by the friction lock on the screw connections.

The maximum accelerations of guidance systems with linear recirculating roller bearings are shown in Table 2.

#### Table 2 · Maximum acceleration

Linear recirculating roller bearing	Linear roller bearing on moving part Acceleration a <sub>max</sub> m/s <sup>2</sup>	Linear roller bearing on stationary part Acceleration a <sub>max</sub> m/s <sup>2</sup>
PR	70	160
RUS RUSKS	50	110



#### Load carrying capacity

The load carrying capacity of linear recirculating roller bearings is limited by:

- the required life L and L<sub>h</sub>
- the required static load safety factor S<sub>0</sub>.

For applications where high demands are placed on accuracy and smoothness of running, the static load safety factor should not be less than  $S_0 = 3$ .

#### Operating temperatures

Linear recirculating roller bearings RUS/RUSW are suitable for temperatures from -30 °C to +90 °C.

Linear recirculating roller bearings PR can be used at temperatures between -40 °C and +120 °C.

## Design of bearing arrangements

The adjacent construction has a significant influence on:

- the load carrying capacity
- the rigidity
- the accuracy
- the smoothness of running
- the operating life

of a guidance system with linear recirculating roller bearings.

When designing the adjacent construction, particular attention must therefore be paid to:

- the design of the raceways
- the geometrical and positional accuracy of the mounting surfaces
- the location of the guidance elements
- the sealing of the bearing arrangement.

#### Design of raceways

Linear recirculating roller bearings require hardened and ground guideways for use as raceways.

HYDREL guideways (page 190 and *dimension table*) are matched to the requirements of linear recirculating roller bearings. They can be used to achieve high precision, rigid linear recirculating guidance systems with high load carrying capacity and low friction.

If these guideways cannot be used, machine parts with appropriate characteristics can be used as raceways instead. Values for the effective hardness depth, hardness and roughness are given in Table 1.

In order to ensure that the high load carrying capacity and rigidity of the linear recirculating roller bearings can be used to the full, the raceways must conform to the accuracies of the guideways.

Table 1 ·	Design o	f raceways
-----------	----------	------------

Linear	Raceway			
recirculating roller bearing	Effective hardness depth	Hardness	Roughness	
Series	Rht	HV	R <sub>a</sub> (R <sub>z</sub> )	
RUS 19 069 to RUS 38 206 PR 14 032 to PR 14 089 RUSW 4020	Rht ≧0,6 mm	≥670 HV (58 HRC) +170 HV	R <sub>a</sub> 0,6 (R <sub>z</sub> 2,5)	
RUS 65 210 RUS 85 280 PR 14 135 PR 14 182	Rht ≧2 mm	≥670 HV (58 HRC) +170 HV	R <sub>a</sub> 0,8 (R <sub>z</sub> 4)	

## Geometrical and positional accuracy of the mounting surfaces

The higher the requirements for accuracy and smooth running of the guidance system, the more attention must be paid to the geometrical and positional accuracy. The mounting surfaces, i.e. the support and locating surfaces, should be designed according to the tolerances in Figure 1 and Figure 2. The surfaces can be ground or precision milled. The objective should be to achieve a mean roughness of  $R_a 1.6$ .

If the tolerances are not met, this will impair the overall accuracy of the guidance system and will influence the operating life. This applies despite the fact that linear recirculating roller guidance systems can partially compensate for inaccuracies.

#### Maximum permissible deviation

The differences  $\Delta H_Q$  and  $\Delta H_L$  (Figure 1 and Figure 2, page 149) indicate the maximum permissible deviation from the theoretically precise position of the mounting surfaces in the longitudinal and transverse axes.

The permissible values for linear recirculating roller guidance systems are:

$$\Delta H_L = a_L \cdot b$$

$$\Delta H_{Q} = a_{Q} \cdot b$$

 $\Delta H_L ~~\mu m$  Maximum permissible deviation in a longitudinal axis from the theoretically precise position

 $\Delta H_Q \qquad \mu m$  Maximum permissible deviation in a transverse axis from the theoretically precise position

 $a_L$ ,  $a_Q$  – Factor dependent on the series (Table 2)

b mm Centre distance between guidance elements.

If larger deviations are present, please consult the INA engineering service.

#### Table 2 · Factor dependent on series

Series	Factor		
	aL	a <sub>Q</sub>	
RUS, RUSKS, PR	0,1	0,15	



Figure 2 · Geometrical and positional accuracy of the mounting surfaces



Figure 1 · Geometrical and positional accuracy of the mounting surfaces in the longitudinal direction

#### Connection to the adjacent construction

The connection between the guidance elements and the adjacent construction influences the effective load carrying capacity of the guidance system. When designing the adjacent construction, particular attention must therefore be paid to:

- the direction of the forces
- the direction of the moments
- the position of the locating faces
- the size of the locating faces
- the load carrying capacity and number of fixing screws.

The better supported a guidance system in relation to the forces occurring, the greater the extent to which the load carrying capacity can be used.

#### Support of lateral guidance forces in one direction

If the friction lock of the screw connections cannot support the lateral guidance forces, the guideways must be laterally supported against a locating edge (Figure 3).

#### Support of lateral guidance forces in two directions

If high lateral forces occur in both directions, the guideways UZ, UZS, UV can be screw mounted in a slot (Figure 4). After fitting, the lateral gaps must be filled with castable resin (page 151, Figure 6, Figure 7).

#### Guideways UGN, UGSN, UZN and UZSN with longitudinal slot

These guideways have a continuous slot (Figure 5).

The guideways are linked to the adjacent construction by means of square steel bars to DIN 178. These transmit the lateral forces into the machine part.

After fitting, the lateral gaps must be filled with castable resin.



Figure 3 · Lateral locating face



Figure 4 · Guideway UV located in slot



Figure 5 · Guideway with longitudinal slot

#### Hollow filling screws

For filling with resin, hollow filling screws HS (Table 3) can be used (*Fitting*, page 162). Recesses should be milled 500 mm apart in the adjacent construction.

In order that the castable resin can reach these recesses, holes must be made in the adjacent construction (Figure 6, Figure 7, Table 4). Due to the combination of the recess and hole, the castable resin flows into the gap between the square steel bar and the adjacent construction.

#### Table 3 · Allocation of hollow screw to guideway

Guideway	Hollow screw	
Designation	Designation	
UGN/UGSN/UZN/UZSN 6628	HS 6628	
UGN/UGSN/UZN/UZSN 9741	HS 9741	
UGN/UGSN/UZN/UZSN 12553	HS 12553	
UGN/UGSN/UZN/UZSN 16260	HS 16260	

## Table 4 · Recommended design of recesses in adjacent construction for hollow filling screws

Guideway	Dimensions		
Designation	D <sub>a</sub> mm	E <sub>a</sub> mm	H <sub>a</sub> mm
UGN/UGSN 6628 / UZN/UZSN 6628	18	4	3,5
UGN/UGSN 9741 / UZN/UZSN 9741	25	6	6
UGN/UGSN 12553 / UZN/UZSN 12553	30	8	7
UGN/UGSN 16260 / UZN/UZSN 16260	30	7	9



Figure 6 · Hollow screws for guideways UGN/UZN



Figure 7 · Hollow screws for guideways UGSN/UZSN

#### Location of guideways

HYDREL guideways are located by means of:

- through holes with cylindrical counterbores for screws to DIN 912
  - series UG, UGN, UZ, UZN, UV, UFB
- threaded blind holes – series UGS, UGSN, UZS, UZSN
- adhesive bonding
  - series UFA (*Fitting*, page 166)
- clamping strip
  - series UFK (Fitting, page 167).

#### Location of guideways UG, UGN, UZ, UZN, UV, UFB

These guideways have through holes with cylindrical counterbores for the screw heads. They are located from the guideway side (Figure 8).

 $\hfill \hfill \hfill$ 

When the counterbores are closed off flush using closing plugs (*dimension table*) or castable resin, this gives a smooth guideway surface (Figure 9). This protects the wipers against damage. Contamination, coolants, etc. can not collect in the counterbores.

#### Location of guideways UGS, UGSN, UZS, UZSN

These guideways have threaded blind holes.

The guideways are located from the machine part (Figure 10). The guideway surface is completely unbroken as a result. The entire width of the guideway can therefore be used as a raceway. There is also no need to close off the cylindrical counterbores.

In order to transmit the tightening torque M<sub>A</sub> of the screw connections, the screws must be embedded in the holes to a length of approximately 2 · d!



Figure 8 · Location from the guideway side



Figure 9 · Closing off the counterbore



Figure 10 · Location from the machine part side

#### Adhesive bonding of guideway UFA

Guideways of this series are bonded by adhesive to the adjacent construction (Figure 11). Conventional metal adhesives are suitable for bonding (*Fitting*, page 166). The standard bond gap of 0,15 mm is matched to two-component adhesives. Anaerobic single-component adhesives that harden in the absence of air give satisfactory adhesion and higher accuracy with thin adhesive layers. For these applications, guideways UFA are available with a bond gap of 0,02 mm (suffix KL 02).

#### Location of guideways UFK with clamping strip

Guideways of this series are retained in the slot by means of a special clamping strip made from spring steel (accessory) – slot widths for the guideways (Table 5, Figure 12).

The clamping strip can be fitted using a shim (Fitting, page 167).

#### Table 5 $\cdot\,$ Slot width for guideways

Guideway	Clamping strip	Slot width B <sub>1</sub> +0,15
Designation	Designation	mm
UFK 3210	UKB 10	32,65
UFK 4710	UKB 10	47,65
UFK 6412	UKB 12	64,65
UFK 8815	UKB 14	88,65
UFK 11518	UKB 14	115,65



Figure 11 · Adhesive bonding of guideway UFA



Figure 12 · Slot width for guideways UFK

#### Multi-piece guideways

If the length required exceeds the maximum guideway length L for single-piece guideways (*dimension table*), guideways can be assembled from individual sections matched to each other and marked accordingly (*Fitting*, page 162).

The matching of the sections avoids significant differences in height at the joints in the guideways and thereby ensures the acceptable functioning of linear recirculating roller bearings.



In order to avoid impaired function of the bearings, the maximum height difference and gap width due to errors in assembly and in the adjacent construction must not be exceeded (Table 6, Figure 14).

The guideways matched to each other form a set. All parts of the same set have the same set number. In addition, the joints are marked consecutively by means of letters (Figure 13).

The guideways should be assembled such that the ends with the same set numbers and letters are adjacent to each other)!

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Guideway grade	Height difference	Gap width	
	ΔH μm max.	ΔS μm max.	
Q2 and Q6	2	0,05	
Q10	3	0,05	

#### Table 6 $\cdot$ Height difference and gap width

#### Overgrinding of preground guideways

In order to achieve acceptable results even when the adjacent construction is of reduced accuracy, preground guideways (suffix VQ) can be used.

 $\hfill \hfill \hfill$ 



Figure 13 · Marking of multi-piece guideways



Figure 14 · Height difference and gap width

#### Hole patterns for guideways

Unless specified otherwise, guideways are supplied with a hole pattern having identical end dimensions ( $C_5 = C_6$ , Figure 15(a)). On customer request, a hole pattern with non-identical end dimensions is also possible ( $C_5 \neq C_6$ , Figure 15(b)). Suffix L<sub>A</sub>  $C_{5}/C_{6}$ 

Note the position of the dimension  $C_5$  (definition of  $C_5$ according to Figure 16)!

Note C<sub>5min/max</sub>, C<sub>6min/max</sub>! Guideways UG, UGN, UGS, UGSN have offset holes (Figure 16). The position of the holes depends on the length of the guideway (dimension table).

#### Calculation of hole patterns

The number of hole pitches is the rounded down whole number equivalent to:

 $n = \frac{L - (C_{5\min} + C_{6\min})}{C_4}$ 

Dimensions C<sub>5</sub>, C<sub>6</sub>:

 $C_5 + C_6 = L - n \cdot C_4$ 

Guideway with identical end dimensions for holes

 $C_5 = C_6 = \frac{1}{2} \cdot (L - n \cdot C_4)$ 

Number of holes:

X = n + 1

n Maximum possible number of hole pitches

mm Guideway length

 $C_5$ ,  $C_6$  mm Distance between start or end of guideway and nearest hole (*dimension* table)

C<sub>4</sub> mm Hole pitch (*dimension table*)

Number of holes



Figure 15  $\cdot$  Hole patterns with identical and non-identical end dimensions



#### Location of linear recirculating roller bearings

Linear recirculating roller bearings are fixed to the machine part by means of hexagon socket cap screws to DIN 912 – screws dimensions according to *dimension table*. Location is possible from either the linear roller bearing or the machine part.

The adjacent construction must have sufficient strength – see VDI guideline 2230!

#### Location from the linear roller bearing side

For this type of location, the machine part has threaded holes. The linear roller bearings are aligned on the machine part and screwed to the machine part from the bearing side using fixing screws (Figure 17).

This type of location can be used for:

■ linear recirculating roller bearings RUS, PR and RUSW.

#### Location from the machine part side

For this type of location, the machine part has through holes and counterbores for the screw heads.

The linear roller bearings are aligned on the machine part and screwed to the machine part from the adjacent construction side using fixing screws (Figure 18).

This type of location can be used for:

linear recirculating roller bearings RUS and PR.

### Location of linear recirculating roller bearing/adjusting gib assembly

Adjusting gibs can be located from the gib side or adjacent construction side (Figure 19):

- adjusting gibs VUS from the gib side or adjacent construction side (①, ②)
- adjusting gibs from the gib side only (1).



Figure 17 · Location from the linear roller bearing side



Figure 18 · Location from the adjacent construction side



Figure 19 · Location of linear recirculating roller bearing/ adjusting gib assembly

## Location of adapter/linear recirculating roller bearing assembly

Adapters are screw mounted on the adjacent construction using hexagon socket cap screws to DIN 912 (Figure 20 and Figure 21):

- APUV through the locating face on the top (Figure 20)
  - high lateral forces can be supported by means of an additional dowel. The adapter has a predrilled dowel hole.
- AUV, AUVL, AV, AVL through the lateral locating face (Figure 21)
  - this type of location can support high lateral forces by means of force lock.
- For the location of adapters and for sorting, note the guidelines in the sections *Features* and *Design and safety guidelines*!

## Location of linear recirculating roller bearings on adjusting gib or adapter

Linear recirculating roller bearings are aligned and fixed to the the adjusting gib or adapter from the bearing side using hexagon socket cap screws to DIN 912.



Figure 20 · Location of adapters APUV



Figure 21 · Location of adapters AUV, AUVL, AV, AVL

#### Sealing of the bearing arrangement

In order to prevent damage to the running system of the linear recirculating roller bearings, the raceways must be kept clean!

The elastic wipers on the end pieces of the linear roller bearings give effective protection of the guidance systems against contamination.

In order to prevent damage to the wiper lips, the counterbores of the fixing screw holes must be closed off (*Fitting*, page 166).

The function and effectiveness of the wipers also depends on correct fitting of the linear recirculating roller bearings (*Fitting*, page 168).

If guidance systems are exposed to severe contamination or aggressive media, additional seals must be provided to protect the running system (example, Figure 22)!

#### Sealing of adapters

If severe contamination is present in the application, the rolling system can be provided with additional protection by fitting the adapters with additional wipers on the end faces and sides (Figure 23).

The wipers are supplied with fasteners.



Figure 22  $\cdot$  Sealing of the bearing arrangement – example



Figure 23 · Sealing of adapters

#### **Design examples**

HYDREL guideways are supplied in various designs (*Features*, page 190). These guideways can be used to achieve open and closed arrangements (*Open arrangements*, page 160, *Closed arrangements*, page 161).

Typical designs with HYDREL guideways and linear recirculating roller bearings are shown in Figure 24.

#### Examples

- Guideways UG, UGN:
  - have four raceways
  - support forces in the main load direction and opposing direction with a counterstay as well as lateral forces in two directions.
- Guideways UZ, UZN:
  - have three raceways
  - support forces in the main load direction and lateral forces in two directions
- Guideways UV:
  - have two raceways arranged at an angle of 45° to the mounting surface
  - support forces in the main load direction in two directions
- Guideways UFA, UFB, UFK:
  - have one raceway
  - support forces in the main load direction only.



#### Open arrangements

This arrangement (Figure 25):

- has one locating bearing side and one or more non-locating bearing sides
- is mainly used for applications with loads acting concentric and vertical to the guidance plane
- allows a large guidance base
- is very easy to fit, since the machine part can simply be placed on and lifted off.

Open arrangements are bearing arrangements without a counterstay!



Figure 25  $\cdot$  Open arrangements – examples

#### Closed arrangements

This arrangement (Figure 26):

- has one or two locating bearing sides
- has a counterstay on both sides
- is mainly used for applications with all types of load directions and moment loads
- is preloaded. This increases the rigidity and accuracy.



Figure 26 · Closed arrangements – examples

## Fitting

Linear recirculating roller bearings and guideways are high precision machine elements. These products must be handled very carefully before and during fitting. Their trouble-free operation depends largely on the care taken during fitting.

#### Delivered condition and storage

Delivered condition of linear recirculating roller bearings Linear recirculating roller bearings are supplied coated with an oil-based preservative.

Do not remove the preservative! If it is washed out, foreign bodies can enter the bearing and damage the running system!

The bearings are supplied in groups with a height sorting of dimension H of 5  $\mu$ m or 10  $\mu$ m.

The group is marked by a coloured sticker on the packaging. Unless specified otherwise in the order, the group GR 3 (yellow) is supplied in preference.

The suffix S with a number indicates linear recirculating roller bearings sorted to half the group tolerance (Accuracy, page 177).

#### Delivered condition of HYDREL guideways

Guideways

- are supplied with a preservative and are packed in anticorrosion paper
- guideways matched to each other form a set. All the individual parts in a set have the same set number, and the joints are marked consecutively with letters. All the parts of a set are packed together. If the parts must be packed in several units due to weight, these are marked accordingly.



Parts with the same set number must be fitted in the same guidance system! The guideways should be assembled such that the ends with the same set numbers and letters are adjacent to each other!

Storage of linear recirculating roller bearings and guideways Linear recirculating roller bearings and guideways should only be stored:

- in the original packaging. Packaging must be kept closed
- in dry, clean rooms with the temperature as constant as possible
- at a relative humidity of 65% max.

Contamination impairs the accuracy and reduces the • operating life of the guidance elements!



Figure 1 · Marking of multi-piece guideways

#### Unpacking of guidance elements

Perspiration from handling leads to corrosion. Hands must be kept clean and dry; protective gloves should be worn if necessary.

Guideways should only be removed from their packaging immediately before fitting. If the fitting procedure is very timeconsuming – for example with complex fitting processes – or fitting is interrupted, bearings should be protected against contamination by appropriate measures. Parts should be held covered in a clean, dry area.

 $\angle !$  Wire wool or fluff-forming cloths must not be used!

The bearings and guideways should be lightly oiled in order to prevent corrosion during fitting. The preservative present on the parts when supplied need not be removed.

#### Design of the assembly area

It must be ensured that work surfaces are bright, clean and free from fibres (e.g. plastic) and that lighting conditions are good (Figure 2).



Machines or equipment that produce swarf or generate dust must not be used in the immediate vicinity of the bearings!

The guidance systems must be protected against dust, contamination, swarf, moisture, adhesives, etc! Contaminants affect the operation and operating life of the guidance elements!

#### Cleaning the adjacent construction

In order to prevent mounting defects, the holes and edges of the adjacent parts must be free of burrs.

The mounting surfaces for the guideways and the locating faces for the linear roller bearings must be clean.



Legal specifications relating to the handling and use of cleaning agents (manufacturer's guidance and regulations covering health and safety at work and environmental protection) must be complied with! Cleaning agents must be disposed of correctly after use!

Suitable cleaning agents are:

 conventional grease solvents (isopropanol, petroleum, diesel oil).

#### Cleaning (Figure 3)

Apply cleaning agents using a brush or suitable cloth.

Clean and dry the surfaces.



Ensure that the adjacent components and lubrication holes are free from cleaning agents, solvents and washing emulsions! The fit surfaces can rust or the raceway system can become contaminated!



Figure 2 · Design of assembly area



Figure 3 · Cleaning the adjacent construction

#### Checking the dimensional and geometrical tolerances of the adjacent construction

The measurement method depends on:

- the measuring equipment used (e.g. Figure 4)
- the geometry of the adjacent components
- the requirements for running accuracy. If high running accuracy is required, it may be necessary to check the measuring machine.

#### Checking the mounting surfaces



The accuracy of the adjacent construction should not be checked if the adjacent components have been refrigerated or heated!

The mounting surfaces of the adjacent construction should not exceed the permissible dimensional and geometrical tolerances (Design of bearing arrangements, page 148 and page 149)!

#### Handling of fasteners

#### Screws, square steel bar

Linear recirculating roller bearings and guideways should only be located using the screws specified. The definitive information is given in the dimension tables.



The specifications relating to the fasteners must be 2! adhered to! Any deviations will affect the security of the screw connections as well as the accuracy, load carrying capacity, rigidity and operating life of guidance systems! Adequate strength of the adjacent construction in accordance with VDI guideline 2230 must be ensured!

Fixing screws are not included in the delivery.

For guideways UGN, UGSN, UZN and UZSN, a square steel bar to DIN 178 should be provided (dimension table, page 196).

#### Adhesives, clamping strips

Conventional metal adhesives of two-component types or single-component adhesives are suitable. If anaerobic singlecomponent adhesives are used, the guideways should have a reduced bonding gap - suffix KL-02.

Adhesives are not included in the delivery.

A clamping strip UKB in accordance with Table 5, page 153 should be provided.



Figure 4 · Checking the adjacent construction



#### Fitting of guideways

Alignment and screw mounting of guideways UG, UGN, UGS, UGSN, UZ, UZN, UZS, UZSN, UV and UFB

 $\wedge$ 

In order that the load carrying capacity, rigidity, accuracy and smooth running of the linear recirculating roller

guidance systems can be used to the full, the guideways must be precisely aligned!

The cylindrical counterbores of the fixing holes have sharp edges! Risk of injury!

Guideways UGN, UGSN, UZN, UZSN

Fitting of the square steel bar (Figure 5):

- Place the square steel bar in the centre of the slot in the adjacent construction
- Fix the square steel bar in place using at least two dowel pins  $\ensuremath{\mathbb O}$  or screws.
- Lightly oil the mounting and locating faces for the guideways
   ② on the adjacent construction
  - this prevents fretting corrosion.
- Insert the fixing screws ③ in the holes in the guideways and tighten finger tight (Figure 6).
- Position the guideways ② (Figure 6)
  - press the guideways against the lateral locating faces if necessary and clamp rigidly against the lateral locating faces by means of suitable devices (screw clamps or clamping fixtures).



All the holes for fixing screws should be used! If a smaller number of fixing screws is used, this will reduce the load carrying capacity of the screw connections and the rigidity of the guidance system!

If locating faces are not present, align the guideways precisely using a plastic hammer ④ and optical measuring device (e.g. light alignment device or laser) (Figure 7).



Figure 5 · Fitting the square steel bar



Figure 6 · Positioning of guideways on the adjacent construction



Figure 7 · Aligning the guideways

- Tighten the fixing screws using a torque wrench ⑤ in three stages to the specified tightening torque M<sub>A</sub> (Figure 8):
  - Stage 1 40% of MA
  - Stage 2 70% of MA
  - Stage 3 100% of MA

In order to prevent the occurrence of unacceptable

- $\frac{1}{2}$  stresses, the location sequence must be adhered to! Tighten the screws in the guideways from the centre in both directions!
  - If the screws are lubricated, the tightening torque MA should be reduced!
  - Check the alignment of the guideways after each stage!
- Guideways UGN, UGSN, UZN, UZSN: fill the space between the square steel bar and the slot with castable resin, ensuring that no bubbles are present. Allow the mass to harden.

For filling the gap, hollow filling screws HS can be used (Design of bearing arrangements, page 151) – comprehensive instructions can be requested.

Close off the counterbores in the guideways flush with the surface by means of closing plugs (dimension table) or castable resin.

#### Adhesive bonding of guideways UFA

Conventional metal adhesives are suitable for adhesive bonding

The standard bond gap of 0,15 mm is matched to twocomponent adhesives.



Anaerobic single-component adhesives that harden in the absence of air generally only give satisfactory results with thin adhesive layers!

For these applications, guideways UFA are available with a bond gap of 0,02 mm (suffix KL 02).

- Protect the mounting surfaces of the guideway against fretting corrosion by means of a thin layer of white fitting paste ① (Figure 9)
- Two-component adhesives should be prepared in accordance with the manufacturer's instructions.
- Apply adhesive 2 to the appropriate surface of the guideway (Figure 10)
- Locate and align the guideways ③ (Figure 10)
  - repeat the procedure for multi-piece guideways.
- Apply sufficient load to the guideways ④ or use clamping methods and allow the adhesive to harden in accordance with the adhesive manufacturer's guidance (Figure 10).



Figure 8 · Tightening of guideway to tightening torque MA



Figure 9 · Protecting the mounting surface using fitting paste



Figure 10 · Adhesive bonding of guideways/hardening of adhesive

#### Location of guideways UFK with clamping strip

- Lightly oil the mounting surfaces for the guideways on the adjacent construction
  - this prevents fretting corrosion.
- Position the guideways ① in the slot in the adjacent construction (Figure 11)
  - use the entire length of the clamping strip.

Press in the clamping strip <sup>(2)</sup> using a shim <sup>(3)</sup> (Figure 11).

#### Fitting of guideway sets

Guideways matched to each other form a set. All parts of the same set have the same set number. In addition, the joints are marked consecutively by means of letters.

 $\wedge$ 

Parts with the same set number must be fitted in the same guidance system! The guideways should be assembled such that the ends with the same set numbers and letters are adjacent to each other!

Fitting of quideways (see section *Fitting*, page 162)

 note the height difference and gap width (Figure 13, Table 1).

#### Table 1 · Height difference and gap width

Guideway grade	Height difference $\Delta H$ $\mu m$ max.	Gap width $\Delta S$ mm max.
Q 2 and Q 6	2	0,05
Q 10	3	0,05

#### Overgrinding of preground guideways after fitting

Experience shows that an acceptable grinding result can only be achieved by taking considerable care. Particular attention must be paid to:

- careful contact grinding
- Iow feed levels
- plentiful and correct supply of coolant.



Figure 11 · Location of guideways using clamping strip



Figure 12 · Marking/fitting of multi-piece guideways



Figure 13 · Height difference and gap width

#### Oiling/greasing of linear recirculating roller bearings

Linear roller bearings should be oiled or greased depending on the lubrication method.

#### Fitting of linear recirculating roller bearings

Direct blows and shocks on the linear roller bearings must be avoided! Fitting forces must never be directed through the rolling elements!

Linear recirculating roller bearings must never be fitted by force – e.g. by direct blows – in preloaded guidance systems!

It must be ensured that the seal lips are not damaged.

- Lightly oil the locating faces for the linear recirculating roller bearings on the adjacent construction
  - this prevents fretting corrosion.
- Insert the fixing screws ① in the holes in the linear roller bearings, tighten finger tight and align the bearings (Figure 14).
- Press the datum side of the linear recirculating roller bearings
   against the locating face of the adjacent construction (Figure 14)
  - the datum side is the unmarked side; it is on the opposite side to the marked side.

 $\Delta$  Check the parallelism (Figure 15); rework the adjacent construction if necessary!

- Tighten the fixing screws in crosswise sequence using a torque wrench in three stages to the specified tightening torque M<sub>A</sub> (Figure 16):
  - Stage 1 40% of M<sub>A</sub>
  - Stage 2 70% of MA
  - Stage 3 100% of MA



Check the alignment of the guideways after each stage!



Figure 14 · Prefitting of linear recirculating roller bearings



Figure 15 · Checking of parallelism



Figure 16  $\cdot$  Tightening of linear roller bearings to tightening torque M<sub>A</sub>

#### Fitting of adjusting gibs

If assemblies comprising a linear recirculating roller bearing and adjusting gib are to be fitted, the linear roller bearing and not the adjusting gib should be aligned against the locating face. For this purpose, the adjusting gibs are slightly narrower than the associated linear roller bearings.

The fitting guidelines in the section *Fitting of linear recirculating roller bearings* must be followed (page 168)!

Fitting of gib to linear recirculating roller bearing

- Separate the gib halves
- Screw the gib ① to the linear recirculating roller bearing ② and tighten the screws finger tight (Figure 17).
- Align the gib to the linear roller bearing.
- Tighten the fixing screws in crosswise sequence using a torque wrench ③ in three stages to the specified tightening torque M<sub>A</sub> (Figure 17)
  - Stage 1 40% of M<sub>A</sub>
  - Stage 2 70% of M<sub>A</sub>
  - Stage 3 100% of M<sub>A</sub>
  - In order to prevent the occurrence of unacceptable stresses, the location sequence must be adhered to! Check the alignment of the guideways after each stage!

#### Fitting of gib to adjacent construction

- Screw the gib 1 to the adjacent construction 5 and tighten the screws finger tight (Figure 18).
- Align the gib to the adjacent construction.
- Tighten the fixing screws using a torque wrench (6) to the specified tightening torque M<sub>A</sub> (Figure 18):
  - Stage 1 40% of M<sub>A</sub>
  - Stage 2 70% of MA
  - Stage 3 100% of M<sub>A</sub>.
- Assemble the two gib halves carefully.

#### Setting the preload

- Determine the preload dimension (*Preload*, page 132).
- Set the calculated preload dimension using the adjusting screw @ (Figure 19).
- Secure the setting by means of a locking screw (Figure 19).



Figure 17 · Fitting of gib to linear recirculating roller bearing



Figure 18 · Fitting of gib to adjacent construction



Figure 19 · Setting/securing the preload

## Fitting of adapter APUV and linear recirculating roller bearing

- Lightly oil the mounting and support surfaces for the adapter and linear recirculating roller bearing
  - this prevents fretting corrosion.

#### Fitting of adapters

- Adapters with the same set number must be fitted in the same guidance system. Adapters with additional marking by means of letters must be fitted in line.
- Press the adapter ① against the locating faces on the adjacent construction (Figure 20).
- Insert the fixing screws @ in the holes and tighten finger tight (Figure 20).
- Tighten the fixing screws ② in three stages to the specified tightening torque M<sub>A</sub> (Figure 20)
  - For the stages, see Fitting of linear recirculating roller bearings, page 168
- Fix the adapter to the adjacent construction by means of dowels using the dowel hole ③ (Figure 21)
  - Dimensions of dowel pins according to *dimension table*.

#### Fitting of linear recirculating roller bearings

- Press the datum side of the linear recirculating roller bearing
   ④ against the locating face of the adapter (Figure 22).
- Insert the fixing screws (5) and tighten finger tight.
- Tighten the fixing screws (5) in three stages to the specified tightening torque M<sub>A</sub> (Figure 22)
  - Stage 1 40% of M<sub>A</sub>
  - Stage 2 70% of M<sub>A</sub>
  - Stage 3 100% of M<sub>A</sub>.



Check the alignment of the guideways after each stage!



Figure 20  $\cdot$  Fitting of adapters



Figure 21 · Location of adapters using dowels



Figure 22 · Fitting of linear recirculating roller bearings

## Fitting of adapters AUVL, AVL and linear recirculating roller bearing

- Lightly oil the mounting and support surfaces for the adapter and linear recirculating roller bearing

   this prevents fretting corrosion.
- Treat the gib surfaces with fitting paste
  - this facilitates setting of the preload.

#### Fitting of linear recirculating roller bearings

This process can be omitted where completely assembled adapters are used (suffix KM).

- Press the datum side of the linear recirculating roller bearing
   ① against the locating face of the adapter (Figure 23).
- Insert the fixing screws (2) and tighten finger tight (Figure 23).
- Tighten the fixing screws ② in three stages (see *Fitting of adapters*) to the specified tightening torque M<sub>A</sub> (Figure 23).

#### Fitting of adapters

- Press the adapter ③ against the locating faces on the adjacent construction (Figure 24).
- Insert the fixing screws ④ in the holes and tighten finger tight (Figure 24).
- Determine the preload dimension (see *Preload*, page 132).
  - on the same basis as calculation of an individual bearing (see *Preload*, page 132)
- Set the calculated preload dimension (see *Preload*, page 132) using the adjusting screw (5) (Figure 24).
- Tighten the fixing screws  $\circledast$  in three stages to the specified tightening torque  $M_A$  (Figure 24)
  - Stage 1 40% of M<sub>A</sub>
  - Stage 2 70% of M<sub>A</sub>
  - Stage 3 100% of M<sub>A</sub>

#### Reference to dismantling of adapter

- Loosen the fixing screws ④ (Figure 24)
- Loosen the adjusting screw (5) for the adjusting gib by several turns (Figure 25).
- Continue the dismantling process by apply the fitting process in reverse.



Figure 23 · Fitting of linear recirculating roller bearings



Figure 24 · Fitting of adapters



Figure 25 · Dismantling of adapters

#### Fitting of adapters AV, AUV

- Lightly oil the mounting and support surfaces for the adapter and linear recirculating roller bearing
  - this prevents fretting corrosion.

#### Fitting of linear recirculating roller bearings

This process can be omitted where completely assembled adapters are used (suffix KM).

- Press the datum side of the linear recirculating roller bearing ① against the locating face of the adapter (arrows, Figure 26).
- Insert the fixing screws 2 and tighten finger tight (Figure 26).
- Tighten the fixing screws @ in three stages (see Fitting of adapters) to the specified tightening torque M<sub>A</sub> (Figure 26).

#### Fitting of adapters

- Press the adapter ③ against the locating faces on the adjacent construction (arrows, Figure 27).
- Insert the fixing screws ④ in the holes in the adapter and tighten finger tight (Figure 27).
- Tighten the fixing screws ④ in three stages to the specified tightening torque M<sub>A</sub> (Figure 27)
  - Stage 1 40% of M<sub>A</sub>
  - Stage 2 70% of M<sub>A</sub>
  - Stage 3 100% of M<sub>A</sub>



Figure 26 · Fitting of linear recirculating roller bearings



Figure 27 · Fitting of adapters







## Linear recirculating roller guidance systems

INA linear recirculating roller bearings

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#### Linear recirculating roller guidance systems

- consist of:
  - INA linear recirculating roller bearings
  - HYDREL guideways
  - HYDREL adapters.
- are suitable for locating/locating and locating/non-locating bearing arrangements
- can in a closed arrangement support loads from all directions and moments about all axes (*Design of bearing arrangements*, page 161)
- have a very high load carrying capacity while requiring little space
- have extremely high rigidity under preload
- can be preloaded by means of adjusting gibs. These give particularly uniform distribution of the preload over the whole length of the linear recirculating roller bearing and allow the preload to be set very easily
- have high accuracy throughout their operating life
- have low, uniform friction
- can be lubricated with oil or grease.

#### Linear recirculating roller bearings RUS, RUS..KS

- run particularly smoothly and with very little noise
- are sealed on their end faces
- can be lubricated through the end piece in the case of RUS..KS only.

#### Linear recirculating roller bearings PR

- have a full complement cylindrical roller set
- are suitable for high temperatures
- are suitable for high speeds and accelerations.

#### Linear recirculating roller bearings RUSW

- are particularly suitable, due to their arrangement of two rows of recirculating rollers at 90° to each other, for closed arrangements (see *Design of bearing arrangements*, page 161)
- are sealed on all sides.

#### Linear recirculating roller guidance system



- basic design comprising:
- linear recirculating roller bearing RUS.., RUS..KS
- HYDREL guideway UG
- metric mounting dimensions

Linear recirculating roller bearings





cylindrical rollers kept apart by spacers – rollers guided between the ribs in the saddle plate elastic, interchangeable wipers on the end pieces metric mounting dimensions RUS..KS with relubrication facility via the end pieces

for operating temperatures from -30 °C to +90 °C









## Linear recirculating roller guidance systems

Accessories for INA linear recirculating roller bearings

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#### Adjusting gibs

- are used where:
- the guidance system must have a precisely defined preload
- increase the rigidity of the linear recirculating roller guidance system by preload
- give particularly uniform distribution of the preload over the whole length of the linear recirculating roller bearing
- are easy to fit
  - the adjusting gibs are screw mounted on the linear recirculating roller bearings and the adjacent construction
  - the preload is set by means of an adjusting screw and secured by means of a locking screw
- can be easily maintained
  - integral ducts feed the lubricant into the return zone of the linear recirculating roller bearings.
  - treat the sliding surfaces with oil or grease in order to reduce the friction.

#### Setting devices

- are used where the deformation of the adjacent construction under preload forces is to be measured
  - this deformation, when added to the deflection of the linear recirculating roller bearing under preload (see *Rigidity*, page 136), gives the required preload dimension
- have a setting block that has the same dimensions as the linear recirculating roller bearing to be fitted
  - the setting block is fitted in place of the linear roller bearing. It can be connected via the pressure pipe and an intermediate distribution block with manometer to a conventional grease gun (*Measuring with setting device EUS and setting the preload*, page 134).
  - after measurement of deformation, the setting block is replaced by the linear recirculating roller bearing and the preload dimension determined is set by means of adjusting gibs or shims.



# EUS

Setting device



consisting of:

- setting block with two hydraulic pistons
- distribution block with manometerhigh-pressure rubber hose

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#### Design and safety guidelines

In order to achieve high guidance and positional accuracy and constant displacement resistance, note the fitting guidelines in the sections *Fitting* and *Design of bearing arrangements*!

If funnel type lubrication nipples to DIN 3405–AM6 are replaced by tube or hose connectors, the maximum screw length cannot be more than 6 mm.

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#### Sorting and tolerances

Accuracy

Linear recirculating roller bearings are sorted by height into 5 groups (Table 1). If no group is indicated in the order, group GR 3 is supplied in preference.

Linear recirculating roller bearings have tolerances according to Table 2, adjusting gibs have tolerances according to Table 3.

Where two or more linear recirculating roller bearings are arranged in parallel or in series and particularly uniform load distribution is required, linear recirculating roller bearings can be supplied sorted within the same group tolerance (Table 4).

Linear recirculating roller bearings sorted together must be indicated in the order by stating the number of sorted bearings and the suffix S.



### Special designs

Available by agreement:

linear recirculating roller bearings PR with wipers on the end faces

- suffix PP.

linear recirculating roller bearings PR with a lubrication nipple on the end face

- suffix KS.

### Table 1 · Sorting of linear recirculating roller bearings – groups

Group Colour of		Tolerance for H		
	packaging sticker	RUS 19 069 to RUS 38 206	RUS 65 210 RUS 85 280	
		PR 14 032 to PR 14 089	PR 14 135, PR 14 182	
		Group tolerance	Group tolerance	
		5 μm	10 µm	
GR1	red	0/ -5	0 / –10	
GR2	blue	-5 / -10	0 / -10	
GR3 <sup>1)</sup>	yellow	–10 / –15	-10 / -20	
GR4	green	–15 / –20	-20 / -30	
GR5	black	-20 / -25	-20 / -30	

1) Preferred series.

#### Table 2 · Tolerances of linear recirculating roller bearings

Linear recirculating	Tolerance		
roller bearing	Height µm	Width µm	Parallelism/ flatness µm
RUS 19 069 to RUS 38 206 PR 14 032 to PR 14 089	-25	-100	2
RUS 65 210 to RUS 85 280 PR 14 135 to PR 14 182	-30	-100	4

#### Table 3 · Tolerances of adjusting gibs

Adjusting gib	Tolerance Parallelism/flatness µm
VUS 19 069 to VUS 38 134 VUSZ 12 044 to VUSZ 24 084	3
RUS 65 210 to RUS 85 280 PR 14 135 to PR 14 182	8

#### Table 4 · Half group tolerance and width tolerance

Series	Group tolerance	Width tolerance
Designation	Η μm	Α <sub>4</sub> μm
RUS 19069 to RUS 38 206	2,5	-
RUS 65 210 to RUS 85 280	5	-
RUSW 4020	5	5





#### Solution Ordering example and ordering designation

Linear recirculating roller bearings RUS/RUS..KS Linear recirculating roller bearing RUS 26 102 Height sorting of dimension H  $-10 \mu m$  to  $-15 \mu m$ . Linear recirculating roller bearing RUS 38 134 KS Height sorting of dimension H  $-10 \mu m$  to  $-15 \mu m$ . Sorting to group 3 according to Table 1, page 177. Ordering designation: RUS 26 102 GR 3 (Figure 1) RUS 38 134 KS GR 3 (Figure 1).

#### Linear recirculating roller bearings sorted together

For eight linear recirculating roller bearings RUS 26 102 sorted together, the bearings are supplied with the restricted height tolerance 2,5  $\mu m.$ 

When ordering, state only the total quantity and not the number of sets, e.g.:

8 off RUS 26 102 8S.

#### Linear recirculating roller bearings PR

Linear recirculating roller bearing PR 14 061 Height sorting of dimension H GR 3. Linear recirculating roller bearing PR 14 135 Height sorting of dimension H GR 3. Sorting to group 3 according to Table 1, page 177. Ordering designation: PR 14 061 GR 3 (Figure 2) PR 14 135 GR 3 (Figure 2).



Figure 1 · Ordering example, ordering designation – RUS/RUS..KS



Figure 2 · Ordering example, ordering designation – PR



#### Adjusting gib

Adjusting gib for linear recirculating roller bearing RUS 26 102, metric dimensions. Ordering designation: VUS 26 102 (Figure 3)



Figure 3 · Ordering example, ordering designation – adjusting gib VUS



Figure 4 · Ordering example, ordering designation – setting device

#### Setting device

One setting block for linear recirculating roller bearing RUS 26102 and two setting blocks for linear recirculating roller bearings 19069

Distributor with manometer VBM 01 (article no. 126 361-7) High-pressure rubber hose HDS 01/250 (article no. 126 358-7)

Ordering designation: 1 off EUS 26 (article no. 126 352-8) 2 off EUS 19 (article no. 126 350-1) 1 off VBM 01 (article no. 126 361-7) 2 off HDS 01/250 (article no. 126 358-7) (Figure 4)
# INA linear recirculating roller bearings

Series RUS RUS..KS



RUS 19069 to RUS 38206

Dimension table · Dimensions in mm											
Designation		Mass	Dimensio	ons		Mounting	g dimensic	ns			
			А	C <sub>8</sub>	Н	A <sub>2</sub>	С	C <sub>2</sub>	C <sub>3</sub>	C <sub>1</sub>	C <sub>9</sub> <sup>2)</sup>
		ka		~		±0.1		±0.1			
RUS 19069	-	0,16	27	75	19	20,6	69	25,5	-	44	50
-	RUS 19069 KS	0,19	27	-	19	20,6	-	25,5	76	44	50
RUS 19 105	-	0,27	27	111	19	20,6	105	50	-	78,5	85
-	RUS 19105 KS	0,29	27	-	19	20,6	-	50	112	78,5	85
RUS 26 086	-	0,41	40	92	26	30	86	28	-	53	63
-	RUS 26 086 KS	0,49	40	-	26	30	-	28	92	53	63
RUS 26 102	-	0,53	40	108	26	30	102	44	-	69	79
-	RUS 26 102 KS	0,61	40	-	26	30	-	44	108	69	79
RUS 26 126	-	0,7	40	132	26	30	126	68	-	93	103
-	RUS 26 126 KS	0,78	40	-	26	30	-	68	132	93	103
RUS 38134	-	1,27	52	133	38	41	133	51	-	85	100
-	RUS 38134 KS	1,53	52	-	38	41	-	51	136	85	100
RUS 38 206	-	2,28	52	206	38	41	206	102	-	158	172
-	RUS 38 206 KS	2,53	52	-	38	41	-	102	209	158	172
RUS 65 210	-	7,5	76	234	65	62	211	76	-	134	-
		17	104	202	0E	0.0 5	201	101 E		105	
RUS 85 280 <sup>9</sup>	-	10	104	303	80	82,5	281	101,5	-	185	-

1) If the lubrication nipple is replaced by a tube or hose connector, the maximum screw length cannot be more than 6 mm.

2) Minimum support length.

<sup>3)</sup> UG guideway for RUS 85 280 available by agreement.

4) Available by agreement.

<sup>5)</sup> Hole connectors on both sides for optional funnel type lubrication nipples to DIN 3405-D6 (supplied loose) or tube or hose connectors. The holes should be closed off using lubrication nipples if a lubrication feed connector is not used.



<sup>6)</sup> Adjusting gibs, page 186.

7) Guideways from page 196.







Suitable	Suitable	K <sub>3</sub> <sup>8)</sup> to	ratings	Basic load	Funnel type								
guideway <sup>7)</sup>	adjusting gib <sup>6)</sup>	DIN 912-12.9	stat. C <sub>0</sub> N	dyn. C N	lubrication nipple <sup>1)</sup>	t	L <sub>W</sub>	K <sub>2</sub> <sup>8)</sup> to DIN 912- 12.9	h <sub>4</sub>	H <sub>3</sub>	H <sub>2</sub>	H <sub>1</sub>	
UG 6628	VUS 19 069	M3	33 000	42 000		6	10	M4	15,5	-	-	0,2	
UV 5323	VUS 19 069	M3	33 000	42 000	DIN 3 405-AM6	6	10	M4	15,5	10	-	0,2	
UFK 3210	VUS 19 105	M3	61 000	68 000		6	10	M4	15,5	-	-	0,2	
UFB 4710	VUS 19 105	M3	61 000	68 000	DIN 3 405-AM6	6	10	M4	15,5	10	-	0,2	
UG 9741	VUS 26 086	M4	56 000	76 000		10	14	M6	21	-	-	0,2	
UV 7532	VUS 26 086	M4	56 000	76 000	DIN 3 405-AM6	10	14	M6	21	13,5	-	0,2	
UFK 4710	VUS 26 102	M4	75 000	95 000		10	14	M6	21	-	-	0,2	
UFB 6412	VUS 26 102	M4	75 000	95 000	DIN 3 405-AM6	10	14	M6	21	13,5	-	0,2	
1	VUS 26 126	M4	103 000	122 000		10	14	M6	21	-	-	0,2	
1	VUS 26 126	M4	103 000	122 000	DIN 3 405-AM6	10	14	M6	21	13,5	-	0,2	
UG 12 553	VUS 38 134	M6	133 000	179000		14	20	M8	31	-	-	0,2	
UV 9542	VUS 38 134	M6	133 000	179000	DIN 3 405-AM6	14	20	M8	31	19,5	-	0,2	
UFK 6412	VUS 38 206	M6	265 000	305 000		14	20	M8	31	-	-	0,2	
UFB 7812	VUS 38 206	M6	265 000	305 000	DIN 3 405-AM6	14	20	M8	31	19,5	-	0,2	
UG 16 260 UV 13 863 UFA 8815 UFK 8815 UFB 10 615	VUS 65 210	M8	345 000	465 000	DIN 3 405-AM8×1 <sup>4)</sup>	22	30	M10	55	34	26	0,5	
UG <sup>3)</sup> UV 16 977 UFA 11 518 UFK 11 518 UFB 14 0185	VUS 85 280	M10	620000	840000	DIN 3 405-AM8×1 <sup>4)</sup>	30	40	M14	73	45	33	0,5	

Fixing screws							
Size	Tightening torque <sup>8)</sup> Nm max.						
M3	1,8						
M4	5						
M5	10						
M6	17						
M8	41						
M10	83						

8) The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.

Ensure that the adjacent construction has adequate strength! See VDI guideline 2230!



**INA** 181







## **INA** linear recirculating roller bearings

Series PR



Dimension table · Dimensions in mm													
Designation	Mass	Dimension	IS		Mounting	Mounting dimensions							
		А	С	Н	A <sub>2</sub>	C <sub>2</sub>	C <sub>1</sub>	C9	H <sub>1</sub>	h <sub>4</sub>	L <sub>W</sub>		
	kg				±0,1	±0,1			min.				
PR 14032	0,095	22,23	51	14,285	17,1	19	37,8	31	0,05	10	9		
PR 14044	0,2	25,4	69	19,05	20,6	25,5	54,5	42	0,05	14	10		
PR 14061	0,65	38,1	96	28,57	31	38	77,5	58,5	0,05	20,8	16		
PR 14089	1,75	50,8	142	38,1	41	51	121,5	90	0,05	28	20		
PR 14 135	5,65	76,2	198	57,15	62	76,2	158	126	0,05	42	30		
PR 14 182	13,25	101,6	264	76,2	82,5	101,6	211	167	0,05	56	40		

<sup>1)</sup> UG guideways for PR 14 182 available by agreement.

2) Delivery not ex stock. Delivery times on request.

<sup>3)</sup> Adjusting gibs, page 186.

<sup>4)</sup> Guideways from page 196.

5) The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.

Ensure that the adjacent construction has adequate strength! See VDI guideline 2230!

Fixing screws							
Size	Tightening torque <sup>5)</sup> Nm max.						
M2,5	1						
M3	1,8						
M4	5						
M5	10						
M6	17						
M8	41						





Basic load rating	S	K <sub>3</sub> <sup>5)</sup> to	Suitable adjusting gib <sup>3)</sup>	Suitable guideway <sup>4)</sup>				
dyn. C N	stat. C <sub>0</sub> N	DIN 912-12.9						
21 700	17 600	M2,5	-	UG 6628 UV 5323	UFA 3210 UFK 3210			
44 000	37 500	M3	VUSZ 12044	UG 6628 UV 5323	UFA 3210 UFK 3210 UFB 4710			
107 000	86 000	M4	VUSZ 18059	UG 9741 UV 7532	UFA 4710 UFK 4710 UFB 6412			
205 000	171000	M5	VUSZ 24 084	UG 12553 UV 9542	UFA 6412 UFK 6412 UFB 7812			
435 000	345 000	M6	VUSZ 36 135 <sup>2)</sup>	UG 16260 UV 13863	UFA 8815 UFK 8815 UFB 10615			
790 000	620 000	M8	VUSZ 48 182 <sup>2)</sup>	UG <sup>1)</sup> UV 16 977	UFA 11518 UFK 11518 UFB 14018			



# HYDREL linear recirculating roller bearing as angled element

Series RUSW



RUSW

Dimension table · Dimensions in mm														
Designation	Mass	Dimensio	Dimensions			Mounting dimensions								
		А	C <sub>8</sub>	H <sup>1)</sup>	A <sub>1</sub>	A <sub>4</sub> <sup>1)</sup>	a <sub>1</sub>	a <sub>3</sub>	С	C <sub>2</sub>	C <sub>3</sub>	C <sub>1</sub>		
	kg					±0,05				±0,1	±0,1			
RUSW 4020	0,47	42	97	60	20	10	13	13	92	50	25	64		

 $^{1)}$  If a set RUSW 4020 is ordered with the suffix .S, the dimensions H and A\_4 are supplied within a tolerance of 5  $\mu m.$ 

<sup>2)</sup> Guideways from page 54.

<sup>3)</sup> The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.
 <u>A</u> Ensure that the adjacent construction has adequate strength!
 <u>See VDI guideline 2230!</u>

Fixing screws							
Size	Tightening torque <sup>3)</sup> Nm max.						
M6	17						



						-		
						Basic load ratin	gs	Suitable guideway <sup>2)</sup>
H <sub>2</sub>	H <sub>3</sub>	H <sub>4</sub> ±0.1	H <sub>5</sub> ±0.1	K <sub>3</sub> <sup>3)</sup> to DIN 912-12.9	L <sub>W</sub>	dyn. C N	stat. C <sub>0</sub> N	
21,5	17,5	8	15	M6	8	50 000	125 000	V 4020





# Adjusting gibs

Series VUS metric

Series VUSZ inch



VUS

Dimension tab	Dimension table · Dimensions in mm														
Designation	Mass	Dimensio	Dimensions			Mounting dimensions									
		А	С	Н	A <sub>2</sub>	A <sub>3</sub>	A <sub>4</sub>	C <sub>10</sub>	C <sub>2</sub>	C <sub>11</sub>	C <sub>12</sub>	C <sub>13</sub>	C <sub>14</sub>		
	kg		max.	min.	±0,1			max.	±0,1			max.			
VUS 19069	0,24	26,5	78	16	20,6	14,9	-	73	25,5	62	16,5	7	16,5		
VUS 19105	0,32	26,5	123	16	20,6	14,9	-	119	50	100	25	15	29		
VUS 26 086	0,6	39,5	97	25	30	20,5	-	89	28	75	20,5	8	19,5		
VUS 26 102	0,71	39,5	113	25	30	20,5	-	105	44	91	20,5	8	27,5		
VUS 26 126	0,9	39,5	137	25	30	20,5	-	129	68	115	20,5	8	39,5		
VUS 38134	1,47	51,5	141	30	41	28,25	-	131	51	115	28	8	30,5		
VUS 38 206	2,1	51,5	250	25	41	28,25	-	240	102	200	49	30	61		
VUS 65 210 <sup>2)</sup>	4,7	75	234	38	62	30,9	21,6	220	76	200	62	10	40,5		
VUS 85 280 <sup>2)</sup>	8,8	100	314	38	82,5	41,25	25	300	101,5	280	89	10	53,5		

Dimension table · Dimensions in mm															
Designation	Mass	Dimensions			Mounting	Mounting dimensions									
		А	С	Н	A <sub>2</sub>	A <sub>3</sub>	A <sub>4</sub>	C <sub>1</sub>	C <sub>2</sub>	C <sub>11</sub>	C <sub>12</sub>	C <sub>13</sub>	C <sub>14</sub>		
	kg		max.	min.	±0,1			max.	±0,1			max.			
VUSZ 12044	0,19	25	78	16	19 <sup>1)</sup>	14,2	-	73	25,5	62	16,5	7	16,5		
VUSZ 18059	0,63	37,6	107	25	31	22,3	-	99	38	85	20,5	8	20		
VUSZ 24084	1,38	50	141	30	41	28,5	-	131	51	115	28	8	30,5		
VUSZ 36 135 <sup>2)</sup>	4,7	75	234	38	62	30,9	21,6	220	76,2	200	62	10	40,5		
VUSZ 48 182 <sup>2)</sup>	8,8	100	314	38	82,5	41,25	25	300	101,5	280	89	10	53,5		

 Distance between fixing holes in the upper gib half, differing from the lower gib half.

<sup>2)</sup> Delivery not ex stock. Delivery times on request.

<sup>3)</sup> Linear recirculating roller bearings, page 180 and page 182.

4) Through lubrication hole.

- <sup>5)</sup> The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.
   A Ensure that the adjacent construction has adequate strength!
   See VDI guideline 2230!
- <sup>6)</sup> Depending on the size, screws to DIN 912 or set screws to DIN 913 are used.

Fixing screws							
Size	Tightening torque <sup>5)</sup> Nm max.						
M3	1,8						
M4	5						
M5	10						
M6	17						
M8	41						
M10	83						
M14	220						





View X

					K <sub>3</sub> <sup>5)</sup> to DIN 912-12.9	Feed		Adjusting screw	Locking and extraction screw <sup>6)</sup>	Suitable linear recirculating roller	
	D <sub>1</sub>	D <sub>2</sub>	h <sub>4</sub>	K <sub>2</sub> <sup>5)</sup> to DIN 912-12.9		ΔH max <sub>.</sub>	per screw revolution	A/F <sub>1</sub>	A/F <sub>2</sub>	bearing <sup>3)</sup>	
	3,5	12	4	M4	M3	0,35	0,035	3	2	RUS 19069	
	3,5	12	3,5	M4	M3	0,5	0,023	3	2,5	RUS 19105	
	5	16	6	M6	M4	0,4	0,05	6	3	RUS 26 086	
	5	16	6	M6	M4	0,4	0,05	6	3	RUS 26 102	
	5	16	6	M6	M4	0,4	0,05	6	3	RUS 26 126	
	5	22	7	M8	M6	0,4	0,062	8	4	RUS 38134	
	5	22	5	-	M6	1	0,05	8	5	RUS 38 206	
	8	8	7	M10	M8	0,5	0,075	12	5	RUS 65 210	
	8	10	6	M14	M10	0,5	0,075	12	4	RUS 85 280	

			K <sub>3</sub> <sup>5)</sup> to DIN 912-12.9	Feed		Adjusting screw	Locking and extraction screw <sup>6)</sup>	Suitable linear recirculating roller	
D <sub>1</sub>	D <sub>2</sub>	h <sub>4</sub>		ΔH max <sub>.</sub>	IH         per screw         A/F1         A/F2           nax         ax         ax         ax         ax         by         by		A/F <sub>2</sub>	bearing <sup>e,</sup>	
3,5	12	4	M3	0,35	0,035	3	2	PR 14044	
5	16	6	M4	0,4	0,05	6	3	PR 14061	
5	22	7	M5	0,4	0,062	8	4	PR 14089	
8	8	7	M6	0,5	0,075	12	5	PR 14135	
8	10	6	M8	0,5	0,075	12	4	PR 14182	



View X VUS 65 210, VUSZ 36 135, VUS 85 280, VUSZ 48 182





# Setting device

Series EUS



E	1.1	
L	U	

Dimension ta	ble · D	imension	s in mr	n		-								
Setting block	for linear		Dimer	nsions		Mountin	ng dime	nsions				Article no.	High-pressure	Distributor
	recircu roller h	ulating Dearing <sup>1)</sup>	A <sub>K</sub>	С	Н	А	A <sub>1</sub>	C <sub>12</sub>	H <sub>2</sub>	G	K <sub>4</sub>		rubber hose HDS 01/	with
		Joannig	cm <sup>2</sup>		max.									gauge VBM
EUS 19	RUS	19069	5	72	19,5	25,4	20,6	28	18	M3×20	R <sup>1</sup> /8"	126 350-1	I = 250	01 Article no. 126 361-7
	RUS	19 105	5	72	19,5	25,4	20,6	28	18	M3×20	R <sup>1</sup> /8"	126 350-1	01/250 Article no	
	RUSZ	12044	5	72	19,5	25,4	20,6	28	18	M3×20	R <sup>1</sup> /8"	126 350-1	126 358-7	
	PR	14044	5	72	19,5	25,4	20,6	28	18	M3×20	R <sup>1</sup> /8"	126 350-1		
EUS 26	RUS	26 086	10	86	28	38	30	33	25	M4×30	R <sup>1</sup> /8"	126 352-8	l = 400 01/400 Article po	
	RUS	26 102	10	86	28	38	30	33	25	M4×30	R <sup>1</sup> /8"	126 352-8		
EUS 14061	RUSZ	18 059	10	85	30	38	31	33	27,5	M4×30	R <sup>1</sup> /8"	152 704-5	126 359-5	
	PR	14061	10	85	30	38	31	33	27,5	M4×30	R <sup>1</sup> /8"	152 704-5		
EUS 26 126	RUS	26 126	15	115	28	38	30	33	25	M4×30	R <sup>1</sup> /8"	126 353-6	I = 1000	
EUS 38	RUS	38 1 34	20	115	40	50,8	41	44	36	M6×40	R <sup>1</sup> /8"	126 354-4	01/1000 Article no	
	RUSV	24 084	20	115	40	50,8	41	44	36	M6×40	R <sup>1</sup> /8"	126 354-4	126 360-9	
	PR	14 089	20	115	40	50,8	41	44	36	M6×40	R <sup>1</sup> /8"	126 354-4		
EUS 38206	RUS	38 206	30	200	40	50,8	41	59	36	M6×40	R <sup>1</sup> /8"	126355-2		
EUS 65	RUS	65 210	60	200	70	75	62	37	60	M8 ×70	R <sup>1</sup> /8"	126 356-0		
EUS 85	RUS	85 280	100	250	90	100	82,5	89	80	M10×90	R <sup>1</sup> /8"	126 357-0		

 $^{1)}$  Linear recirculating roller bearings, page 180 and page 182.



# Linear recirculating roller guidance systems

HYDREL guideways

A	F	Page
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	Design and safety guidelines	192
► <b>⊘</b> +	Accuracy	193
Sp.	Special designs	194
AAAAA	Ordering example and ordering designation	195
	Dimension tables	196



#### HYDREL guideways

- are high precision elements that can be combined with INA linear recirculating roller bearings to give high precision linear recirculating guidance systems
- are made from through hardened tool steel (670 +170 HV)
- have precision ground raceways R<sub>a</sub>0,4 (R<sub>7</sub>2)
- are supplied as single-piece elements up to the maximum length (*dimension table*)
  - longer guideways are made up from matched and marked sections.

#### Guideways UG, UGN, UGS, UGSN

- have four raceways for linear recirculating roller bearings
- support forces in the main load direction and opposing direction with a counterstay as well as lateral forces in two directions (*Design examples*, page 159)
  - UGN, UGSN have a continuous slot for a square steel bar to support high lateral forces in two directions.

#### Guideways UZ, UZN, UZS, UZSN

- have three raceways for linear recirculating roller bearings
- the upper raceway is arranged between the holes for the fixing screws.
- support forces in the main load direction and lateral forces in two directions (*Design examples*, page 159)
  - UZN, UZSN have a continuous slot for a square steel bar to support high lateral forces in two directions.

#### Guideways UV

- have two raceways arranged at an angle of 45° to the mounting surface
- support forces in the main load direction and lateral forces in two directions (*Design examples*, page 159)
- allow simple adjacent construction designs when combined with HYDREL adapters.

#### Guideways UFA, UFB, UFK

- have only one raceway
- allow a low guidance height due to the low section height of the guideway
- support forces in the main load direction only (*Design examples*, page 159)
- are located by adhesive bonding (UFA), screw mounting (UFB) or a clamping strip in a slot (UFK).





rectangular cross-section with four raceways, offset hole pattern

through holes with cylindrical counterbores for fixing screws to DIN 912

UGN has a milled continuous slot for a square steel bar to DIN 178 in order to support high lateral forces





rectangular cross-section with three raceways, parallel hole pattern

threaded blind holes

UZSN has a milled continuous slot for a square steel bar to DIN 178 in order to support high lateral forces



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#### Linear recirculating roller guidance systems

HYDREL guideways



#### Design and safety guidelines

The running accuracy of the linear recirculating roller guidance system is determined to a significant extent by the accuracy and alignment of the guideways!

For guidance systems with a length greater than  $L_{max}$ , the guideway assembly is composed of several sections. Precise production and matching allow consistent accuracy and smooth running over the whole traverse distance.

The parts of multi-piece guideways are matched to each other, packed together and form a set. All parts of the same set have the same set number. The joints are additionally marked by means of consecutive letters (Design of bearing arrangements, page 148; Fitting, page 162).

Further	information	Page
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	Rigidity	136
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	Sealing, operating limits	147
	Design of bearing arrangements	148
	Fitting	162



Figure 1 · Quality and parallelism of HYDREL guideways



### 

#### HYDREL guideways

Guideways are available in three quality grades (Figure 1):

- Q2
  - For exacting requirements in high precision machines. This grade should only be used if the adjacent construction is of correspondingly high accuracy.
- 🛛 Q6
  - For precision guidance systems in machine tool design. Standard grade for arrangements with counterstay.
- Q10

For requirements in general machine building.

The guideways have length tolerances according to Table 1.

#### Table 1 · Length tolerances of HYDREL guideways

Guideway	Length tolerances of guideways						
Designation	Single-piece guideways	Multi-piece guideways					
UG, UZ, UV UFA, UFB, UFK	L +2‰	L <sub>tot</sub> ±2 mm					

The positional tolerance of the hole pattern  $\emptyset$  X is 1‰ of the guideway length L (Figure 2). Guideways up to the maximum length according to Table 2 can be fitted to a predrilled hole pattern if a positional tolerance of  $\emptyset$  0,2 can be achieved for the hole pattern.

If longer guideways are to fulfil this requirement, they must be ordered using the suffix P (positional tolerance).

# Table 2 · Maximum length of guideways for fitting to a predrilled hole pattern

Fixing screw	Guideway length
Size	max. mm
M4 / M5	1 000
M6	1 200
M8 / M12	1 700
M10 / M16	2 200



Figure 2 · Hole pattern of guideways



#### Linear recirculating roller guidance systems

HYDREL guideways

#### Multi-piece guideways

For multi-piece guideways, it is only necessary to state the total length. Further sorting information is not necessary.

The hole pitches over the joints are identical to those on singlepiece guideways (dimension  $C_4$ , *dimension table*).

Example:

■ 2 off UGN 9741×3600 Q6 each comprising two sections.

#### Sorted guideways

Guideways are sorted together if two or more guideways of the same profile are fitted in the same plane adjacent to each other or in series.

The sorting affects the positional accuracy of the raceways in relation to the mounting surfaces. The guideways are, within the respective quality grade, sorted and marked according to height and/or width. Sorting of the guideways is indicated by the suffix S and the number of sorted guideways. Example:

■ 2 off UG 9741×2000 Q6 2S.

#### Guideways for non-locating bearing side

If lateral guidance is only fulfilled by one guideway in a design solution, it is not necessary to machine the lateral surfaces of the second guideway to the raceway accuracy.

This more economical design is indicated by the suffix H after the grade symbol.

If various designs are to be used in the case of sorted guideways (suffix X), the ordering data must be marked by means of the suffix X.

#### Example:

■ 5 off UG 9741×1600 Q6 2SX and 5 off UG 9741×1600 Q6H 2SX.

#### Guideways with grinding allowance

If guideways are overground after fitting, guideways with preground raceways are used (suffix VQ).

The grade indicates the geometrical accuracy and surface quality of the mounting and locating surfaces.

The grinding allowance of the raceways is up to 0,1 mm Example:

■ 2 off UFB 4710×1200 VQ.



#### Special designs

Available by agreement:

- guideways in accordance with customer drawing
- lateral surfaces of guideways not machined to the raceway accuracy
  - suffix H
- all surfaces of guideways with grinding allowance
   suffix VGS
- mounting surfaces of the guideways precision ground
   suffix VQ
- mirror image hole pattern to hole pattern L
  - suffix R.



#### C Ordering example and ordering designation

12 guideways for linear recirculating roller bearings RUS 26 126 Length 1000 mm (symmetrical hole pattern) Sorted in matched pairs for 6 machines Profile size UG 9741 Quality grade Q6. Ordering designation: 12 off UG 9741×1000 Q6 2S (Figure 3).

#### Sorted guideways

Guideways for linear recirculating roller bearings RUS 19 105 Length 1600 mm Profile size UG 6628 (hole pattern on left side) Profile size UZ 6628 Quality grade Q6. Sorted for 10 machines Ordering designation: 10 off UG 6628×1600 L Q6 2SX 10 off UZ 6628×1600 Q6 2SX (Figure 4).

#### Multi-piece guideways

1 guideway for linear recirculating roller bearings RUS 19 069 Length 3200 mm (in two pieces) Profile size UV 5323 Quality grade Q10. Ordering designation: 1 off UV 5323×3200 Q10 (two pieces) (Figure 5).











Figure 5 · Ordering example, ordering designation – UV

# **HYDREL** guideways

Series UG UGN UGS UGSN Standard grade Q6



UG, UGN (view X)

Dimension table · Dimensions in mm											
Designation	Dimension	S		Mounting dimensions							
	А	Н	L <sup>1)</sup>	A <sub>1</sub>	A <sub>2</sub>	C <sub>4</sub>	C <sub>5</sub> /C <sub>6</sub>	C <sub>5</sub> /C <sub>6</sub>	Ca	h <sub>4</sub>	
	-0,1	-0,1	max.				min	max	max.		
UG 6628	66	28	2000	12	18	40	15	25	28	17	
UGN 6628	66	28	2000	12	18	40	15	25	28	17	
UGS 6628	66	28	2000	12	18	40	15	25	28	_	
UGSN 6628	66	28	3000	12	18	40	15	25	28	-	
UG 9741	97	41	3000	15	30	40	15	25	41	28	
UGN 9741 A	97	41	3000	15	30	40	15	25	41	28	
UGS 9741	97	41	3000	15	30	40	15	25	41	-	
UGSN 9741 A	97	41	3000	15	30	40	15	25	41	-	
UG 12 553	125	53	3000	18	35	40	15	25	53	38	
UGN 12553 A	125	53	3000	18	35	40	15	25	53	38	
UGS 12 553	125	53	3000	18	35	40	15	25	53	-	
UGSN 12553 A	125	53	3000	18	35	40	15	25	53	-	
UG 16 260	162	60	3000	20	44	40	20	25	77	40	
UGN 16260 A	162	60	3000	20	44	40	20	25	77	40	
UGS 16 260	162	60	3000	20	44	40	20	25	77	-	
UGSN 16260 A	162	60	3000	20	44	40	20	25	77	-	

<sup>1)</sup>  $\overline{L}$  = available as single-piece guideways up to L<sub>max</sub>: note alternating hole pattern. Guideways longer than length L<sub>max</sub> are supplied as multi-piece guideways. The hole pattern of multi-piece guideways continues regularly over the joint.

<sup>2)</sup> The remaining gap is filled with castable resin after fitting.

<sup>3)</sup> Square steel bar to DIN 178 not included in delivery.

 $^{\rm 4)}$  Indicate hole pattern and end pieces C\_5/C\_6 when ordering.

<sup>5)</sup> Linear recirculating roller bearings, page 180 and page 182.

<sup>6)</sup> The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.

Ensure that the adjacent construction has adequate strength! See VDI guideline 2230!

Fixing screws							
Size	Tightening torque <sup>6)</sup> Nm max.						
M8	41						
M10	83						
M12	140						
M16	350						







Fixing holes		Slots <sup>2)</sup>				Square steel bar <sup>3)</sup>	Closing plugs	Suitable for linear recirculating roller		
K <sub>1</sub> <sup>6)</sup> to DIN 912-12.9	t	C <sub>a1</sub>	C <sub>a2</sub>	H <sub>1</sub>	Ha	DIN 178		bearing <sup>3)</sup>		
M8	-	-	-	-	-	-	KVK 15	RUS 19069	PR 14 032 PR 14 044	
M8	-	17,75	6,5	3,5	2,5	5× 5	KVK 15	RUS 19 105		
M8	16	-	-	-	-	-	-			
M8	16	17,75	6,5	3,5	2,5	5× 5	-			
M10	-	-	-	-	-	-	KVK 18,5	RUS 26 086	PR 14061	
M10	-	23,25	12	6,5	5	10×10	KVK 18,5	RUS 26 102		
M10	22	-	-	-	-	-	-	100 20 120		
M10	22	23,25	12	6,5	5	10×10	-			
M12	-	-	-	-	-	-	KVK 20	RUS 38 134	PR 14089	
M12	-	27	14	7,5	6	12×12	KVK 20	RUS 38 206		
M12	30	-	-	-	-	-	-			
M12	30	27	14	7,5	6	12×12	-			
M16	-	-	-	-	-	-	KA 26,5 M	RUS 65 210	PR 14 135	
M16	-	31,25	18	9,5	8	16×16	KA 26,5 M			
M16	34	-	-	-	-	_	_			
M16	34	31,25	18	9,5	8	16×16	-			

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172 931

h<sub>4</sub>









# **HYDREL** guideways

Series UZ UZN UZS UZSN Standard grade Q6



UZ, UZN (view X)

Dimension table · Dimensions in mm											
Designation	Dimensions			Mounting dimensions							
	А	Н	L <sup>1)</sup>	A <sub>1</sub>	A <sub>2</sub>	C <sub>4</sub>	C <sub>5</sub> /C <sub>6</sub>	C <sub>5</sub> /C <sub>6</sub>	h <sub>4</sub>		
	-0,1	-0,1	max.				min	max			
UZ 6628	66	28	2 000	11	44	80	15	65	17		
UZN 6628 A	66	28	2 000	11	44	80	15	65	17		
UZS 6628	66	28	2 000	11	44	80	15	65	-		
UZSN 6628 A	66	28	2 000	11	44	80	15	65	-		
UZ 9741	97	41	2 960	15	67	80	15	65	28		
UZN 9741 A	97	41	2 960	15	67	80	15	65	28		
UZS 9741	97	41	2 960	15	67	80	15	65	-		
UZSN 9741 A	97	41	2 960	15	67	80	15	65	-		
UZ 12553	125	53	2 960	18	89	80	15	65	38		
UZN 12553 A	125	53	2 960	18	89	80	15	65	38		
UZS 12 553	125	53	2 960	18	89	80	15	65	-		
UZSN 12553 A	125	53	2 960	18	89	80	15	65	-		
UZ 16 260	162	60	2 960	26	110	80	20	65	40		
UZN 16260 A	162	60	2 960	26	110	80	20	65	40		
UZS 16 260	162	60	2 960	26	110	80	20	65	-		
UZSN 16260 A	162	60	2 960	26	110	80	20	65	-		

1)  $\overline{L}$  = available as single-piece guideways up to L<sub>max</sub>. Intermediate lengths available by agreement. Guideways longer than length L<sub>max</sub> are supplied as multi-piece guideways. The hole pattern of multi-piece guideways continues regularly over the joint.

<sup>2)</sup> The remaining gap is filled with castable resin after fitting.

<sup>3)</sup> Square steel bar to DIN 178 not included in delivery.

<sup>4)</sup> Linear recirculating roller bearings, page 180 and page 182.

<sup>5)</sup> The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.

Ensure that the adjacent construction has adequate strength! See VDI guideline 2230!

Fixing screws	
Size	Tightening torque <sup>5)</sup> Nm max.
M8	41
M10	83
M12	140
M16	350



UZS, UZSN (view X)





Fixing holes		Slots <sup>2)</sup>				Square steel bar <sup>3)</sup>	Closing plugs	Suitable for linear recirculating roller bearing <sup>4)</sup>		
K <sub>1</sub> <sup>5)</sup> to DIN 912-12.9	t	C <sub>a1</sub>	C <sub>a2</sub>	H <sub>1</sub>	Ha	DIN 178				
M8	-	-	-	-	-	-	KVK 15	RUS 19069	PR 14032	
M8	-	17,75	6,5	3,5	2,5	5× 5	KVK 15	RUS 19105	PR 14044	
M8	16	-	-	-	-	-	-			
M8	16	17,75	6,5	3,5	2,5	5× 5	-			
M10	-	-	-	-	-	-	KVK 18,5	RUS 26 086	PR 14061	
M10	-	23,25	12	6,5	5	10×10	KVK 18,5	RUS 26 102 RUS 26 126		
M10	22	-	-	-	-	-	-	100 20 120		
M10	22	23,25	12	6,5	5	10×10	-			
M12	-	-	-	-	-	-	KVK 20	RUS 38 134	PR 14089	
M12	-	27	14	7,5	6	12×12	KVK 20	RUS 38 206		
M12	30	-	-	-	-	-	-			
M12	30	27	14	7,5	6	12×12	-			
M16	-	-	-	-	-	-	KA 26,5 M	RUS 65 210	PR 14 135	
M16	-	37,25	18	9,5	8	16×16	KA 26,5 M			
M16	34	-	-	-	-	-	-			
M16	34	37,25	18	9,5	8	16×16	-			





# **HYDREL** guideways

Series UV Standard grade Q10





Dimension	Dimension table · Dimensions in mm															
Designation	Dimer	nsions			Mou	nting d	imen	sions			Fixing	g holes	Closing	Suitable for lin	ear	
	А	Н	h <sub>x</sub>	L1)	a <sub>1</sub>	a <sub>2</sub>	C <sub>4</sub>	$C_5/C_6$	$C_5/C_6$	Ha	h <sub>4</sub>	K <sub>1</sub> <sup>3)</sup>	plug	recirculating ro	oller bearing <sup>2)</sup>	
	-0,1	-0,1	-0,1	max.				min.	max.			to DIN 912-12.9				
UV 5323	53	23	29,5	2 000	13	28,3	80	10	70	3	15	M6	KVK 11,5	RUS 19 069 RUS 19 105	PR 14 032 PR 14 044	
UV 7532	75	32	40,5	2 000	17	41	80	15	65	3	21	M8	KVK 15	RUS 26 086 RUS 26 102 RUS 26 126	PR 14 061	
UV 9542	95	42	52	2 000	20	53	80	15	65	4,5	29	M10	KVK 18,5	RUS 38 134 RUS 38 206	PR 14 089	
UV 13 863	138	63,5	75	2 000	23	81	80	15	65	6	48,5	M12	KVK 20	RUS 65 210	PR 14135	
UV 16977	169	77,5	92	2 000	29	99	80	20	60	7,5	57,5	M16	KA 26,5 M	RUS 85 280	PR 14182	

L = available as single-piece guideways up to L<sub>max</sub>. Intermediate lengths available by agreement. Guideways longer than length L<sub>max</sub> are supplied as multi-piece guideways. The hole pattern of multi-piece guideways continues regularly over the joint.

<sup>2)</sup> Linear recirculating roller bearings, page 180 and page 182.

<sup>3)</sup> The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.

Ensure that the adjacent construction has adequate strength! See VDI guideline 2230!

Fixing screws	
Size	Tightening torque <sup>3)</sup> Nm max.
M6	17
M8	41
M10	83
M12	140
M16	340

# **HYDREL** guideways

Series UFA UFB UFK

Standard grade Q10



UFA, for adhesive bonding

Dimension table · Dimensions in mm																
Designation	Dimensions			Mounting dimensions			Fixing	) holes	6		Bond width	Suitable clam-	Suitable for linear recirculating roller bearing <sup>4)</sup>			
	A _0 1	H _0 1	L <sup>1)</sup>	C <sub>4</sub>	C <sub>5</sub> C <sub>6</sub> min	$C_5$ $C_6$ max	A <sub>1</sub>	A <sub>2</sub>	h <sub>4</sub>	K1 to DIN 912-8.8	А <sub>3</sub>	ping strip <sup>3)</sup>				
1154 2210	-0,1	10	2000			тпах.					0		DUE 10.040	DUC 10 10F	DD 14 022	
UFA 3210	32	10	2000	-	-	-	-	-	-	-	8	-	KUS 19 009	KUS 19 105	PR 14 032	PK 14 044
UFK 3210	32	10	2000	-	-	-	-	-	-	-	-	UKB 10				
UFA 4710	47	10	2000	-	-	-	-	-	-	-	12	-	RUS 26 086	RUS 26 102	RUS 26 126 PR 14 044 RUS 26 126	PR 14 061
UFB 4710	47	10	2000	80	10	70	5,5	36	4	M4	-	-	RUS 19 069	RUS 19 105		
UFK 4710	47	10	2000	-	-	-	-	-	-	-	-	UKB 10	10520000	105 20 102	105 20 120	
UFA 6412	64	12	2000	Ι	-	-	-	-	Ι	-	15	-	RUS 38 134	RUS 38 206	PR 14 089 RUS 26 126 PR 14 089	PR 14 061
UFB 6412	64	12	2000	80	10	70	6	52	5	M5	I	-	RUS 26 086 RUS 38 134	RUS 26 102 RUS 38 206		
UFK 6412	64	12	2000	Ι	-	-	-	-	Ι	-	I	UKB 12		100 00 200	111100/	
UFB 7812	78	12	2000	80	10	70	7	64	5	M5	-	-	RUS 38 134	RUS 38 206	PR 14 089	
UFA 8815	88	15	2960	Ι	-	-	-	-	Ι	-	18	-	RUS 65 210		PR 14 135	
UFK 8815	88	15	2960	-	-	-	-	-	-	-	-	UKB 14				
UFB 10 615	106	15	2960	80	10	70	8	90	7	M6	-	-	RUS 65 210		PR 14 135	
UFA 11 518	115	18	2960	-	-	-	-	-	-	-	-	-	RUS 85 280		PR 14 182	
UFK 11 518	115	18	2960	-	-	-	-	-	-	-	-	UKB 14				
UFB 14 018	140	18	2960	80	15	65	11	118	7	M8	-	-	RUS 85 280		RUS 14 182	

<sup>1)</sup>  $\overline{L}$  = available as single-piece guideways up to L<sub>max</sub>. Intermediate lengths available by agreement. Guideways longer than length L<sub>max</sub> are supplied as multi-piece guideways. The hole pattern of multi-piece guideways continues regularly over the joint.

 $^{2)}$  Slot width  $\mathsf{B}_1$  for clamping strip, see page 153, Table 5.

<sup>3)</sup> The clamping strip must be ordered separately.

<sup>4)</sup> Linear recirculating roller bearings, page 180 and page 182.

<sup>5)</sup> If anaerobic single-component adhesives are used, a reduced bonding gap of 0,02 mm should be used. Suffix KL 02.





UFB, for screw mounting



# Linear recirculating roller guidance systems

HYDREL adapters

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#### HYDREL adapters

- provide a simple means of installing the linear recirculating roller bearings in 45° arrangements
- are suitable for both open and closed arrangements
- can be provided with all-round sealing.

#### Adapters APUV

- are used for open arrangements and in conjunction with HYDREL UV guideways
- are predominantly used for loads in the main load direction
- can support additional lateral forces by means of a predrilled dowel hole.

#### Adapters AUV

- are used for closed arrangements and in conjunction with HYDREL UV guideways
- can also transmit high lateral forces due to the lateral screw connections.

#### Adapters AUVL

- are used for closed arrangements and in conjunction with HYDREL UV guideways
- have an adjusting gib that can be used to set the preload on the guidance system
- can also transmit high lateral forces due to the lateral screw connections.

#### Adapters AV

- are used for closed arrangements and in conjunction with HYDREL V guideways
- can also transmit high lateral forces due to the lateral screw connections (in the adapter and guideway).

#### Adapters AVL

- are used for closed arrangements and in conjunction with HYDREL V guideways
- have an adjusting gib that can be used to set the preload on the guidance system
- can also transmit high lateral forces due to the lateral screw connections (in the adapter and guideway).

#### Wipers

are end face and longitudinal wipers for HYDREL adapters.







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#### Linear recirculating roller guidance systems

HYDREL adapters



#### Design and safety guidelines

- The advantages of a preloaded guidance system should be utilised (see section *Preload*, page 132). Preload can be applied by means of adapters of series AUVL or AVL in closed arrangements (Figure 1).
- The adapters can be lubricated via the end piece in linear recirculating roller bearings of series RUS..KS.
- If adapters with end face seals are used, attention must be paid to the ordering designation
- The length of the fixing screws for linear roller bearings must not be exceeded (Table 1)
- Adapters must be fitted taking account of the set number.

## Table 1 · Maximum length of fixing screws for linear recirculating roller bearings in adapters

Adapter APUV, AUV, AUVL, AV, AVL	Fixing screw	Length max.
19	M3	22
26	M4	30
38	M6	40
14032	M2,5	16
14044	M3	20
14061	M4	30

#### Further information

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Page



Adapters of type US – suffix US – are manufactured with a tolerance of 0,02 for dimension  $A_3$  and are additionally sorted within 0,01 for dimension H. These are packed in units of two pieces and identified by a set number. With this sort, up to four adapters are used for the normal installation positions. For arrangements with more than four adapters, the arrangement must be stated when ordering.



#### Special designs

Adapters completely assembled with linear recirculating roller bearings and wipers if necessary – suffix KM... (Figure 2).





Figure 2 · Adapter completely assembled with linear recirculating roller bearings and wipers



#### Sector 2 Content of the sector

Adapter without wipers

Adapter APUV for linear recirculating roller bearing RUS 19 069 and guideway UV 5323.

Ordering designation: 1 off APUV 19069 (Figure 3).

#### Adapter with all-round sealing

Adapter with end face seals

Ordering designation: 1 off AUV 14061

and guideway UV 7532.

wipers.

Adapter AUVL for linear recirculating roller bearing RUS 38 206 KS and guideway UV 9542.

Adapter sealed on longitudinal and end faces, relubrication facility via the end pieces ① of the linear recirculating roller bearings and a screwed pipe connection @.

Adapter AUV for linear recirculating roller bearing PR 14061

Adapter sealed on end faces, no relubrication facility via the end pieces of the linear recirculating roller bearing, without lateral

2 off AK 7532-100 (Figure 5).

Ordering designation: 1 off AUVL 38 206 2 off AK 9542-021 2 off AL 38 206 (Figure 4).

# APUV 19069

Figure 3 · Ordering example, ordering designation – adapter APUV



Figure 4 · Ordering example, ordering designation – AUVL



Figure 5 · Ordering example, ordering designation – wiper



# **HYDREL** adapters

Series APUV AUV AUVL



APUV

Dimension table · Dimensions in mm												
Adapter	Linear recirculating roller	Dimension	IS				Mounting	dimensions				
	bearing	C <sub>1</sub> -0,4	А	H <sup>1)</sup> -0,2	A <sub>3</sub> -0,1	A <sub>4</sub> -0,1	H <sub>2</sub>	H <sub>1</sub>	C <sub>8</sub>			
APUV 19069 AUV 19069 AUVL 19069	RUS 19 069 RUS 19 069 KS	77	89	47 60 64	44,5	26,5	26,7 39,7 43,7	3	96			
APUV 19105 AUV 19105 AUVL 19105	RUS 19 105 RUS 19 105 KS	113	89	47 60 64	44,5	26,5	26,7 39,7 43,7	3	132			
APUV 14032 AUV 14032 AUVL 14032	PR 14032	77	89	40,33 53,33 57,33	44,5	26,5	26,7 39,7 43,7	3	96			
APUV 14044 AUV 14044 AUVL 14044	PR 14044	83	89	47,07 60,07 64,07	44,5	26,5	26,7 39,7 43,7	3	102			
APUV 26 086 AUV 26 086 AUVL 26 086	RUS 26 086 RUS 26 086 KS	94	119	67 79 83	59,5	37,5	42 54 58	3	116			
APUV 26102 AUV 26102 AUVL 26102	RUS 26 102 RUS 26 102 KS	110	119	67 79 83	59,5	37,5	42 54 58	3	132			
APUV 26126 AUV 26126 AUVL 26126	RUS 26 126 RUS 26 126 KS	134	119	67 79 83	59,5	37,5	42 54 58	3	156			
APUV 14061 AUV 14061 AUVL 14061	PR 14061	102	119	70,63 82,63 86,63	59,5	37,5	42 54 58	3	124			
APUV 38134 AUV 38134 AUVL 38134	RUS 38 134 RUS 38 134 KS	137	155	91 106 110	77,5	47,5	55,5 70,5 74,5	4,5	162			
APUV 38 206 AUV 38 206 AUVL 38 206	RUS 38 206 RUS 38 206 KS	210	155	91 106 110	77,5	47,5	55,5 70,5 74,5	4,5	235			
APUV 14089 AUV 14089 AUVL 14089	PR 14089	147	155	91,14 106,14 110,14	77,5	47,5	55,5 70,5 74,5	4,5	172			

<sup>1)</sup> The height adjustment of the gib from the central position is  $\pm 0.3$  mm.

<sup>2)</sup> For adapters AUV/AUVL 38206, only fixing screws of grade 10.9 can be used.

 $^{3)}$  Drilling out of diameter D<sub>1</sub> H7 for dowel pins to DIN 6 325 (tolerance m6).

4) Ordering example: see page 205.

<sup>5)</sup> 00 No relubrication facility via end piece of RUS. For use without lateral wipers.
 01 No relubrication facility via end piece of RUS. For use with lateral wipers.

10 Relubrication facility via end piece of linear recirculating roller bearing with lubrication nipple (RUS..KS). For use without lateral wipers.

11 Relubrication facility via end piece of linear recirculating roller bearing with lubrication nipple (RUS..KS). For use with lateral wipers. 20 Relubrication facility via end piece of linear recirculating roller bearing and screwed pipe connection, fitted to RUS..KS.

For use without lateral wipers.

21 Relubrication facility via end piece of linear recirculating roller bearing and screwed pipe connection, fitted to RUS.KS. For use with lateral wipers

<sup>6)</sup> The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation. Ensure that the adjacent construction has adequate strength! See VDI guideline 2230!





AUV

AUVL

Fixing holes					Guideway	Wipers	
K <sub>3</sub> <sup>2)6)</sup> to DIN 912-12.9	a <sub>4</sub>	H <sub>4</sub>	C <sub>2</sub>	D <sub>1</sub> <sup>3)</sup>		End face <sup>4)</sup>	Lateral <sup>4)</sup>
M8 M10 M10	7,8 78 78	- 12,6 16,6	44 43 43	10 - -	UV 5323	<b>AK 5323-0</b> <sup>5)</sup>	<b>AL 19069</b> <sup>5)</sup>
M8 M10 M10	7,8 78 78	- 12,6 16,6	85 80 80	16 - -	UV 5323	<b>AK 5323-0</b> <sup>5)</sup>	<b>AL 19 105</b> <sup>5)</sup>
M8 M10 M10	7,8 78 78	- 12,6 16,6	44 43 43	10 - -	UV 5323	<b>АК 5323-1</b> <sup>5)</sup>	<b>AL 14 032</b> <sup>5)</sup>
M8 M10 M10	7,8 78 78	- 12,6 16,6	44 43 43	10 - -	UV 5323	<b>AK 5323-1</b> <sup>5)</sup>	<b>AL 14 044</b> <sup>5)</sup>
M10 M12 M12	15,5 106 106	- 16,3 20,3	62 66 66	16 - -	UV 7532	<b>AK 7532-0</b> <sup>5)</sup>	<b>AL 26 086</b> <sup>5)</sup>
M10 M12 M12	15,5 106 106	- 16,3 20,3	62 66 66	16 - -	UV 7532	<b>AK 7532-0</b> <sup>5)</sup>	<b>AL 26 102</b> <sup>5)</sup>
M10 M12 M12	15,5 106 106	- 16,3 20,3	98 84 84	16 - -	UV 7532	<b>AK 7532-0</b> <sup>5)</sup>	<b>AL 26 126</b> <sup>5)</sup>
M10 M12 M12	15,5 106 106	- 16,3 20,3	62 66 66	16 - -	UV 7532	<b>AK 7532-1</b> <sup>5)</sup>	<b>AL 14061</b> <sup>5)</sup>
M12 M16 M16	25 138 138	- 21,4 25,4	105 90 90	16 - -	UV 9542	<b>AK 9542-0</b> <sup>5)</sup>	<b>AL 38 134</b> <sup>5)</sup>
M12 M16 M16	25 138 138	- 21,4 25,4	168 140 140	14 - -	UV 9542	<b>AK 9542-0</b> <sup>5)</sup>	<b>AL 38 206</b> <sup>5)</sup>
M12 M16 M16	25 138 138	- 21,4 25,4	105 90 90	20 - -	UV 9542	<b>AK 9542-1</b> <sup>5)</sup>	AL 14089 <sup>5)</sup>

Fixing screws										
Size	Tightening torque <sup>6)</sup> Nm max.									
M8	41									
M10	83									
M12	140									
M16	350									



# **HYDREL** adapters

Series AV AVL



Dimens	Dimension table · Dimensions in mm													
Adapter		Linear recirculating	Dimensio	ons				Mounting	g dimensi	ons				
		roller bearing	C <sub>1</sub> -0,4	A H <sup>1)</sup> -0,2			A <sub>3</sub> -0,1	A <sub>4</sub> -0,1	A <sub>4</sub> -0,1		H <sub>1</sub>		C <sub>8</sub>	
					V 7040	V 8050		V 7040	V 8050		V 7040	V 8050		
AV AVL	19069 19069	RUS 19069 RUS 19069 KS	77	89	77,5 81,5	84,5 88,5	44,5	20	25	39,7 43,7	24	26	96	
AV AVL	19 105 19 105	RUS 19105 RUS 19105 KS	113	89	77,5 81,5	84,5 88,5	44,5	20	25	39,7 43,7	24	26	132	
AV AVL	14032 14032	PR 14032	77	89	70,9 74,9	77,9 81,9	44,5	20	25	39,7 43,7	24	26	96	
AV AVL	14 044 14 044	PR 14044	83	89	77,6 81,6	84,6 88,6	44,5	20	25	39,7 43,7	24	26	102	

<sup>1)</sup> The height adjustment of the gib from the central position is  $\pm 0.3$  mm.

<sup>2)</sup> Complete ordering designation: see page 205.

<sup>3)</sup> No wipers can be fitted in conjunction with guideway V 7040.

<sup>4)</sup> 00 No relubrication facility via end piece of RUS. For use without lateral wipers.
<sup>6)</sup> 01 No relubrication facility via end piece of RUS. For use with lateral wipers.
<sup>6)</sup> 10 Relubrication facility via end piece of RUS. For use with lateral wipers.
<sup>6)</sup> 10 Relubrication facility via end piece of linear recirculating roller bearing with lubrication nipple (RUS..KS). For use without lateral wipers.
<sup>6)</sup> 11 Relubrication facility via end piece of linear recirculating roller bearing with lubrication nipple (RUS..KS). For use without lateral wipers.
<sup>6)</sup> 20 Relubrication facility via end piece of linear recirculating roller bearing and screwed pipe connection, fitted to RUS..KS.
<sup>6)</sup> 21 Relubrication facility via end piece of linear recirculating roller bearing and screwed pipe connection, fitted to RUS..KS.
<sup>6)</sup> 21 Relubrication facility via end piece of linear recirculating roller bearing and screwed pipe connection, fitted to RUS..KS.

21 Relubrication facility via end piece of linear recirculating roller bearing and screwed pipe connection, fitted to RUS..KS. For use with lateral wipers.

<sup>5)</sup> The values apply to screws with a preservative. If there is a possibility of settling, the screws should be secured against rotation.

Ensure that the adjacent construction has adequate strength!

See VDI guideline 2230!

Fixing screws					
Size	Tightening torque <sup>5)</sup> Nm max.				
M10	83				





Fixing holes				Guideway	Wipers	
K <sub>3</sub> <sup>5)</sup> to DIN 912-12.9	a <sub>4</sub>	H <sub>4</sub>	C <sub>2</sub>		End face <sup>2)</sup>	Lateral <sup>2)</sup>
					V 8050	V 8050
M10	78	12,6 16,6	43	V 7040 <sup>3)</sup> V 8050	<b>AK 8050-0</b> <sup>4)</sup>	AL 19069 <sup>4)</sup>
M10	78	12,6 16,6	80	V 7040 <sup>3)</sup> V 8050	<b>AK 8050-0</b> <sup>4)</sup>	AL 19105 <sup>4)</sup>
M10	78	12,6 16,6	43	V 7040 <sup>3)</sup> V 8050	<b>AK 8050-1</b> <sup>4)</sup>	AL 14032 <sup>4)</sup>
M10	78	12,6 16,6	43	V 7040 <sup>3)</sup> V 8050	<b>AK 8050-1</b> <sup>4)</sup>	<b>AL 14044</b> <sup>4)</sup>



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Each section in the publication FRF begins with a description of the product series and their features.

The significant characteristics of the guidance systems are indicated by means of pictograms.

The advantages of this type of presentation include:

- the reading required is reduced
- the required information can be accessed more quickly
- a direct comparison between alternative bearings is possible.

#### Definition of pictograms

Pictogram	Description
	Linear locating bearing arrangement
	Linear non-locating bearing arrangement
J	Main load direction
	Four raceways for linear roller bearings
	Three raceways for linear roller bearings
JA CA	Two raceways for linear roller bearings, arranged at 45° to the support face
*	One raceway for linear roller bearings
	Can be lubricated with grease
	Can be lubricated with oil
C C	Note temperature
70	This pictogram indicates the page number of the relevant dimension table
	If this information is not adhered to, there is a direct or indirect hazard to the product and/or adjacent construction
	Further information must be noted





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FRF · HYDREL flat cage guidance systems/Linear recirculating roller guidance systems